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# Technical Report

No. 13515

AGT 1500 POWERPACK IMPROVEMENT PROJECT (M1 TMEPS)

CONTRACT NUMBER DAAE07-87-C-R006

MARCH. 1991

VOLUME II OF II

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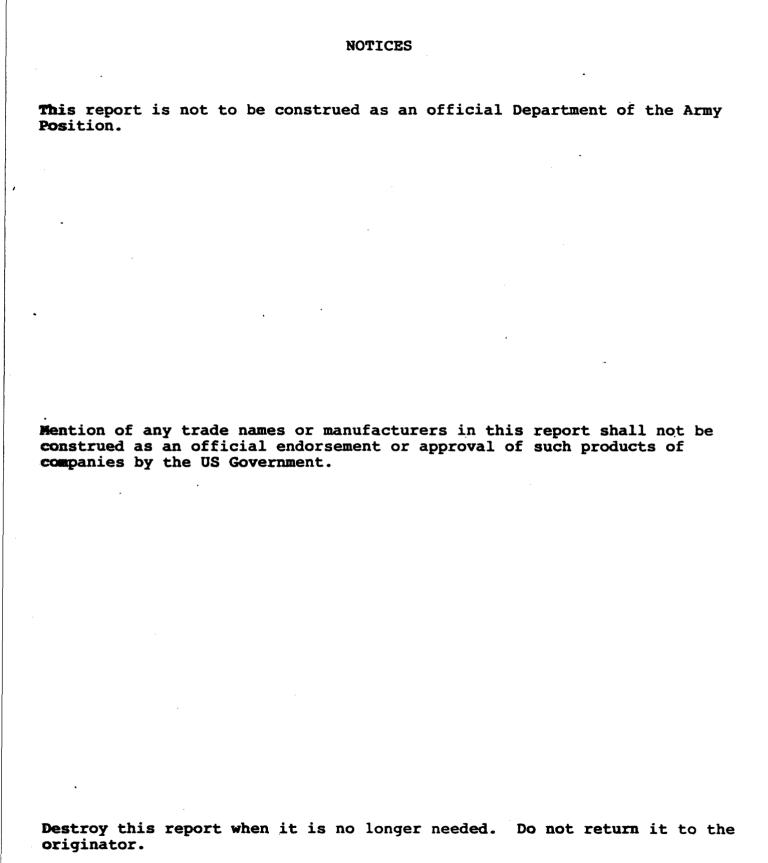
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APPENDIX I

Report No. LYC 90-18 (03-P-902-90)

TME Interchangeability
Demonstration Test

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Manager

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II	TEMPS/MIAI Engine Module Interchangeability Testing: Performance Results

## I. ABSTRACT

Compatibility of AGT 1500 and TME (AGT 1500A) engine forward and rear modules and control units was successfully demonstrated during interchangeability testing conducted between 11 July and 1 August 1989. The interchangeability test consisted of steady-state and transient evaluation of various module configurations (as specified in government contract DAC-100001) of one AGT 1500 and one AGT 1500A (TME) automotive gas turbine selected to participate. Interchangeability testing of the standard electronic control unit and the TME digital electronic control unit (DECU) was also conducted.

Steady-state engine performance demonstrated improved SFC at part power and idle with operation of the TME DECU. Trim power demonstrated for each configuration tested revealed acceptable requirements for vehicle mobility. Engine mechanical data was also found to be acceptable.

Transient response of each configuration was evaluated and data indicates a slight increase in acceleration and deceleration time with operation of the TME DECU. Surge-free operation of all configurations was demonstrated by engine waveoff procedures.

## II. BACKGROUND

The AGT 1500 and AGT 1500A (TME) are both automotive gas turbine engines which are comprised of a two-spool gas generator with a stationary recuperator and free power turbine. The power turbine assembly on both engine models utilizes variable geometry for optimum efficiency at all power levels. The Transverse Mount Engine (TME) design is part of a combined effort by Textron-Lycoming, General Dynamics Land Systems (GDLS), Allison Transmission Division (ATD) of GM and Donaldson Corporation to produce an advanced propulsion system for an improved M-1 Abrams derivative tank. The TME system when fully developed will result in a thirty percent reduction in power-pack volume when compared to the current Abrams main battle tank. Changes in engine design will also result in improved thermodynamic efficiency, increased power output and lower fuel consumption.

## TME Featured Hardware

Principal changes in engine design of the Transverse Mount Engine as related to the current production AGT 1500 automotive gas turbine which result in improved operating efficiency and power output are described briefly below.

## High Pressure Turbine Rotor

The HP turbine rotor was redesigned for increased efficiency and durability. Turbine cooling was optimized and the blades consist of single crystal design (see Figure 1). Modifications to the high pressure turbine account for approximately one percent in SFC improvement.

## High Pressure Turbine Cylinder

The high pressure turbine features a ceramic (TBC) coated cylinder (see Figure 1) for increased efficiency through a reduction in blade tip leakage.

## Recuperator

Improvements to the present design include Hastelloy-S material and increased pre-load for greater durability and effectiveness.

## Power Turbines

The TME power turbine assembly incorporates fuel economy (FEP) turbines which are resized for improved part power performance. Thermodynamic efficiency is increased by 4.5 percent over the current production power turbines.

#### Electronic Control Unit

The electronic control unit utilizes digital technology for increased capabilities. The engine operating schedules are also enhanced for improved efficiency and durability. The TME digital electronic control unit (DECU) includes 'Easy-Ride' protection features which consist of improved starting logic, and the T7-Lag logic to reduce thermal gradients during transient operation. The digital control unit will also feature (when fully developed) diagnostic software to assist in troubleshooting engine related problems.

## Accessory Gearbox Module

The TME accessory gearbox assembly is redesigned to allow for reduced engine volume. The customer power take-off pad used on the current AGT 1500 engine to drive a hydraulic pump for vehicle turret rotation is relocated to the TME (7) seven speed transmission.

As part of sub-contract DAC-100001, an interchangeability test was required to verify compatibility of engine hardware between AGT 1500 and TME configurations. The following module configurations were selected to complete engine dynamometer testing:

TME front module/AGT 1500 rear module - TME DECU
TME front module/AGT 1500 rear module - Standard ECU
AGT 1500 front module/TME rear module - TME DECU
AGT 1500 front module/TME rear module - Standard ECU
Full AGT 1500 engine - TME DECU
Full AGT 1500A (TME) engine - standard ECU

One AGT 1500 engine (A68, S/N LE87680) and one AGT 1500A (TME) engine (A66, S/N LE87100) were selected to complete interchangeability testing and demonstrate compatibility of engine modules and control units.

## Test Objectives

Objectives of the interchange test are as follows:

- 1. Demonstrate compatibility of AGT 1500 and AGT 1500A engine modules and control units.
- 2. Compare steady state engine performance (power and SFC requirements) of engine configurations.
- 3. Review engine transient response handling and stability checks.

## III. TEST EQUIPMENT

## Starter

A Delco Remy 1113883 or Leece Neville 17414MA electric starter powered by a 28 VDC Hobart 6T28-400CL motor generator is used to start the engine.

#### Power Absorption

A Textron Lycoming water brake is used for engine power absorption (LC28800). In addition, this water brake has provisions for monitoring its speed and radial vibration.

## Power Extraction

A Bendix model 30858-3-B 400 ampere starter/generator with associated switches and load bank (United Manufacturing Model DCLB) is used for power extraction from the engine customer power takeoff pad on the M1A1 configuration only.

## Oil Coolers

Engine oil is cooled by means of an industrial type oil to water heat exchanger, Lycoming TES 77-1.

#### Oil Level Measurement

A Robertshaw Controls, Model 5000, indicator and probe were used to measure oil level.

## **Vibrations**

Vibration measurement will consist of Trig-Tek vibration meters in conjunction with velocity-displacement pickups.

#### Temperatures

Temperatures are measured by Chromel Alumel (Type K) thermocouples. Signals are conditioned through appropriate analog to digital converters.

#### **Pressures**

Pneumatic and hydraulic pressures are measured through calibrated pressure transducers.

#### Flows

Turbine flow meters with associated converters, amplifiers and readouts are used to measure oil and fuel flow rates.

#### Speeds

Magnetic pickups with associated electrical hardware are used to measure the compressor/turbine and output shaft speeds.

## Torque

A strain gauged torque element which supports an AGT 1500 water brake is used to measure engine torque.

#### Air Flow

A calibrated axial bellmouth assembly (ASME standards) with static and total pressure probes is utilized for airflow measurement during performance calibrations.

Engine main bearing flow rates, pressures and temperatures were observed to be within specified limits for all configurations tested. Oil consumption measured during interchange testing was also found to be in specification.

## VI. CONCLUSIONS AND RECOMMENDATIONS

It is concluded that all configurations tested (as outlined in contract DAC100001) demonstrated acceptable compatibility of AGT 1500 and TME engine modules.

Engine performance indicates improved SFC at idle and part power operation with use of the TME digital electronic control unit and hardware (see Appendix II).

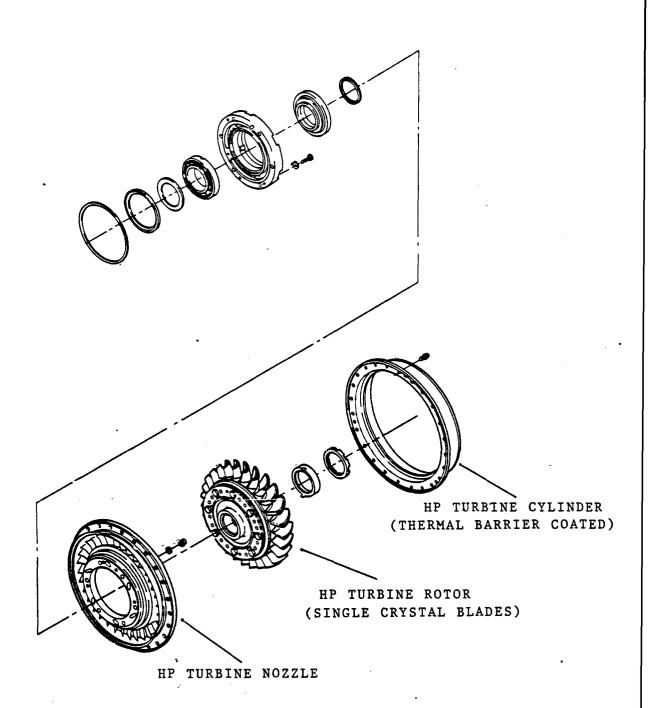
All module configurations tested demonstrated adequate trim power requirements (as needed for vehicle mobility). When the standard AGT 1500 control unit was utilized in each configuration additional power was available by readjusting the trim NH governor setting established during the baseline AGT 1500 test.

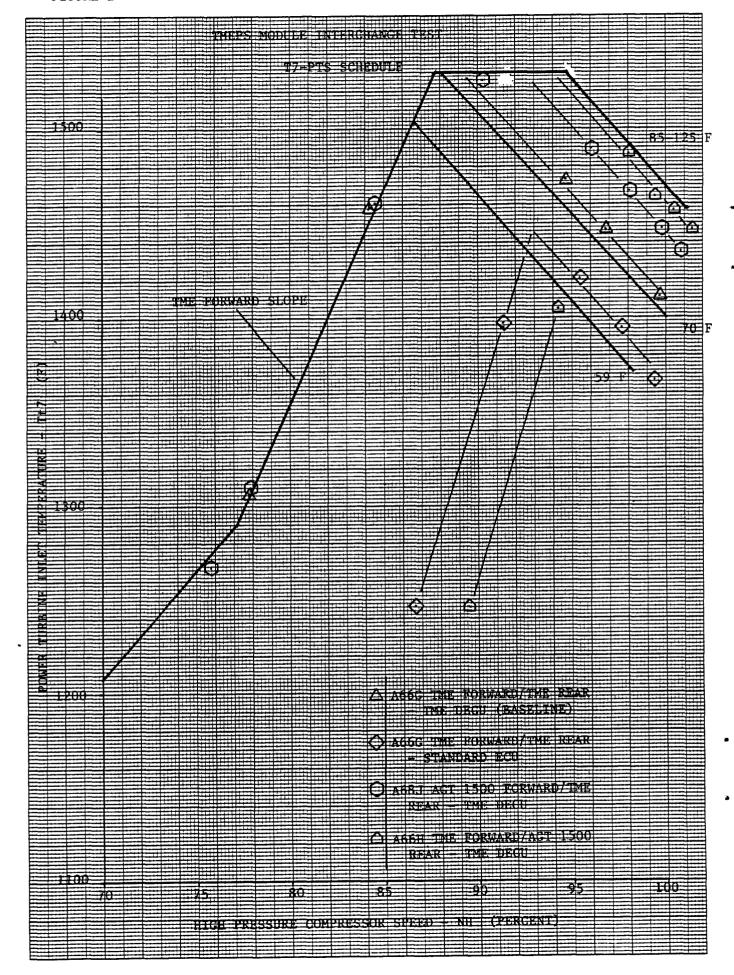
Mechanical data measured during the test did not reveal any significant problems when AGT 1500 and TME hardware are interchanged. All configurations tested operated free of surge as verified by engine waveoff procedures. Transient response (acceleration and deceleration time measurement) does indicate a slight increase when the TME DECU is utilized. The acceleration and deceleration times measured during the interchangeability test were observed to be within the specification limits defined for operation with the Production AGT 1500 Fuel Economy DECU.

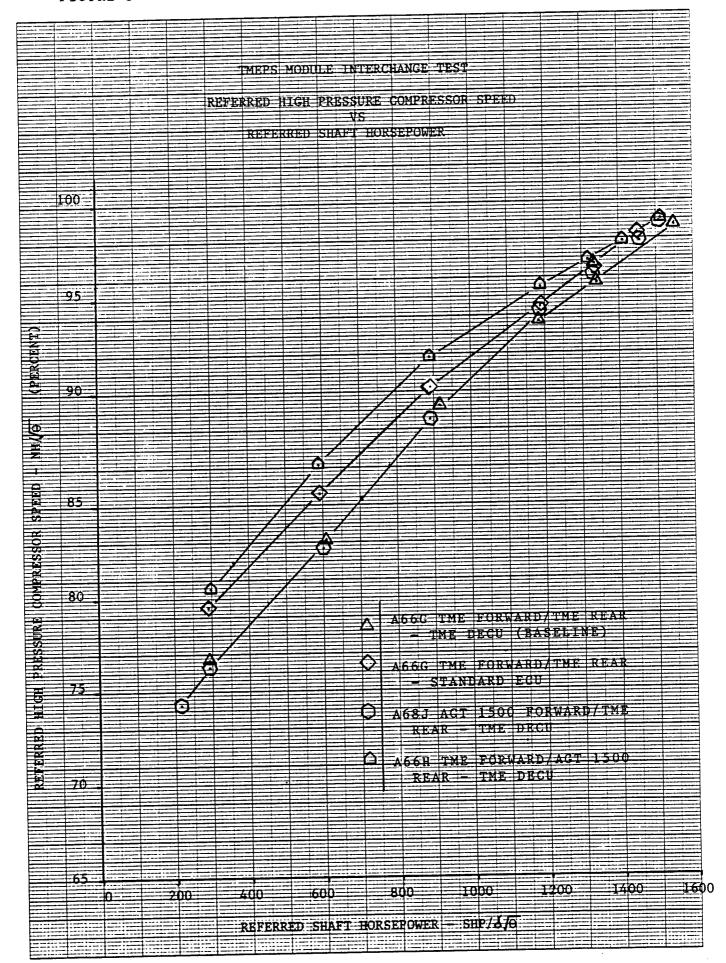
It is recommended that if the TME digital electronic control unit (DECU) is used with AGT 1500 forward or rear module components, the trim NH governor should be readjusted to maintain AGT 1500 operating conditions.

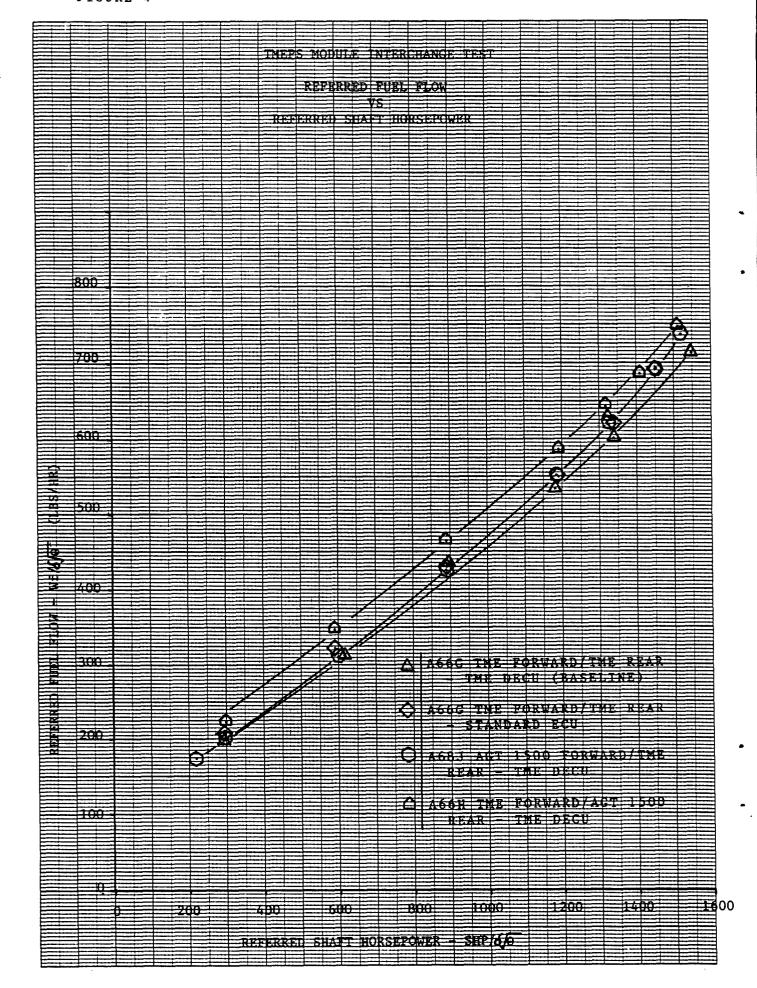
It is recommended that this report be accepted as evidence of the successful demonstration of interchangeability between AGT 1500 and TME modules and control units. FIGURES

## HP TURBINE ASSEMBLY - TME

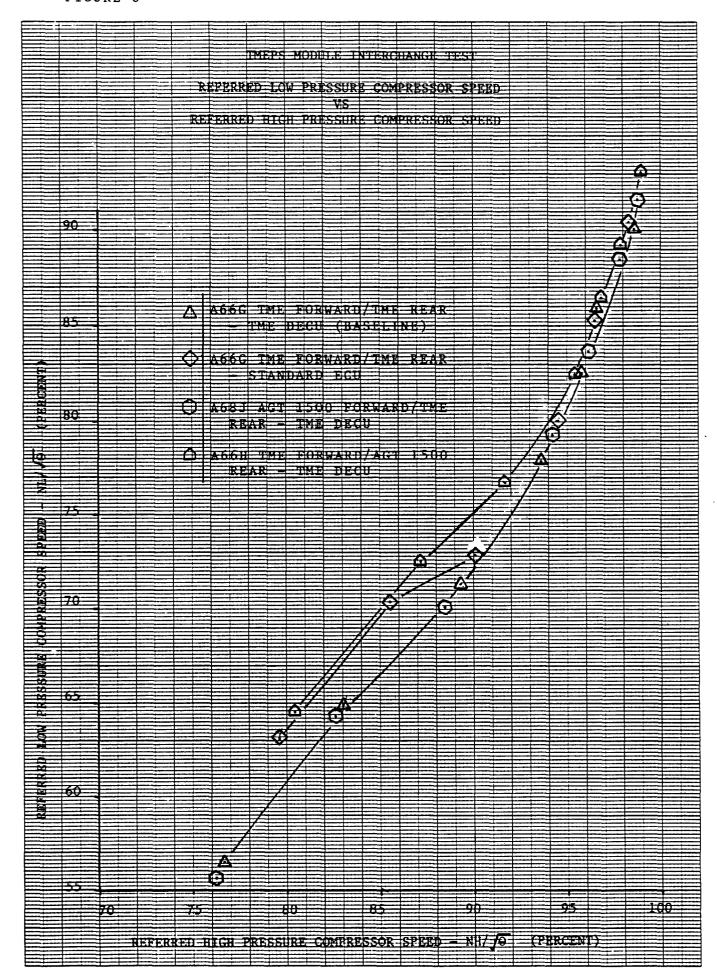








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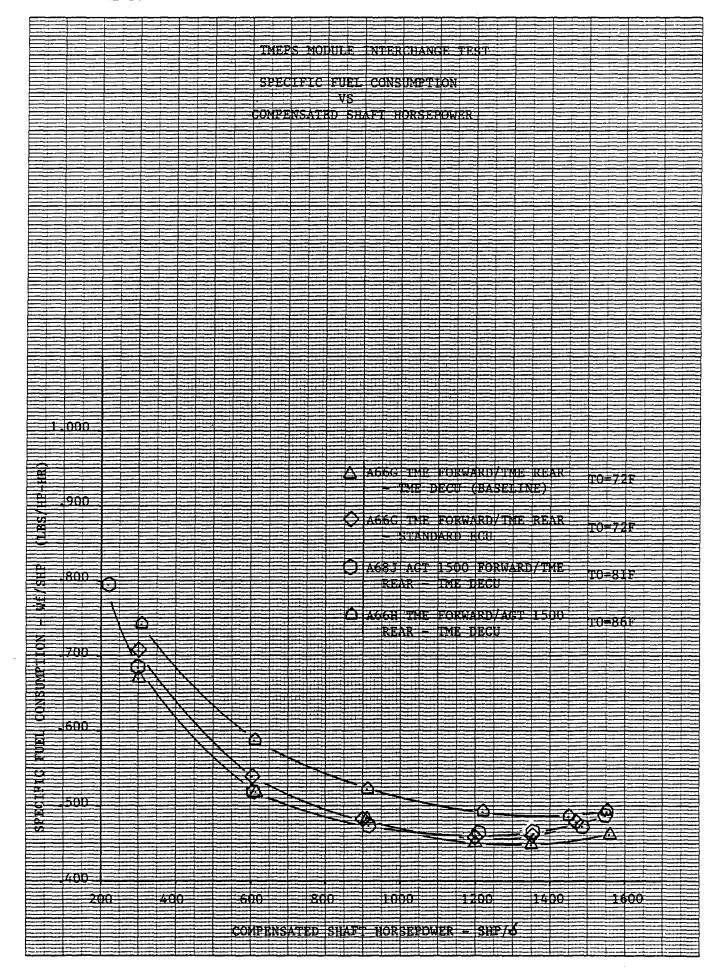


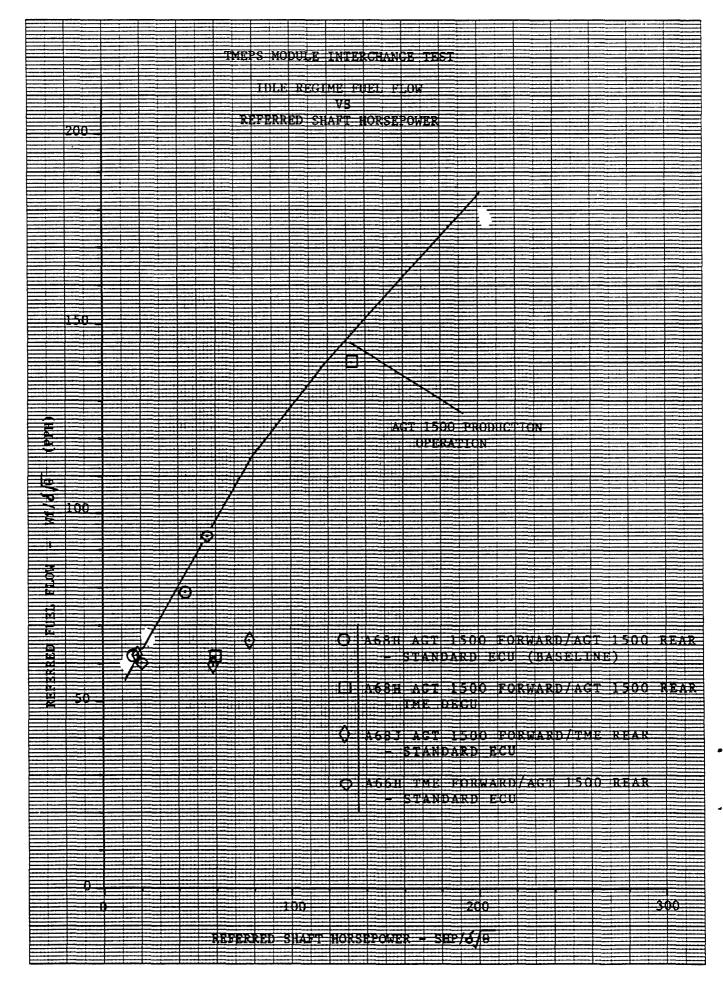
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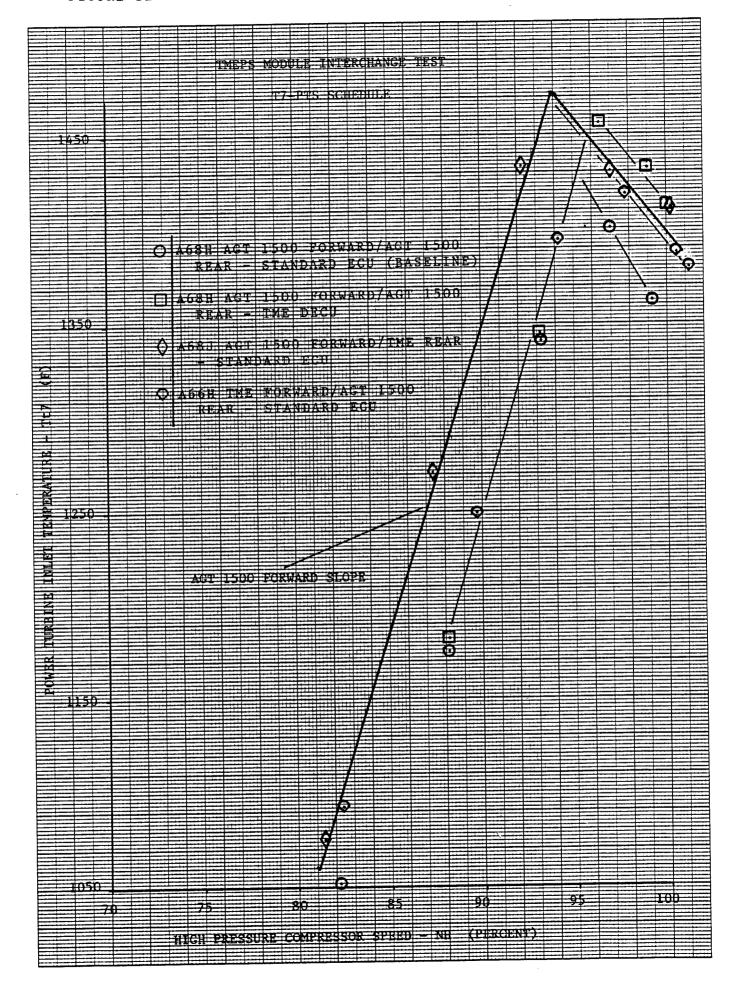
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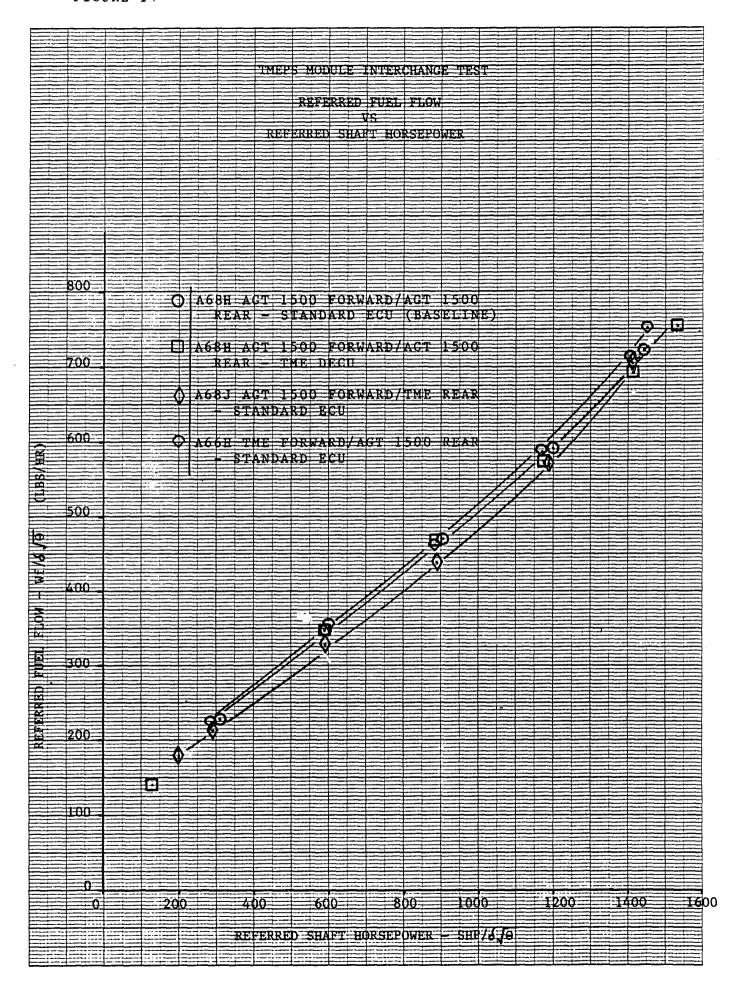
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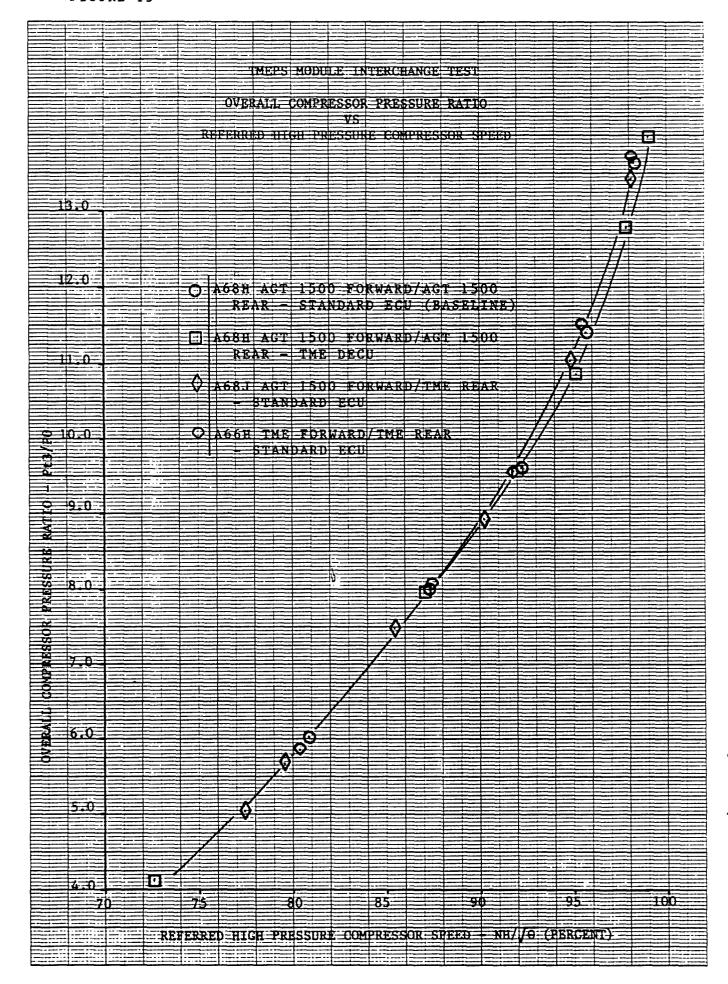


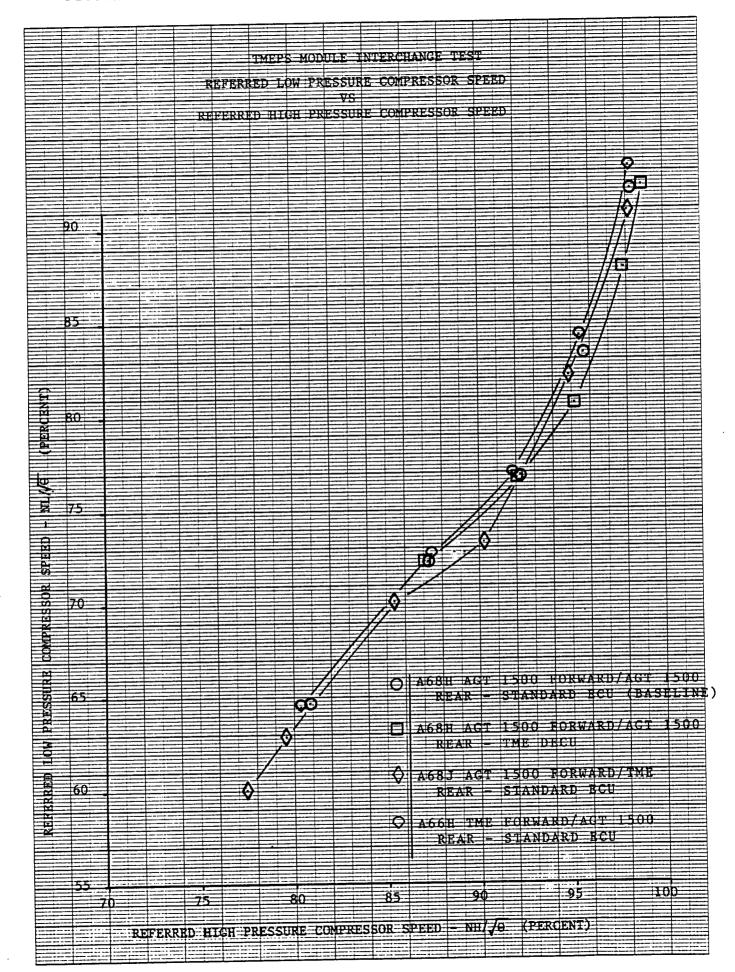


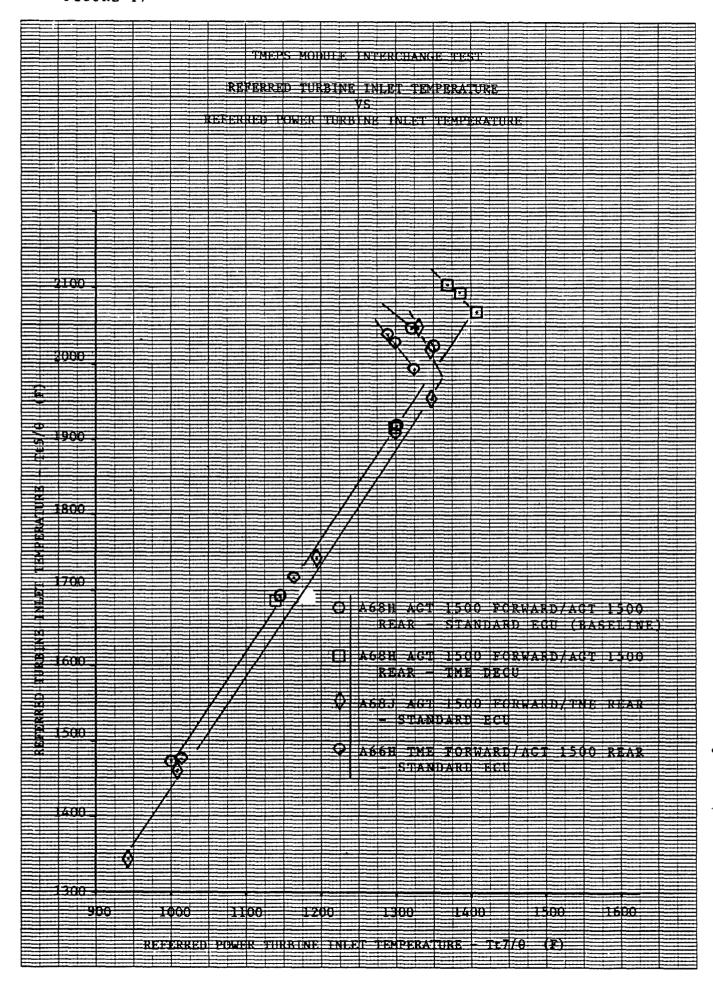


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B	90	
35   30   30   30   30   30   30   30		
## C ##		
REAR   STANDARD EGU (BASELINE)		
A68B AGT 1500 FORWARD/AGT 1500		
75		
## 70		A A68.1 ACT 1500 FORWARD/TMR PRAB
70 - STANDARD ECU - S	High the second	C A66H TME FORWARD/ACT 1500 REAR
0 200 400 600 800 1000 1200 1400 1600		- STANDARD ECU
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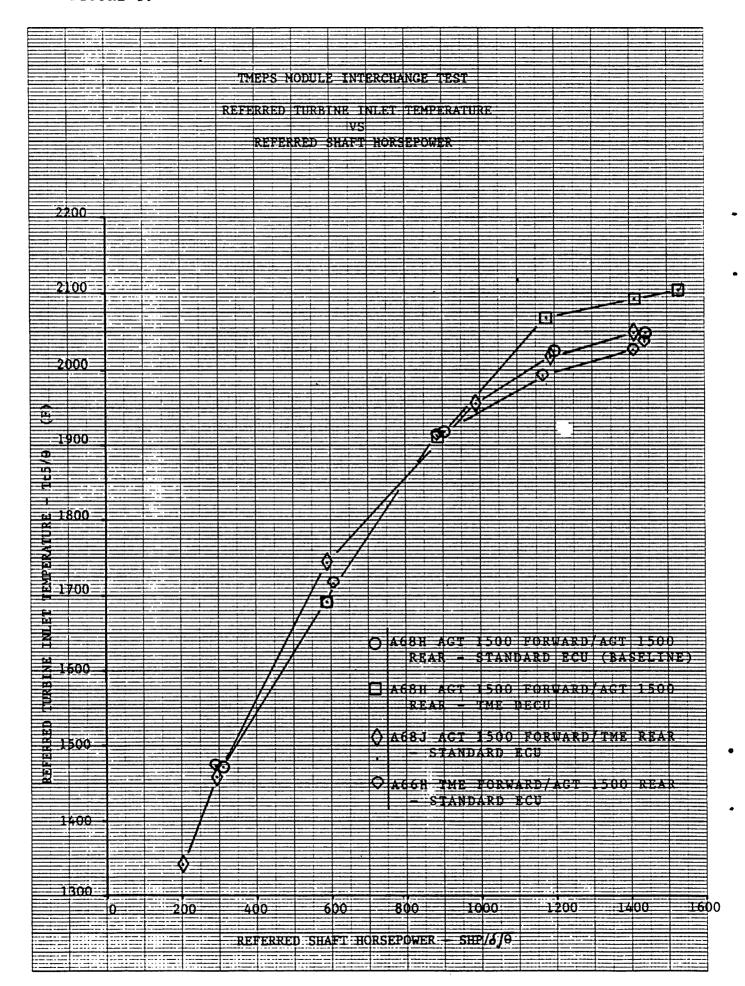


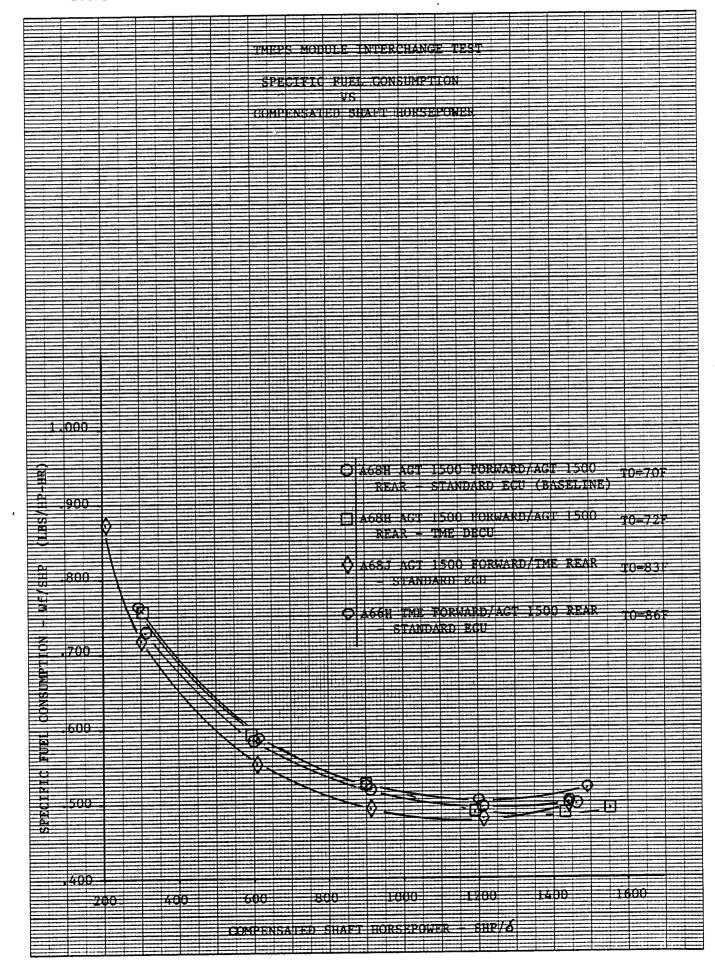


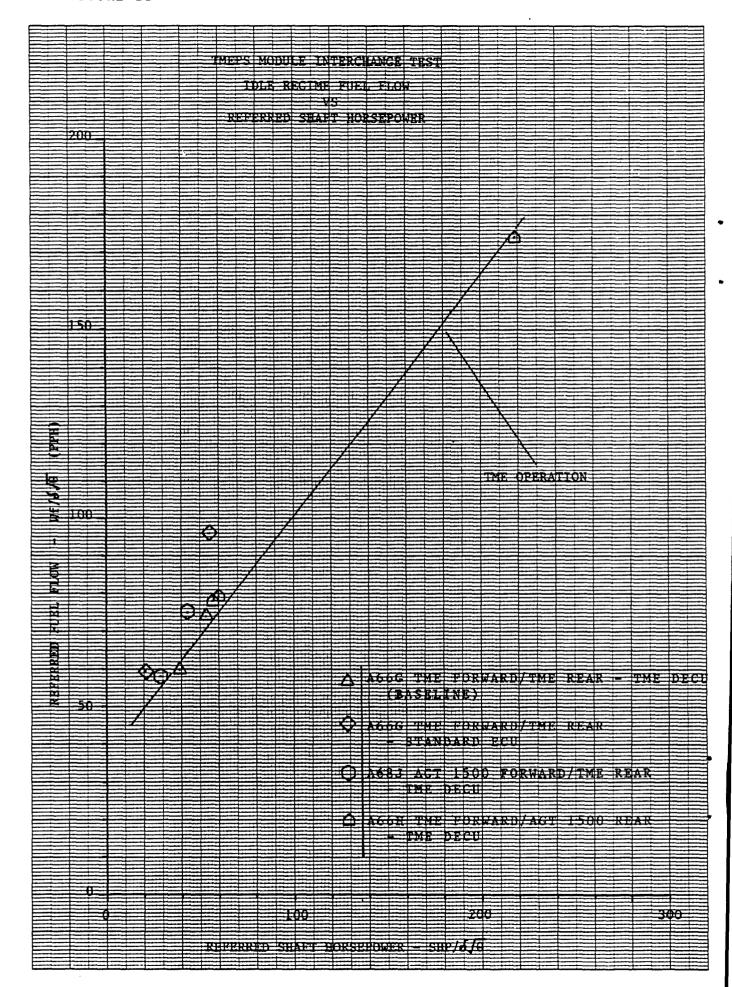


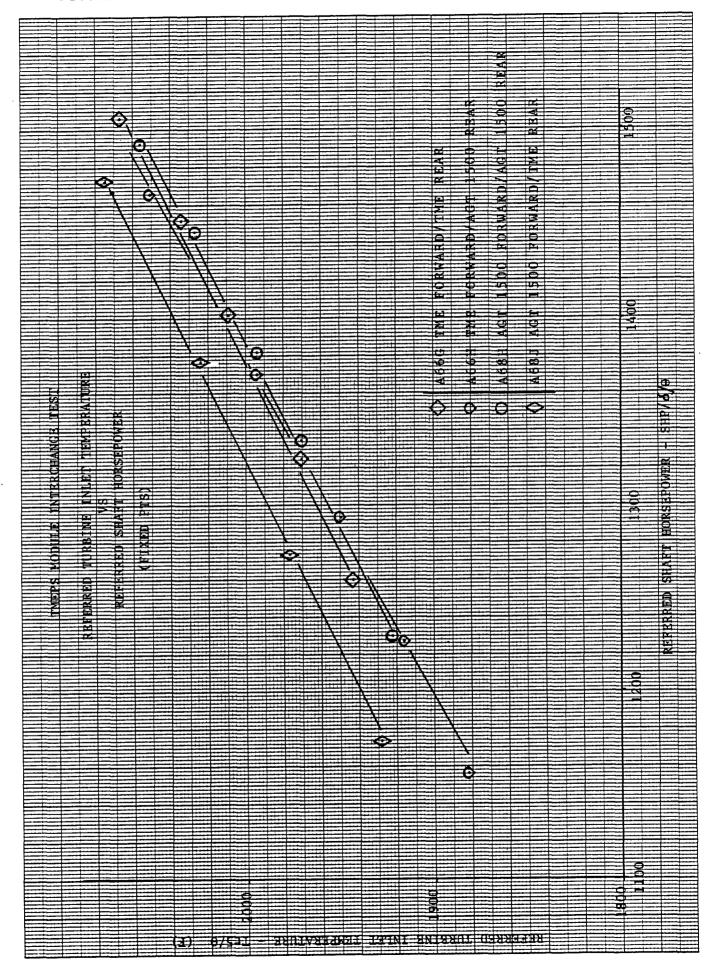


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KOE 10 X 10 TO THE CENTIMETER 18 X 25 CM.

TABLES

## TEMPS MODULE INTERCHANGE TEST

# ENGINE A68/A66 SUMMARY OF TEST EVENTS

AGT 1500

Description of Test Activity	Engine A68H (Baseline AGT 1500) installation in cell T-11. Compressors B&B cleaned prior to initial calibration (performance testing).	Initial mechanical checkout and fixed PTS optimum power setting. Baseline engine performance calibration with analog control unit (accessory load on and off). Handling & stability checks also completed.	Engine performance calibration with TME digital control unit. Handling and stability checks completed.	Engine removed from cell T-11.
Date	7-11-89	7-14-89	7-17-89	7-17-89
Accumulated Engine Run Time (Hrs)		7.98	10.25	10.25

## TEMPS MODULE INTERCHANGE TEST

# ENGINE A68/A66 SUMMARY OF TEST EVENTS

AGT 1500

Description of Test Activity	Engine A66 installation in cell T-11. Mechanical checkout and PTS optimization performed.	Performance (baseline) calibrations with TME DECU and A68 standard analog control units completed. Handling and stability checks with both control units completed.	Engine forward module (A66) removed (split) from cell T-11 to accommodate interchange of TME and AGT 1500 modules.
Date	7-18-89	7-18-89	7-26-89
Accumulated Engine Run Time (Hrs)	0	11.27	11.27

## TEMPS MODULE INTERCHANGE TEST

# ENGINE A68/A66 SUMMARY OF TEST EVENTS

AGT 1500

Description of Test Activity	AGT 1500 engine forward module (including accessory gearbox) installed on rear module of engine A66 (TME).	Performance calibration and handling and stability checks conducted with both control units (TME and AGT 1500).	Engine A68J (AGT 1500 forward module and TME rear module) removed from cell T-11.
Date	7-26-89	7-28-89	8-1-89
Accumulated Engine Run Time ( <u>Hrs)</u>	0	4.78	4.78

## TEMPS MODULE INTERCHANGE TEST

# ENGINE A68/A66 SUMMARY OF TEST EVENTS

AGT 1500

## SUMMARY OF EVENTS

Description of Test Activity	Engine A66 (TME) forward module mated with A68 (AGT1500) rear module. Installation in cell T-11 completed.	Engine performance calibrations including handling and stability checks completed with each control unit.	Engine A66H removal from cell T-11 completed. Interchangeability testing completed.
Date	8-2-89	8-3-89	8-4-89
Accumulated Engine Run Time (Hrs)	<b>o</b> .	5.87	5.87

(59 DEG. F DAY)

MGT	1362	1410	1415	1369	1413	1434	1387	1451
IM POWER <u>NH</u>	99.3	100.2	6.66	99.5	100.3	100.7	100.6	101.3
DEMONSTRATED TRIM POWER <u>BIENT SHP/</u> δ <u>NH</u>	1464 (2)	1553	1566	1473	1452	1548	1446	1558
DEMONS AMBIENT	02	11	70	72	83	79	85	83
SFC AT POWER 1200 SHP	.488	.483	.448	.456	.471	.458	.490	.478
SHP (1) (87 DEG, F DAY)	1542	1600	1596	1530	1476	1503	1490	1576
CONTROL	STANDARD	TME DIGITAL	TME DIGITAL	STANDARD	STANDARD	TME DIGITAL	STANDARD	TME DIGITAL
MODULE I REAR	AGT1500		TME		TME		AGT1500	
MOD	AGT1500		TME		AGT1500		TMË	
BUILD	A68H		A66G		A68J		, A66H	

NOTES

1) MAXIMUM POWER CAPABILITY (87 DEG. F DAY) FOR EACH CONFIGURATION PERTAINS TO EACH ENGINE (TME/AGT1500)

MODEL LIMITS (T5, T7, NH)

<sup>2)</sup> ENGINE A68H TRIM NH SETTING SET FOR 1500 SHP (WITH 20 SHPX). THE 1464 SHP/6 DATA POINT LISTED PERTAINS TO TRIM NH SETTING WITH NO HORSEPOWER EXTRACTION.

### INTERCHANGE TEST DATA ORGANIZATION

### Group 1 (Figures 2-11)

Test <u>Sequence</u>	Engine <u>Build</u>	Modul <u>Forward</u>	e <u>Rear</u>	Control . <u>Unit</u>	Description
3	A66 G	TME	TME	TME DECU	TME Baseline
4	A66 G	TME	TME	Standard	Effect of Standard Control Unit
6	A68 J	AGT 1500	TME	TME DECU	Effect of AGT 1500 forward module
8	A66 H	TME	AGT 1500	TME DECU	Effect of AGT 1500 rear module

### Group 2 (Figures 12-21)

Test <u>Sequence</u>	Engine Build	Modu Forward	le <u>Rear</u>	Control Unit	Description
1	A68 H	AGT 1500	AGT 1500	Standard	AGT 1500 Baseline
2	A68 H	AGT 1500	AGT 1500	TME DECU	Effect of TME DECU
5	A68 J	AGT 1500	TME	Standard	Effect of TME rear module
7	A66 H	TME	AGT 1500	Standard	Effect of TME forward module

NOTE: See Appendix II (page 2) for additional test configuration descriptions.

TABLE 4
ENGINE TRANSIENT PERFORMANCE

Engine Build	Control Unit	Acceleration (sec)	<u>Deceleration</u> (sec)
A68 H (STD)	Standard TME DECU	2.5	2.0 2.6
A66 G (TME)	Standard	2.2	2.0
	TME DECU	4.2	3.1
A68 J	Standard	2.4	2.0
	TME DECU	3.7	4.6
А66 Н	Standard	2.5	2.1
	TME DECU	3.6	2.8

NOTE: Acceleration times measured from gear engaged idle to 90 percent of maximum compensated power.

Deceleration times measured from trim power to 30 percent compensated power.

APPENDICES

### TME INTERCHANGEABILITY PROGRAM TEST PLAN ENGINE A66/A68 LYC 89-44

Prepared by:

F. Macri

AGT 1500 Development Test Engineer

Approved by:

AGT 1500 Development

Test Manager

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5.0	Test Requirements	. 2
6.0	Test Procedures	6
7.0	Data .	7

TITLE: TMEPS/AGT 1500 Module Interchangeability Test

### 1.0 <u>Introduction</u>:

This test plan describes test requirements associated with the TMEPS/MIA1 interchangeability test as described in Section 5.10 of TME subcontract DAC100001. Testing will be performed at Textron Lycoming in Stratford, Connecticut and will commence in July of 1989. Duration of testing will be approximately two weeks. This test plan is prepared under the Contract DAAE07-87-C-R006.

The interchange program will involve the use of one AGT 1500 engine (A68, S/N LE87680) and one AGT 1500A engine (A66, S/N LE87100) to demonstrate interchangeability of engine modules. Testing will involve the the following configurations (as specified in the TME statement of work):

- 1) TMEPS Front Engine Module/M1A1 Rear Engine Module with TMEPS ECU.
- 2) TMEPS Front Engine Module/M1A1 Rear Engine Module with M1A1 ECU.
- 3) TMEPS Rear Engine Module/MlAl Front Engine Module with TMEPS ECU.
- 4) TMEPS Rear Engine Module/M1A1 Front Engine Module with M1A1 ECU.
- 5) TMEPS Engine with M1A1 ECU.
- 6) M1A1 Engine with TMEPS ECU.

### 2.0 Applicable Documents:

- 2.1 A68/A66 Build parts list.
- 2.2 TME Fuel Economy Test Plan (LYC88028, 03-P-802-88).
- 2.3 TME Engineering Test Requests.
- 3.0 Objectives:
- 3.1 Demonstrate interchangeability of M1A1 and TMEPS engine configurations.

### 4.0 <u>Test Article Description/Configuration</u>:

The AGT 1500 and AGT 1500A (TME) are both recuperative, free power turbine shaft, automotive engines which incorporate a two-spool compressor and variable geometry operation. The AGT 1500A engine features more advanced components which include reduced cooling flow single crystal H.P. turbine blades, fuel economy power (FEP) turbines and a digital electronic control unit. The TME

### 4.0 Test Article Description/Configuration (continued):

Accessory gearbox has also been redesigned in which the customer power takeoff pad is removed and relocated (transmission).

### 5.0 <u>Test Requirements:</u>

All engine configurations shall be performed in one designated test cell with the same test equipment and instrumentation utilized throughout.

### 5.1 Test Facilities:

A Textron Lycoming development test cell, capable of steady-state and transient condition monitoring with data acquisition systems.

### 5.2 Test Conditions:

Testing is to be conducted under prevailing ambient temperature (no greater than 87°F) and sea level pressure.

### 5.3 Special Test Equipment

### 5.3.1 Starter

A Delco Remy 1113883 or Leece Neville 17414MA electric starter powered by a 28 VDC Hobart 6T28-400CL motor generator is used to start the engine.

### 5.3.2 Power Absorption

A Textron Lycoming water brake is used for engine power absorption (LC8800). In addition, this water brake has provisions for monitoring it s speed and radial vibration.

### 5.3.3 Power Extraction

A Bendix model 30858-3-8 400 ampere starter/generator with associated switches and load bank (United Manufacturing Model DCLB) is used for power extraction from the engine customer power takeoff pad on the M1A1 configuration only.

### 5.3.4 Oil Coolers

Engine oil is cooled by means of an industrial type oil to water heat exchanger, Lycoming TES 77-1.

### 5.4 Instrumentation

A list of instrumentation required for the test is provided in Table I. Critical installation procedures as specified in Lycoming T.O.D. B0003-69 will be applied to engine performance parameters during the initial phase of interchangeability testing.

### 5.4.1 Vibrations

Vibration measurement will consist of Trig-Tek vibration meters in conjunction with velocity-displacement pickups.

### 5.4.2 <u>Temperatures</u>

Temperatures are measured by Chromel Alumel (Type K) thermocouples. Signals are conditioned through appropriate analog to digital converters.

### 5.4.3 Pressures

Pneumatic and hydraulic pressures are measured through calibrated pressure transducers.

### 5.4.4 Flows

Turbine flow meters with associated converters, amplifiers and readouts are used to measure oil and fuel flow rates.

### 5.4.5 Speeds

Magnetic pickups with associated electrical hardware are used to measure the compressor/turbine and output shaft speeds.

### 5.4.6 Torque

A strain gauged torque element which supports an AGT 1500 water brake is used to measure engine torque.

### 5.4.7 Air Flow

A calibrated axial bellmouth assembly (ASME standards) with static and total pressure probes is utilized for airflow measurement during performance calibrations.

### 5.4.8 Accuracy of Data

For all engine and component calibrations and tests, or demonstrations, reported data shall have a steady-state accuracy within the tolerances shown below. The accuracy

### 5.4.8 Accuracy of Data (continued)

of transient data and the corresponding instrument calibration methods shall be subject to the approval of the using service and shall be described in the test report. All instruments and equipment shall be calibrated as necessary to ensure that the required degree of accuracy is maintained.

### Item of Data

### Tolerance

Rotor Speed(s)

± 0.2 percent of the value obtained at maximum rating.

Torque

± 0.5 percent of the value measured for intermediate rating and above.

± 0.5 percent of the value measured at intermediate rating for all values below intermediate rating.

Air Flow

± 1.0 percent of the value measured for intermediate rating and above.

± 1.0 percent of the value measured at intermediate rating for all values below intermediate rating.

Temperatures

±1.0°C up to 200°C. ±2.0°C between 200°C and 800°C. ±4.0°C above 800°C.

Engine Weight

±1.0 lbs or ±0.1 percent of the weight being determined, whichever is greater.

Vibration Velocity

±5.0 percent of specified engine limit during 4.5.7 vibration survey, 4.6.2.4.2 vibration scan and resonant search and 4.6.6.5 vibration and stress test.

### 5.4.8 Accuracy of Data (continued)

### Item\_of Data

### Tolerance

Vibration Velocity

± 10.0 percent of specified engine limit for all other

tests.

All Other Data

± 2.0 percent of the value obtained at maximum rating.

### 5.5 Data Acquisition Equipment

### 5.5.1 Automatic Data Acquisition Equipment

Continuous automatic monitoring is facilitated by a HP 1000 processor based turbine engine measurement and data acquisition system. The computer, in conjunction with individual measurement sensors, will acquire, display and calculate steady-state and transient data. A Gould model 2800 strip chart recorder will record transient data for selected parameters continuously during engine operation.

### 5.6 Operating Parameters

### 5.6.1 Fuel and Oil Data

Samples of fuel and oil shall be taken at the discretion of the responsible test engineer. As a minimum, a fuel and oil sample shall be obtained at the start and completion of the test. Both samples shall be analyzed for physical and chemical properties to determine conformance with applicable fuel and oil specifications.

### 5.6.2 Engine Operating Limits

The engine operating limits are to be used as guidelines for the operator during the test

Maximum Gas Generator Speed - NH: 103.4% (TME)

102.0% (AGT 1500)

Maximum Output Shaft Speed - NPT: 104.0% (TME)

104.0% (AGT 1500)

Maximum H.P. Turbine Inlet Temperature: 2225°F (TME) 2200°F (AGT 1500)

Maximum Measured Temperature  $(T_T^7)$ : 1570°F (TME) 1530°F (AGT 1500)

### 6.0 Test Procedures:

### 6.1 Test Matrix

Interchangeability testing will be performed on the following engine configurations:

- 1) M1A1 (AGT 1500) baseline
  - a) M1A1 ECU
  - b) TMEPS DECU
- 2) TMEPS (AGT 1500A) baseline
  - a) MlA1 ECU
  - b) TMEPS DECU
- 3) TMEPS Rear Module/MIA1 Forward Module
  - a) MlA1 ECU
  - b) TMEPS DECU
- 4) TMEPS Forward Module/M1A1 Rear Module
  - a) MlA1 ECU
  - b) TMEPS DECU

An engine calibration will be performed to establish the performance characteristics of each configuration. Testing conducted on the AGT 1500 baseline configuration shall be completed with and without customer power extraction. In addition, prior to the beginning of each calibration, the engine may be cleaned with B&B solution and water as specified in an acceptable procedure.

A PTS (power turbine stator) optimization procedure will be completed on each baseline configuration (M1A1/TMEPS) to establish the engine maximum power capability at 87°F seal level, uninstalled.

Engine transient performance shall be evaluated for each configuration as specified in Table II. Data obtained for each configuration specified above shall be acquired at power levels described in Table II.

### 6.2 Inspections

Boroscope inspection and limited disassembly may be performed as necessary to document hardware conditions during the test.

### 7.0 <u>Data</u>:

### 7.1 <u>Data Collection Methodology</u>

### 7.1.1 <u>Miscellaneous Data</u>

The date, test title, engine model designation and serial number shall be recorded on each log sheet.

### 7.1.2 Test Notes

Notes shall be placed on the log sheets of all incidents of the run, such as leaks, vibrations and other irregular functioning of the engine or equipment, and corrective measures taken.

### 7.1.3. Steady-State Data

During operation at each specified steady state condition and after performance stabilization, the data shall be recorded as specified in Table I. Stabilization time at each steady-state power level in Table II shall be no less than 5 minutes.

### 7.1.4 Transient Data

For each calibration transient performed, the data shall be recorded as specified in Table I.

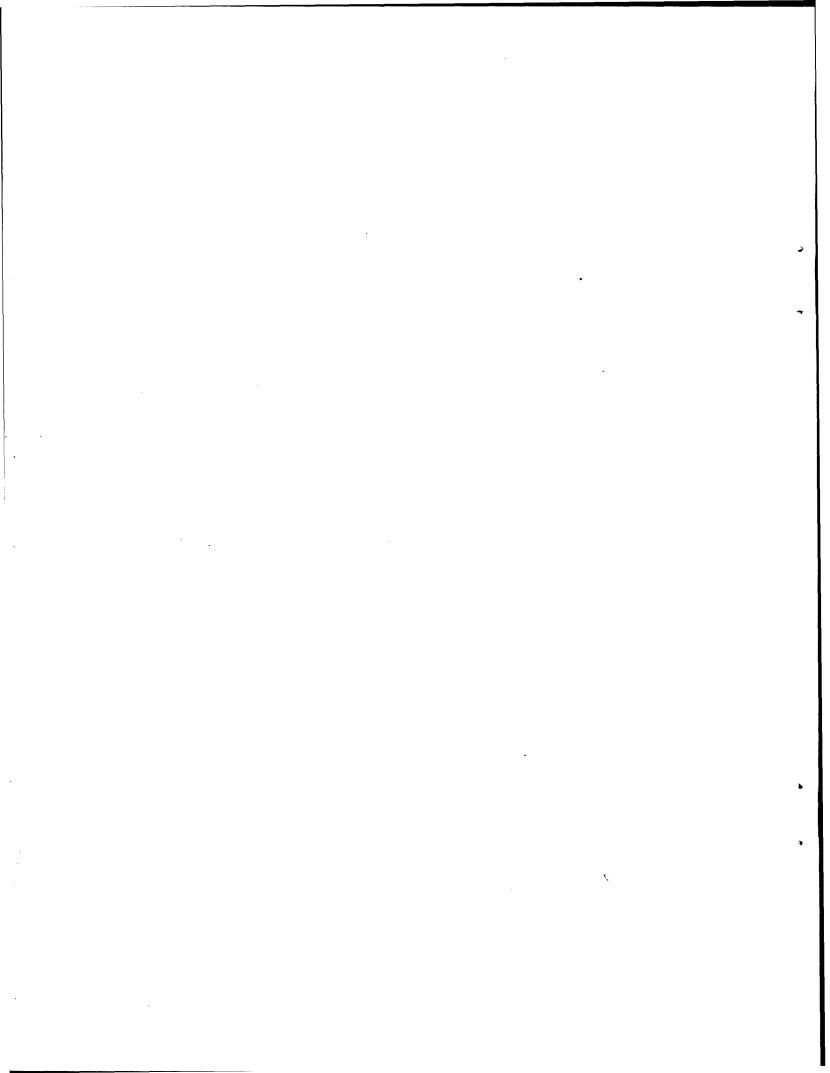
7.1.5 During each start requiring start data evaluation, the data shall be recorded as specified in Table I.

### 7.1.6 Barometer Reading

The barometer reading shall be corrected for temperature and shall be read and recorded at intervals not exceeding three hours, if required for engine performance calibrations.

### 7.1.7 <u>Corrections</u>

Reading of shaft horsepower, rotor speed, airflow rate, fuel flow rate, specific fuel consumption, gas pressure and gas temperatures will be corrected to standard sea level atmospheric conditions.



### TABLE I

		STARTING	STEADY STATE	PRE POST TEST CAL TRANSIENTS
1.	Time of Day	x	x	
2.	Engine Build Run Time		X	
3.	Power Setting		X	
4.	L.P. Compressor Rotor Speed,			
	% RPM	X	X	X
5.	H.P. Compressor Rotor Speed,		•	
	% RPM	X	X	X
6.	Power Turbine Rotor Speed,			
	% RPM	X	×	X
7.	Variable Inlet Guide Vane			
	Position, Volts		X	X
8.	Power Turbine Stator Position,	,		
	Volts		X	. <b>X</b>
9.	Fuel Flow LBM/HR	X	X	X
10.	Bellmouth Inlet Total Pressure	≥,		
	in H <sub>2</sub> O		<b>X</b> .	
11.	Bellmouth Inlet Static Pressur	:e,		
• •	in H <sub>2</sub> O (4)	_	X	
12.	Bellmouth Inlet Delta Pressure	<b>}</b> ,		
10	in H <sub>2</sub> O (4)	. •	X	
13.	L.P. Compressor Discharge Tota	II	49	••
1.4	Pressure, PSIA	. 1	X	X
14.	H.P. Compressor Discharge Tota	7.7	v	v
15.	Pressure, PSIA L.P. Compressor Discharge Tota		· <b>X</b>	X
10.	Temperature, F	r.T.	X	x
16.	H.P. Compressor Discharge Tota	.7	A	. ^
10.	Temperature, F	L.L.	X	x
17.	Recuperator Total Discharge		A	A
	Temperature (4) F		X	x
18.	Measured Gas Temperature, OF	x	X	X
19.	Maximum Measured Gas		••	••
	Temperature, °F	x	x	X
20.	Gas Generator Rotor Inlet			
	Temperature, OF (Calculated)			
21.	Oil Level, QTS.		X	
22.	No. 1 Bearing Scavenge			
	Pressure, PSIG		X	
23.	No. 1 Bearing Scavenge			
	Temperature, F		X	
24.	No. 1 Bearing Oil Flow, LBM/HR	}	X	
25.	No. 2 Bearing Feed Pressure, P		x	
26.	No. 2 Bearing Oil Flow, LBM/HR		X	
27.	No. 4 Bearing Feed Pressure, P		X	
28.	No. 4 Bearing Scavenge Pressur	e,		
	PSIG	•	X	

### TABLE I (Cont'd)

		TARTING	STEADY STATE	PRE POST TEST CAL TRANSIENTS
29.	No. 4 Bearing Scavenge Temperature, F		x	
30.	No. 4 Bearing Oil Flow, LBM/HR		x	
31.	No. 5 & 6 Bearing Scavenge Pressure, PSIG		x	
32.	No. 5 & 6 Bearing Scavenge Temperature, F		x	
33.	No. 5 & 6 Bearing Oil Flow, LBM/HR		<b>x</b>	
34.	No. 6B Bearing Scavenge Pressure, PSIG		x	
35.	No. 6B Bearing Scavenge Temperature, F		x	
36.	Reduction Gearbox Oil Flow, LBM/HR		х	
37.	Accessory Gearbox Scavenge Pressure, PSIG		X	
38.	Accessory Gearbox Oil Flow, LBM/HR		x	
39.	Oil Cooler Inlet Temperature,	F	X	
40.	Oil Cooler Exit Temperature, OF	•	X	
41.	Fuel Inlet Pressure, PSIG		X	
42.	Final Inlat Tamparature OF		X	
		_	X	•
44.	Water brake Lube Pressure, PSIG Water brake Inlet Temperature, Water brake Exit Temperature,	°F .	X	
45.	Water brake Exit Temperature, O	F	X	
46.	Water brake Inlet Pressure, PSI		X	
47.	Displacement - L.P. Compressor Hsg, MIL		x	X
48.	Velocity - L.P. Compressor Hsg, IPS		<b>x</b> .	x
49.	Displacement - Intermediate Hsg, MIL		X	x
50.	Velocity - Intermediate Hsg, IPS		X	X
51.	Displacement Air Diffuser, MIL		x	x
52.	Velocity - Air Diffuser, IPS		X	X
53.	Displacement - Water brake, MIL		X	X
54.	Velocity - Water brake, IPS	_	X	x
55.	Output Shaft Torque, LBF-FT		X	X
56.	Barometer		X	X
57.	Start Number	X	A	•
58.	Time to Ignition Actuation	X		
59.	Time to light Off, Sec.	X		
60.	Time to light off, sec.  Time to Starter Cutout, Sec.	X		
61.	Time to Stabilize Idle			
62	<pre>% RPM, Sec. Additional Data as Required</pre>	X X	x	x

NOTE: Transient recording capability may be altered to meet test requirements.

### TABLE II

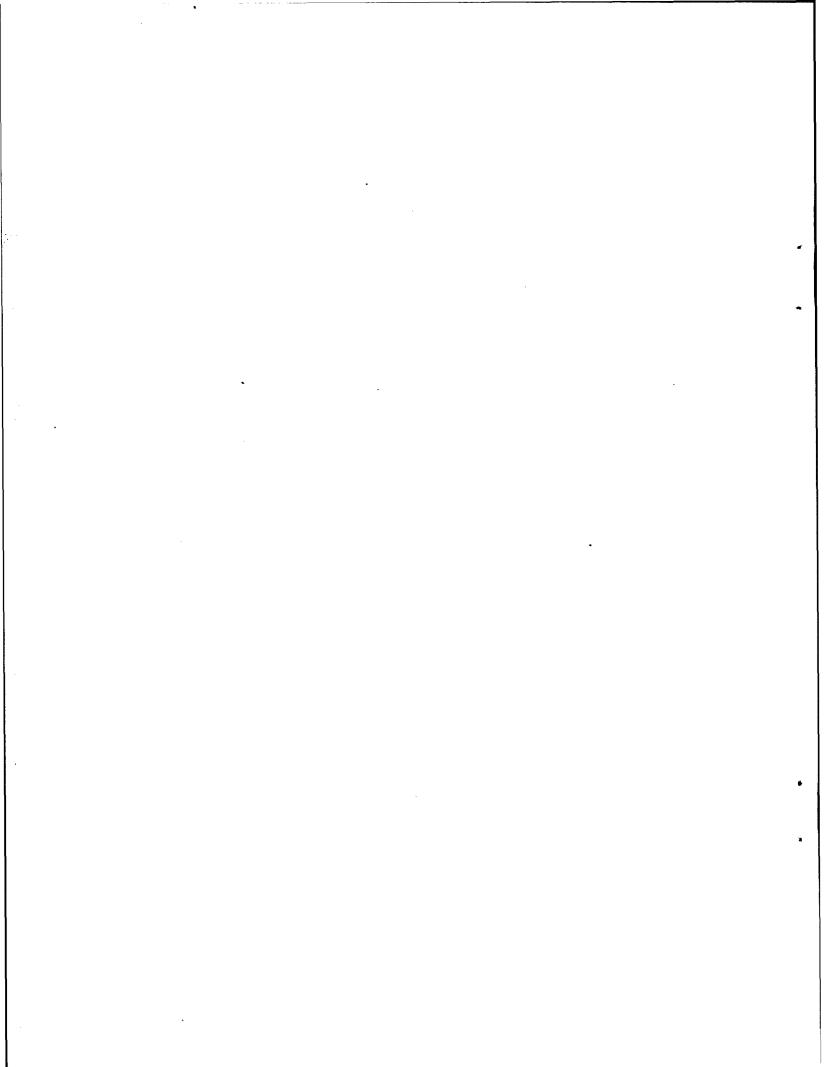
### I. TMEPS/MIA1 INTERCHANGE DATA POINTS.

NOTE: The following data points are to be collected as a minimum:

Power Level	NPT % (M1A1)	NPT % (TMEPS)
Maximum SHP	100.00	100.0
1200 SHP	99.0	. 98.7
900 SHP	94.0	96.3
600 SHP	83.0	87.5
Tactical Idle	43.3	43.3
Low Idle	30.0	30.0

### II. HANDLING AND STABILITY CHECK

- 1. Accelerate to 20 percent compensated power and record a long scan after a 5 minute stabilization. Mark P3 on the Gould recorder and log sheet.
- 2. Accelerate to 90 percent compensated power and record a long scan after a 5 minute stabilization. Mark P3 on the Gould recorder and log sheet at this power.
- 3. Set maximum power. Obtain a long scan after a 5 minute stabilization period.
- 4. Snap decelerate to idle, leaving water brake preset.
  Time deceleration to P3 (from step 1) noted on Gould.
  Initiate a transient during this sequence.
- 5. Stabilize at idle for 10 minutes.
- 6. Jam throttle to maximum power (throttle stop adjusted). Time to P3 noted on Gould in Step 2. Initiate a transient during this sequence.
- 7. Waveoff engine in 5 percent NH increments until 75 percent NH is reached. Waveoffs must be surge free. Initiate a transient during this sequence.
- 8. Decelerate to idle and shutdown after 10 minutes.



### APPENDIX II

### TMEPS/M1A1 ENGINE MODULE INTERCHANGEABILITY TESTING: PERFORMANCE RESULTS

### BACKGROUND

Since 1980 Textron Lycoming, General Dynamics Land Systems (GDLS) and Allison Transmission Division (ATD) have jointly been developing an improved derivative of the M1A1 Abrams Tank propulsion system. This system is termed the Transverse Mounted Engine Propulsion System (TMEPS). The TMEPS incorporates an advanced model AGT1500A engine mated to a new 7-speed ATD transmission tranversely mounted in the GDLS Abrams Tank. The contractural guarantee was to reduce the M1A1 Peacetime Annual Duty Cycle fuel usage (excluding APU usage) by 10 percent. In 1988, an AGT1500A engine demonstrated a Peacetime fuel usage reduction of 15.3 percent and 87°F day power of 1550 SHP.

Unique to the AGT1500A are a new power turbine optimized for part power fuel consumption; single crystal high pressure turbine rotor blades with reduced cooling flow; a new Digital Electronic Fuel Control Unit (DECU) scheduled to provide a significant increase in part power turbine inlet temperature and a Hastelloy-S recuperator with increased pre-load to improve effectiveness.

TMEPS hydraulic power is supplied by power takeoff from the transmission; accordingly the engine accessory gearbox module has been redesigned to eliminate the hydraulic pump power takeoff drive present in the AGT1500 gearbox.

Note, for clarity, herein the TMEPS engine is referred to as TME (rather than AGT1500A); the current production engine is referred to as the AGT1500.

### **PURPOSE**

The subject series of engine tests were designed to verify the functional interchangeability of engine modules between the AGT1500 and TME engine. This appendix documents the successful test demonstrations of this interchangeability feature in terms of overall engine performance.

TABLE I: TEST CONFIGURATIONS

	MODULE			TEST PURPOSE DEMONSTRATION		
TEST	FORWARD	REAR	CONTROL			
1 2 3 4 5 6 7 8	AGT1500 AGT1500 TME TME AGT1500 AGT1500 TME TME	AGT1500 AGT1500 TME TME TME TME AGT1500 AGT1500	AGT1500 ECU TME DECU TME DECU AGT1500 ECU AGT1500 ECU TME AGT1500 ECU TME AGT1500 ECU	AGT1500 Production Baseline TME DECU Effect on AGT1500 TME Baseline AGT1500 ECU Effect on TME TME Rear Module Effect on AGT1500 AGT1500 Front Module Effect on TME TME Front Module Effect on AGT1500 AGT1500 Rear Module Effect on TME		

### CONCLUSIONS

Overall engine functionality of module interchangeability was demonstrated by safe operation of all builds. The following table summarizes the overall performance effect of these tests.

TABLE II: MAX SHP, PART POWER SFC EFFECT

TEST	PURPOSE	MAXIMUM POWER (SHP)	SFC @1200 SHP 59°F DAY
1	AGT1500 Acceptance Test (20 HPX)	1508	0.486
	BASELINE TESTS	,	
1A	AGT1500 Baseline (0 HPX)	1464	0.488
3	TME Baseline	1566	0.448
	TME MODULES ON AGT1500 .		
7	TME Forward Module Effect on AGT1500	1446	0.490
5	TME Rear Module Effect on AGT1500	1452	0.471
2	TME DECU Effect on AGT1500	1553	0.483
	AGT1500 MODULES ON TME		
6	AGT1500 Forward Module Effect on TME	1548	0.458
8	AGT1500 Rear Module Effect on TME	1558	0.478
4	AGT1500 ECU Effect on TME	1473	0.456

### DISCUSSION

The TME engine has improved components in each module:

TABLE III: THE MODULE FEATURES

MODULE	FEATURES				
FORWARD	<ul> <li>Decreased interstage bleed</li> <li>LPC IGV closed setting changed to 35° to raise low power surge line</li> <li>Single crystal HPT blades with -1.5 percent cooling air</li> <li>Improved HPT efficiency (tighter running tip clearance)</li> </ul>				
REAR	<ul> <li>LPT nozzle with -0.3 percent cooling air</li> <li>Power turbine redesigned for improved part power efficiency</li> <li>-0.76 percent cooling air to power turbine reduction gearbox</li> <li>Recuperator with higher preload, improved effectiveness</li> <li>Increased travel power turbine linkage</li> <li>Enhanced intermodule sealing</li> </ul>				
DECU	• Schedules set to raise high power T5 25° and part power T5 120°				

The location of horsepower extraction for vehicle hydraulic power is a factor in evaluating module interchangeability effects. The source of hydraulic power on the TMEPS power pack is the transmission, while the M1A1 system uses power extracted from the AGT1500 high spool through the accessory gearbox module. Maximum throttle AGT1500 production engine power is controlled to 1500 SHP (+20, -0) by setting the fuel control speed bias to the high spool speed (NH) required to assure flat rated power up to 87°. This calibration, made on every production engine, is accomplished with a referee 20 SHP extraction (hpx) that is included in the quoted rated 1500 SHP. In contrast, due to transmission supplied accessory power, a TME engine is "trimmed" to max power (1545 SHP minimum) with no extraction from the engine high spool. To facilitate TME AGT1500 comparison in this test series, the baseline AGT1500 engine (Test 1) max throttle was calibrated to an NH yielding 1508 SHP per the Acceptance Test Spec ETS1500, with 20 hpx. Subsequently, while holding that speed bias NH setting fixed, another max throttle test point was acquired with 0 hpx. This test (1A) represents the baseline AGT1500 calibration for interchangeability comparative purposes. Removal of high spool power extraction is equivalent to an efficiency increase in the high spool (removal of a parasitic loss). This yields a cooler gas producer turbine inlet temperature and corresponding decrease in power at a fixed (previously "trimmed") NH.

All control variables (compressor inlet guide vane schedule; compressor intercompressor bleed schedule and power turbine maximum area) were maintained at base engine values (Tests 1A and 3); no retrimming was done. This test series therefore simulated a module change in a field environment. This is in contrast to a module change at a repair depot where an engine test cell calibration would facilitate optimization of engine variables, including (degradation permitting) retrim of maximum power to a desired value.

Table II illustrates the effects of discretely placing TME modules on the AGT1500 (Tests 7, 5 and 2 relative to 1A) and conversely, AGT1500 modules on the TME (Tests 6, 8 and 4 relative to 3).

TME Modules Incorporated on the AGT1500: TME forward and rear modules had minor effects on AGT1500 maximum power (Tests 7 and 5). Due to a high-side of nominal overall compressor efficiency on the baseline AGT1500 relative to a nominal level on the TME, the benefits inherent in this module (Table III) are negated, resulting in a slight power decrease and SFC increase (Test 7). Advantages offered in part power SFC due to the TME power turbine was illustrated in Test 5. Note, while the TME power turbine was optimized to provide peak efficiency in the part power region an efficiency penalty may be encountered at the open geometry settings run by the AGT1500 (larger than TME), resulting in a power loss at a fixed ECU NH (Test 5 relative to 1A).

The TME DECU (Test 2) significantly increases AGT1500 power due to higher trim NH and high pressure turbine inlet temperature (T5). Test 2 also reflects the SFC advantage of the increased T5 scheduled by the TME DECU, however note the full part power temperature potential offered by the TME DECU cannot be fully achieved with the available AGT1500 power turbine area variability. That is, in the absence of the TME increased travel linkage, the AGT1500 power turbine cannot close enough to attain the front slope MGT-NH schedule (and therefore T5) the DECU requests. Refer to Figure 2, comparison of steady state TME DECU and AGT1500 ECU MGT NH schedules.

AGT1500 Modules Incorporated on the TME: AGT1500 modules, as expected, had an effect on baseline TME power, however, the configurations were safely functional. The max power loss due to module change was less than 1-1/4 percent (Tests 6 and 8). The significant increase in SFC upon inserting the AGT1500 rear module reflects the inherent design modifications in the TME power turbine to improve peak and part power efficiency and the higher effectiveness recuperator. The power loss experienced upon incorporation of the AGT1500 ECU (Test 4) is due to the 25° cooler T5 this control schedules and the max throttle speed bias (NH) setting which is 0.6 percent NH lower. Note: approximately 50 SHP could be recovered if the AGT1500 ECU NH were retrimmed to the baseline TME level.

Idle Performance: At a common 35 SHP the TME baseline (Test 3) demonstrated a 29 percent lower fuel flow than the AGT1500 (Test 1). Interchangeability tests demonstrated that the dominant factor is the gas producer inlet temperature (T5) difference due to the MGT-NH control difference.

Several physical differences between AGT1500 and TME interact to produce the idle region fuel consumption comparison displayed on Figure 9.

		GOVERNING MODULE	FIGURE
•	MGT-NH Schedule	DECU/ECU	2
•	Low Compressor IGV Schedule; effect on surge line	FORWARD	10
•	Power Turbine Variable Geometry Travel Capability	REAR	
•	Interstage Bleed	DECU/ECU	11

These factors combine to yield the following matrix of low power configurations and accordingly impact idle fuel flow and cycle matching on the Low Compressor map (Figure 10) and MGT-NH Schedule (Figure 2).

TEST	1	DULE REAR	CONTL	MAX CLOSED IGV(°)	INTER- STAGE BLEED	POWER TURBINE SCHEDULED ACTION	GENERAL MGT-NH LEVEL ATTAINED
1 2 3 4 5 6 7 8	A A T T A A T	A A T T T A A	A T T A A T	30 30 35 35 30 30 35 35	Full Reduced Reduced Full Reduced Full Reduced	Full Open Scheduled to MGT-NH Scheduled to MGT-NH Full Open Full Open Scheduled to MGT-NH Full Open Scheduled to MGT-NH	TME AGT AGT TME AGT

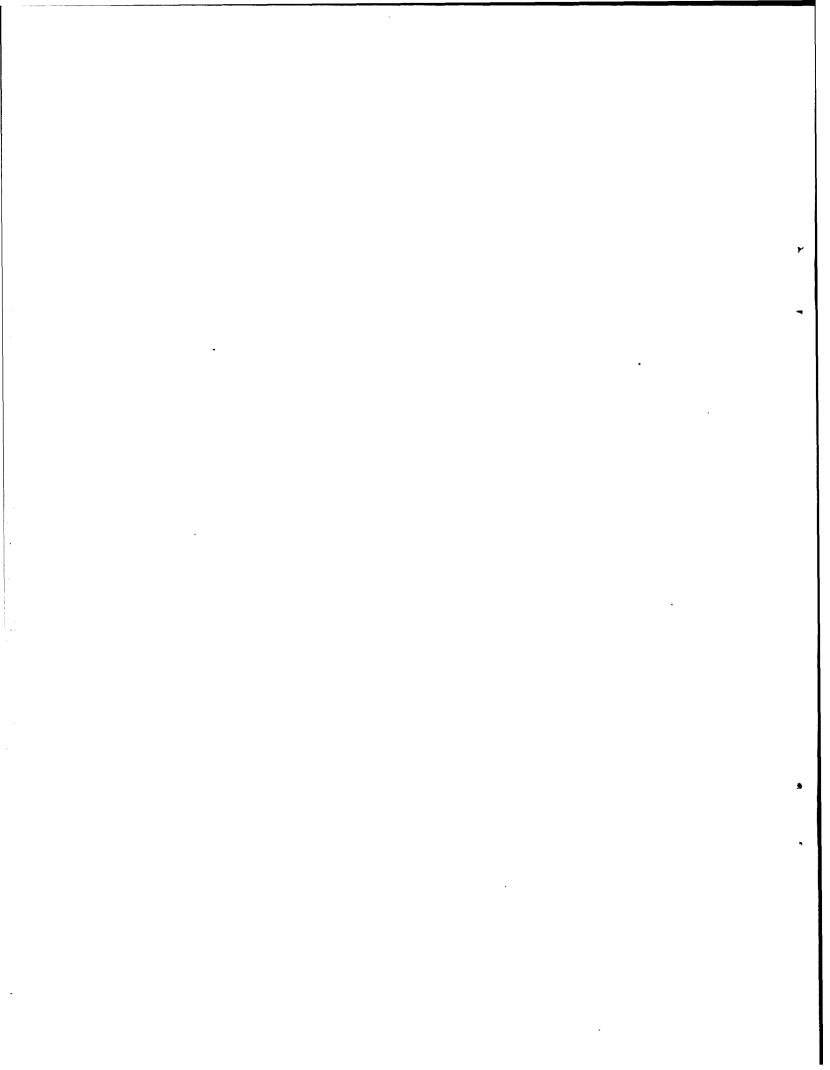
A: AGT1500 T: TME

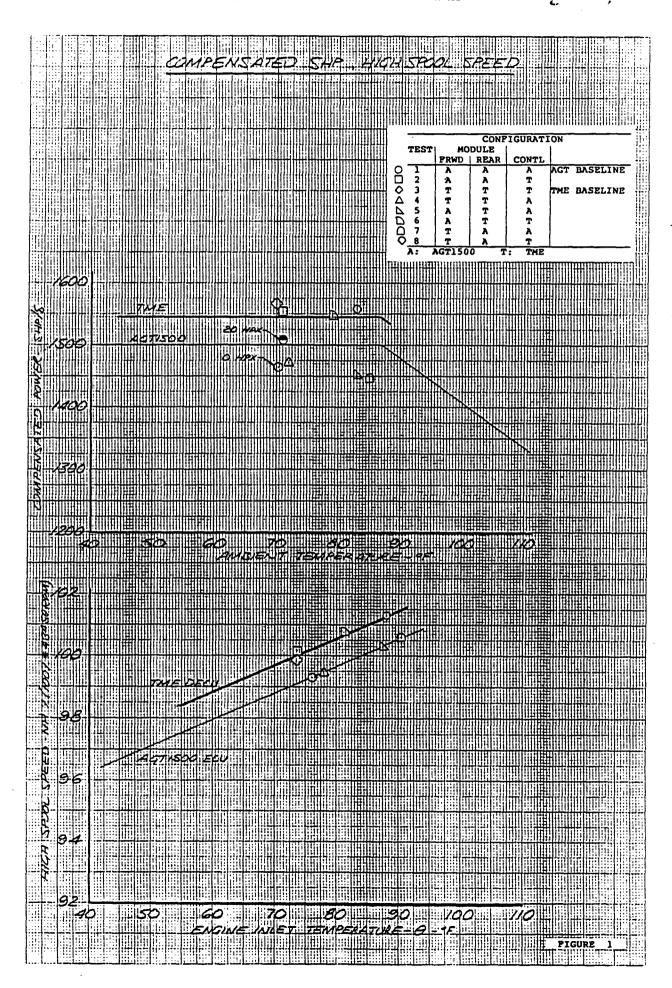
On a percentage change, the AGT ECU increased (penalized) TME fuel flow more than the TME DECU improved AGT fuel flow. This is due to the absence of the power turbine increased travel capability in the AGT engine which prevents it from attaining the TME schedule (Figure 2) and capitalizing fully on the increased T5 potential. This is illustrated on Figure 2 by Tests 2 and 8. Note interchange of the forward modules has the least effect on idle fuel flow.

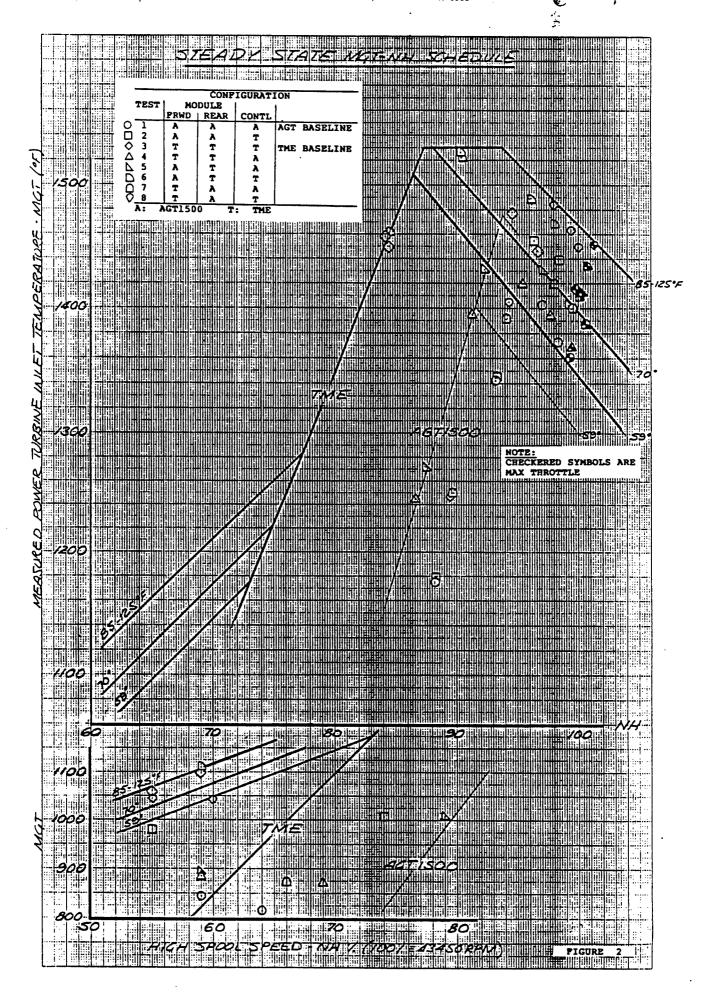
Figure 10 displays low compressor loading in the idle region. Note the improved surge line offered by the more closed IGV setting. Two distinct load lines were attained. The higher was attained by configurations with reduced bleed (Tests 2, 3, 6, 8). The lower load line is composed of configurations with full interstage bleed (Tests 1, 4, 5, 7). All configurations operated surge free.

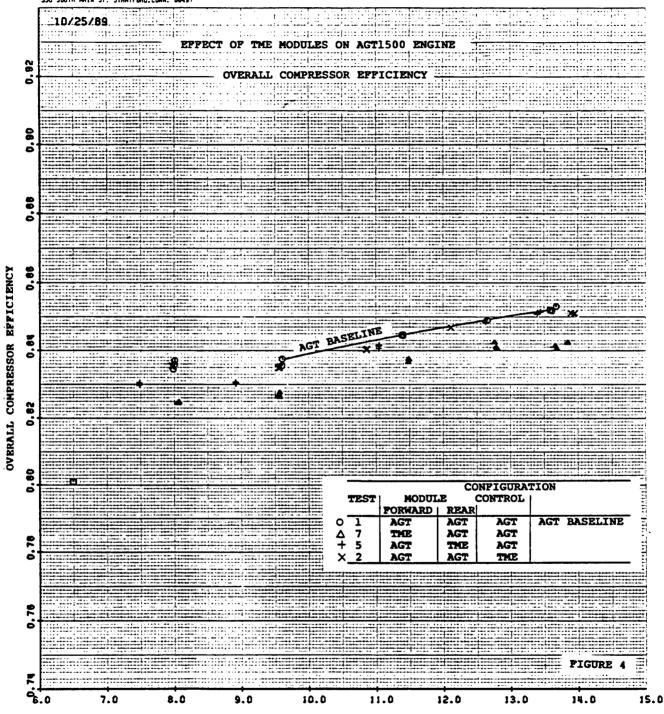
### CONCLUSIONS

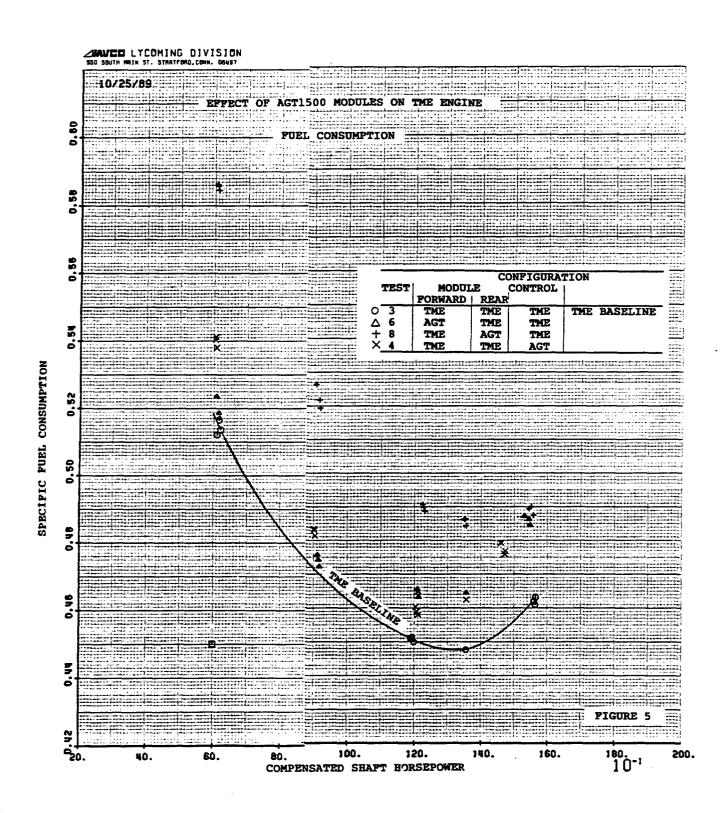
- 1. The contractual requirement to verify interchangeability of engine modules between the M1Al and TMEPS configurations was achieved.
- 2. All configurations operated stably and surge free during all transient movements.



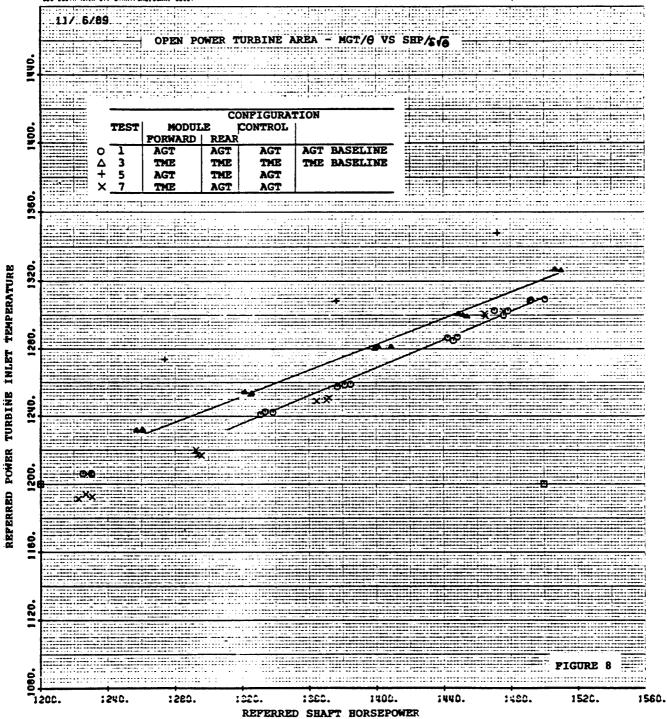






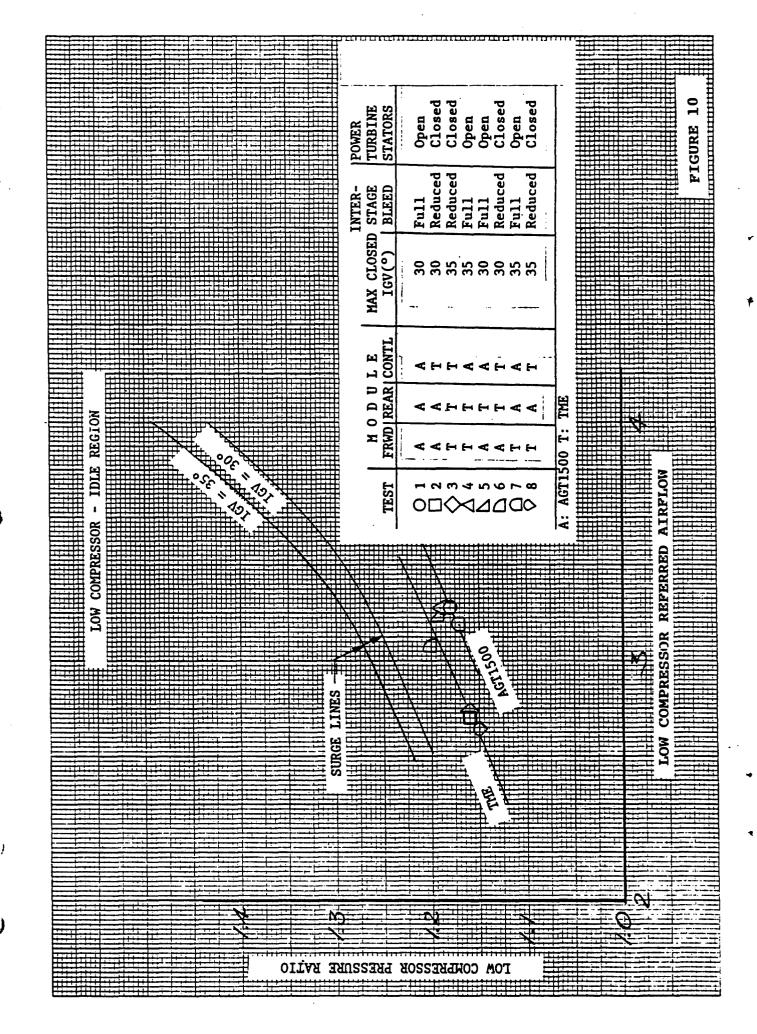


OVERALL COMPRESSOR PRESSURE RATIO



)

	IDLE PE	RFORMANCE	
	CON TEST MODULE FRWD REAR		
	$ \begin{array}{c cccc} 0 & 1 & A & A \\ 2 & A & A \\ 3 & T & T \\ 4 & T & T \\ 5 & A & T \\ 6 & A & T \end{array} $	A AGT BASELINE T TME BASELINE A A T	
	☐ 7	T: TME	
/00			
00v - WE/6√0			
REFERRED FUEL FLOW			
REFERENCE TO THE PERSON OF THE			
	20 AO REFERRED SHAFT HORS	60 S6 SEPOWER - SHP/8√9	FIGURE 9



•

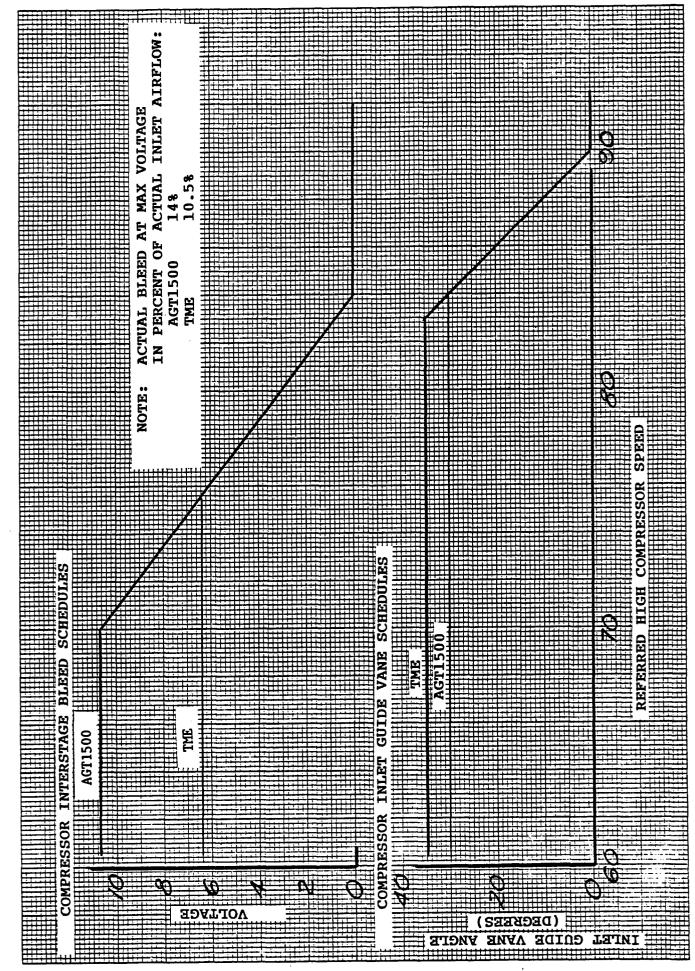


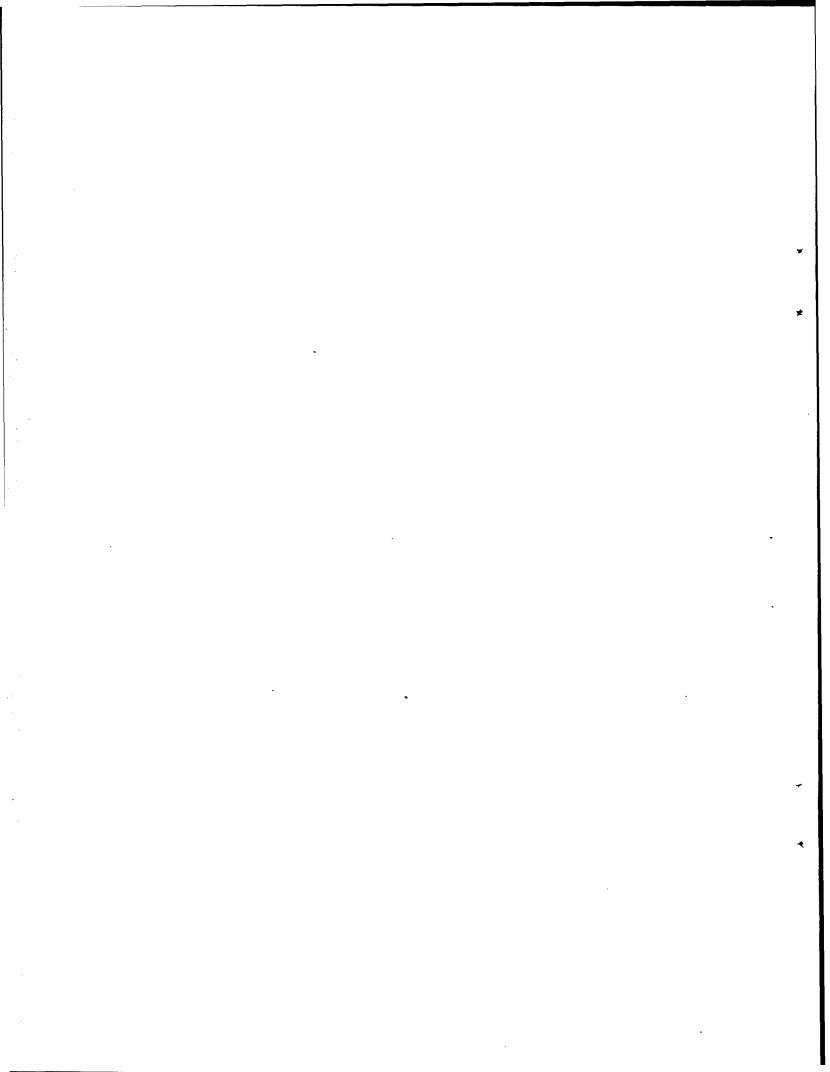
FIGURE 11

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APPENDIX II

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Report No. LYC 88-39
AGT 1500A
TACOM TME FUEL ECONOMY DEMONSTRATION TEST
ENGINE T202N
October 20, 1988

Prepared by

F. Macri

AGT1500 Test Engineer

Approved by

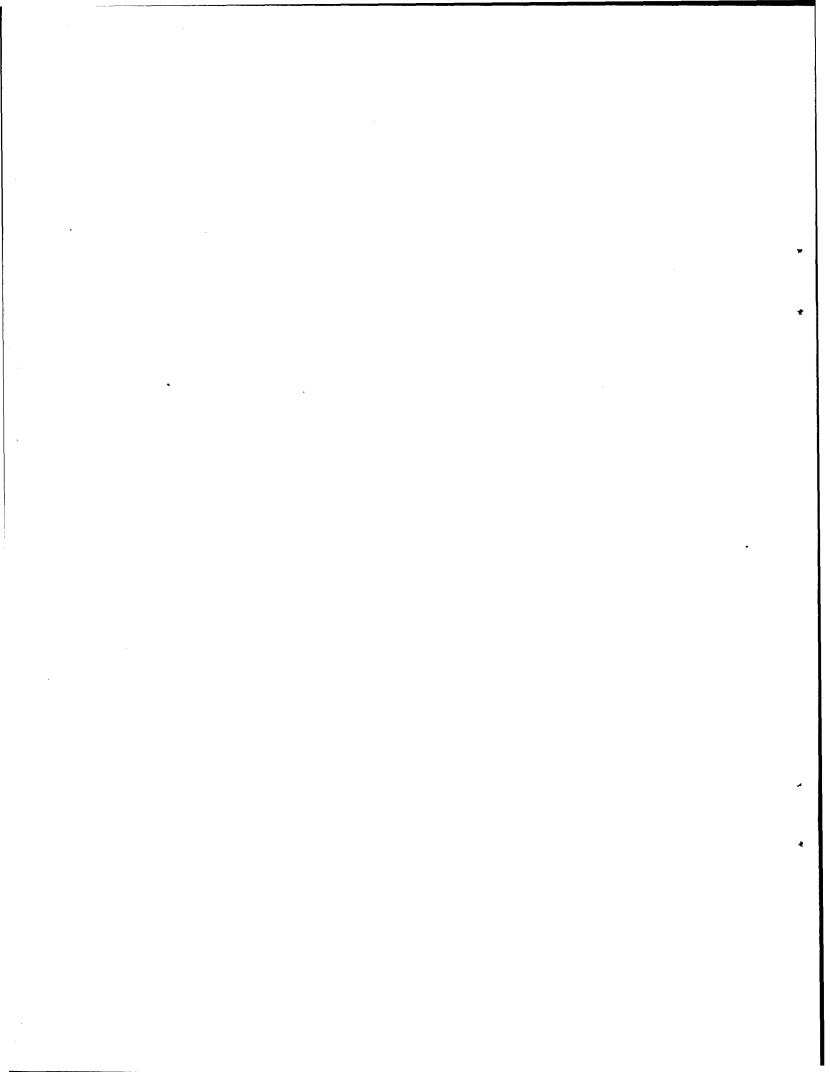
R. Hudson

AGT 1500 Engine Test Manager

Concurred by

R. Horan

TME Development Manager



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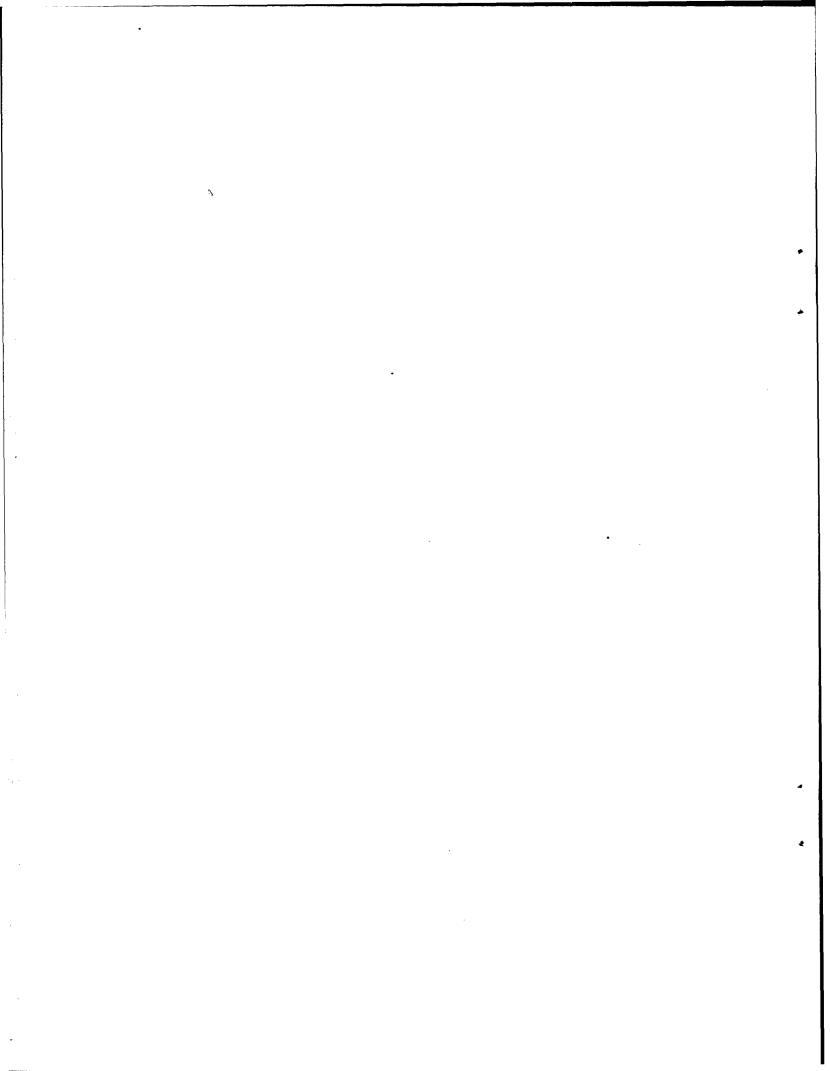
APPENDIX - AGT 1500A TME Fuel Economy Test Plan

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#### **SUMMARY**

Engine S/N T202, Build N, successfully completed a demonstration of TME performance on 20 October 1988 in accordance with Test Plan LYC 88-28 during which all objectives were accomplished. This report documents results obtained during testing at Textron Lycoming, Stratford, Connecticut. Engine operation was conducted at an ambient temperature of 87  $\pm$  5 Degrees F and prevailing barometric pressure.

Weighted fuel savings relative to the current M1A1 production fabrication specification (E2180C) based on the peacetime annual usage duty cycle (excluding normal idle points) achieved during the demonstration were calculated at 15.3 percent. Fuel savings estimated for battlefield day operation were 12.2 percent on an annual basis.

Optimum line engine specific fuel consumption (SFC) was evaluated at 600, 900, 1200 and trim SHP power levels.

Maximum power achieved during the test was 1560 SHP (1550 SHPC) which exceeded the contracted power commitment of 1545 SHP. Testing was conducted over a range of steady-state operating conditions from idle to maximum power. Engine transient maneuvers were also successfully demonstrated. Total run time during the demonstration was 9.97 hours.

All requirements specified in the TME contract DAC100001 were successfully demonstrated with representatives from the Army (TACOM), General Dynamics Land Systems (GDLS) and Allison Transmission Division (ATD) present.

#### **BACKGROUND**

During 1986, Textron Lycoming began development of an improved version of the M1A1 Abrams Tank Propulsion System. This advanced design powerpack termed TMEPS (Transverse Mounted Engine Propulsion System) is part of a joint effort with GDLS (General Dynamics Lands Systems), Allison Transmission Division (ATD) and Donaldson (SCAF) in which the goal is to reduce propulsion compartment volume in the current M1 design by 30 percent (See Figures 1 & 2) to allow for additional munitions. In addition, engine operating costs would be lower mainly through a reduction in fuel consumption.

Like the AGT 1500, the TME system is a recuperative free power turbine shaft engine which incorporates a two-spool compressor and variable power turbine stators. The Transverse Mount Engine also incorporates a digital electronic fuel control unit (DECU) with additional engine schedule changes and more advanced components including a Hastelloy S material recuperator, advanced design power turbines (FEP) and reduced cooling flow single crystal H.P. turbine blades. The engine accessory gearbox has also been redesigned in which the customer power takeoff pad is removed (The hydraulic pump is driven off the vehicle mounted accessory gearbox in the TMEPS).

A significant contractual requirement of the program is the ability of the engine to achieve no less than ten percent weighted fuel savings relative to MIAI Peacetime Mission cycle (non-normal idle points) and an increased power commitment of 1545 SHP. This is basically achieved through improved design hardware and operation of the engine on a more efficient fuel control schedule.

Featured hardware in the TME build is as follows:

- HPT Nozzle 3-110-250-X70, S/N BD8076, EFA = 5.342, GFA = 5.72
- HPT Cylinder 3-110-400-18, S/N 1151
- HPT Wheel Rebladed 3-110-010-88, S/N M807037
- LP Nozzle 3-110-140-33, S/N 202H EFA = 14.34
- New LP Turbine Wheel 3-110-120-11, S/N 88E17
- RGB TBC Double Wall Deflector 3-022-010-04
- RGB Reduce Cooling Flow Turbe 3-162-030R01, S/N 85B031
- RGB Assembly 3-020-400-X10, S/N 0044
- LE9022 Hp & LP Compressors New Bearings and Seals
- Int Housing Assembly 3-105-010 (HSGN 3-105-002R34) Special Sta. 2.1 Instrumentation, S/N 4899
- TME Diffuser Liner 3-130-070-16
- TME Collector 3-130-090-41
- Rope Seal Band Assembly 3-150-350X01
- New Curl Assembly 3-130-020-15
- PT HSGN 3-142-020-01, S/N 82B029
- AGB Module 3-000-050-06, S/N 85B035

#### METHOD OF TEST

Engine T202, Build N, completed both steady-state and transient operation per procedures set forth in Lycoming Test Plan LYC 88-28 (See Appendix 1) which was reviewed and approved by GDLS. Testing was completed on 20 October 1988 at the Lycoming Division of Textron in Stratford, Connecticut, with personnel from TACOM, ATD and GDLS present. The engine test cell utilized for the demonstration was T-11. The test was conducted at an ambient temperature of  $87 \pm 5$  Degrees F and corrections to standard day barometer were made.

Engine operation was performed with fuel conforming to VV-F-800, Grade DF-2 diesel fuel, and lubricating oil conforming to MIL-L-23699.

The test procedure conducted during the demonstration was completed in the following order:

- Peacetime Mission Cycle Calibration
- TME Tt7 PTS Tracking Calibration
- Engine Transient Evaluation
- Optimum Area Fixed PTS Calibration

Each procedure will be discussed below.

### Peacetime Mission Cycle Calibration

Power levels associated with the simulated Peacetime Mission cycle were set in accordance with GDLS input and account for all projected system power demands (losses). Actual points required are summarized in Table 1. Additional power levels were set above and below the specified points to allow for preliminary analysis of data in the test cell. Stabilization time at each point was ten minutes. Following the initial start, a fuel sample was collected for lower heating value analysis to allow for corrections to be made from the assumed 18500 BTU/LBM value.

#### TME Tt7 - PTS Tracking Calibration

Engine steady-state operation was conducted during the demonstration in which powers ranging from idle to trim power were obtained. Power levels were set in descending order from trim power to idle in 100 SHP increments. Total stabilization time at each point was five minutes.

#### **Engine Transient Evaluation**

Engine transient maneuvers were completed as specified in attachment three of LYC 88-28 (power and speeds updated for TME operation).

## Optimum Area - Fixed PTS Calibration

A five-point open area calibration was conducted at a fixed setting of the PTS (power turbine stator) position as required in the test procedure. Stabilization time at each point was five minutes.

#### TEST EQUIPMENT

#### Power Absorption

A Textron Lycoming (LC 28800) waterbrake was used to absorb engine output shaft horsepower. The brake is supported from the engine on four calibrated strain-gauged beams that sense engine output shaft torque. Conversion of the strain-gauge signal to torque (Ft-Lbf) was accomplished by a transducer with signal conditioning to the data acquisition system.

No power extraction was facilitated during testing since TME design does not include an engine customer power takeoff pad.

#### Engine Starting System

Starting of the engine was accomplished by a Delco Remy, Model 1113883, electronic stepper motor (See Figure 3 for TME starter pad configuration). Power for starting was provided by a 28 VDC Hobart, Model No. 6T28-400CL, motor generator.

#### Oil Coolers

Engine oil (Mobil 254) was cooled by means of an industrial type oil to water heat exchanger, Lycoming TES 77-1.

### Engine Inlet Air Temperature

Engine inlet air temperature was maintained at  $87 \pm 5$  Degrees F from a tempered air system comprised of 600 SHP motor with a plenum assembly, hydraulic valves, and a variable speed blower with steam heating coils. All operation is monitored by a computer.

### Computerized Data Acquisition Equipment

Engine test data was recorded by a Hewlett Packard HP1000 data acquisition system comprised of a digital processor (via disk drive), printer and CRT displays. The system is capable of processing steady-state and transient data collection.

#### Inlet Airflow Measurement

Engine inlet airflow was measured with a calibrated axial bellmouth assembly TE 26816-01 per ASME standards. Static and total pressures in the bellmouth were measured using pressure transducers in conjunction with a suitable analog to digital digital converter. The inner bellmouth housed the NL rotor speed pickup.

### Vibration

Engine vibrations were measured with CEC/IMO velocity transducers in conjunction with Trig-Tek vibration meters, Model No. 203J.

#### Fuel and Oil Measurement

Cox turbine flowmeters (ANC) with associated signal converters, amplifiers and readouts were used for measuring oil and fuel flow rates.

#### Rotor Speeds

Magnetic pulse generators in conjunction with signal conditioning and readout equipment were used to measure and display main rotor speeds.

#### Pressures

Calibrated bourdan tube gauges and transducers measured pneumatic and hydraulic pressures.

#### Temperatures

Temperatures were measured by I.C. and C.A. thermocouples. Signals were conditioned by analog to digital converts and displayed by means of a printer and CRT.

### Transient Recorders

Gould Brush, Model 2800, recorders were used during the demonstration in conjunction with pressure transducers, flowmeters, thermocouples, potentiometers and magnetic speed pickups to provide continuous monitoring of the engine. A high speed analog to digital converter manufactured by Neff Instruments was also utilized in recording transient data.

### Oil Level Measurement

A Robertshaw controls, Model 5000, and indicator probe were used to measure engine oil level.

#### TEST RESULTS

Total engine run time accumulated during the demonstration was 9.97 hours (total run time accumulated on T202N to date is 43.12 hours). Engine oil consumption calculated during testing was .02 gal/hr. Results obtained from each section of the test procedure are summarized below.

### Peacetime Duty Cycle - Weighted Fuel Savings

Weighted annual fuel consumption summarized for each engine power condition of the Peacetime Mission points (Table 1) is shown in Table 2. Corrections for an installed configuration at 500 feet altitude (See Figures 4 & 5) and actual measured fuel lower heating value was accounted for.

Uninstalled optimum engine specific fuel consumption measured at each Peacetime Mission point is shown in Figure 7.

Based on test results, engine T202N demonstrated 15.3 percent fuel savings relative to the M1 production specification Peacetime Mission cycle (normal idle points excluded). Engine weighted fuel savings estimated for the M1A1 Combat Mission cycle (battlefield day) were calculated at 12.2 percent on an annual basis (See Table 3). The largest fuel savings occurred at the tactical idle power setting in which an estimate of 217 gallons of fuel would be saved (M1A1 spec - 740 gallons, TME demonstration - 523 gallons) which represents a 29 percent reduction in annual fuel consumption during peacetime operation. This is a significant achievement because a large percentage of tank operation occurs in the low power region.

### TME - Performance Calibration

Engine performance calibrations (tracking and open area) conducted during the demonstration were completed in compliance with the TME ETP 1500A Test Specification. Data obtained is plotted in Figures 7 through 15.

Engine specific fuel consumption (See Figure 12) measured during the tracking calibration at each of the specified points noted in attachment three of LYC 88-28 is summarized below:

Power (SHP)	NPT <u>(%)</u>	SFC (LBM/HP-HR)
1500	100.0	.459
1200	98 <b>.</b> 7 ·	.451
900	96.3	.462
600	87.5	.507
Tac Idle	44.0	1.309
Idle	37.0	1.493

The TME maximum power commitment of 1545 SHP (87 Degrees F) was exceeded during testing. Engine T202N demonstrated 1550 SHPC with a 27 Degree F H.P. turbine inlet temperature margin (Tt5 limit = 2230 Degrees F). Engine trim shaft horsepower characteristics are shown below:

TO	85 Degrees F
Power	1550 SHP
NH	103 Percent (100% = $43450 \text{ RPM}$ )
Tt5	2203 Degrees F (2230 limit)
Tt7	1432 Degrees F
PTS	-5.35 VDC

### Engine Transient Performance

Results from engine transients conducted during the demonstration are shown below (See Figures 16-18):

Condition	T202	Requirement
Decel (20% SHPC)	4.8 seconds	5.0 seconds
Accel (90% SHPC)	3.8 seconds	4.0 seconds

Engine waveoffs to 75 percent NH - surge free.

Additional features incorporated into the DECU for TME operation during engine transient operation include schedule modifications to reduce temperature gradients.

#### CONCLUSIONS

In compliance with TME Contract No. DAC100001, engine T202N successfully demonstrated all objectives described in Lycoming Test Plan LYC 88-28.

Weighted fuel savings estimated for both peacetime and battlefield day operation were 15.3 and 12.2 percent, respectively, when compared to the current M1A1 fabrication specification. In addition, significant part-power optimum SFC reduction was demonstrated.

Maximum shaft horsepower demonstrated during the test was 1550 SHPC (87°F day commitment - 1545 SHPC) with a 27°F Tt5 margin.

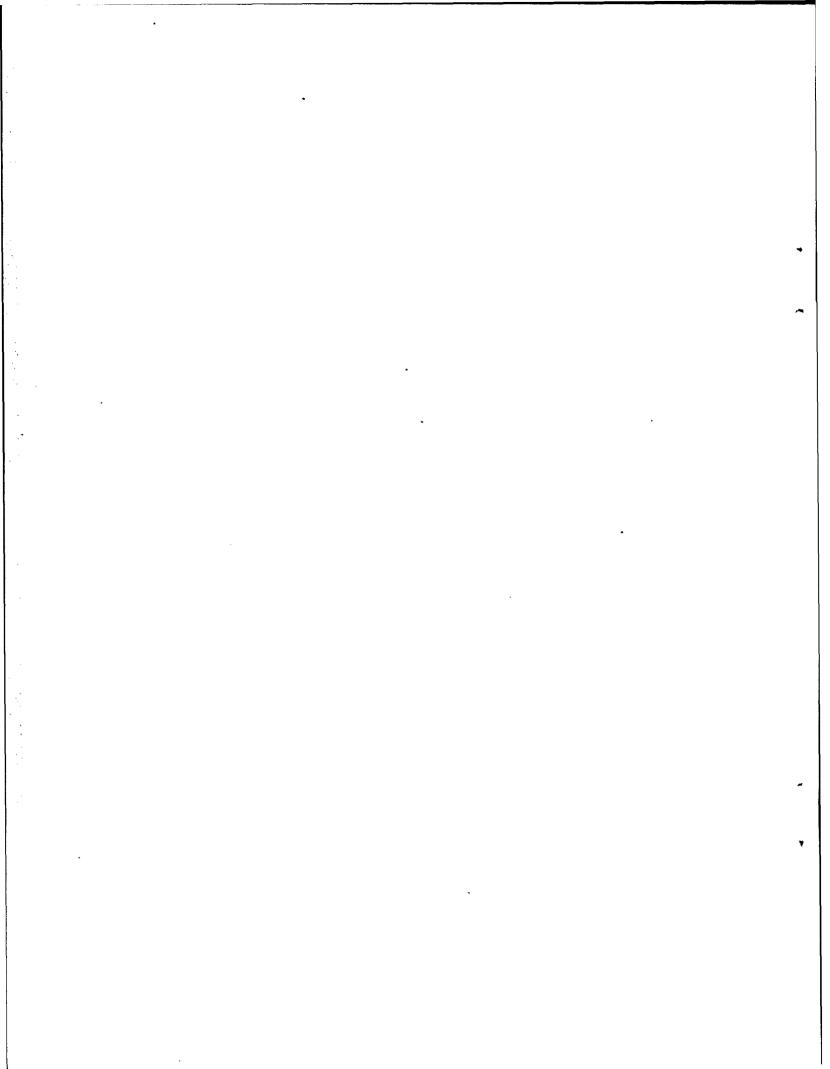
Transient performance evaluated during starting and accelerations indicate that recuperator durability will be increased with the additional logic incorporated into the TME DECU. This is accomplished by limiting recuperator core temperature gradients during transient operation.

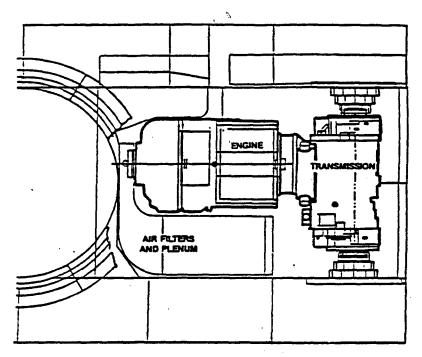
#### **RECOMMENDATIONS**

It is recommended that this report be accepted as evidence of the capability of AGT 1500A engine T202N to demonstrate transverse mounted engine operating principles as required in TME Contract No. DAC100001.

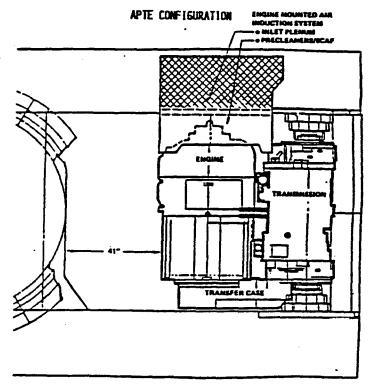
It is further recommended that engine T202N complete a Final Acceptance Test as specified in ETP 1500A prior to shipment from Textron Lycoming for further development with the ATD seven-speed transmission and ATR vehicle.

FIGURES

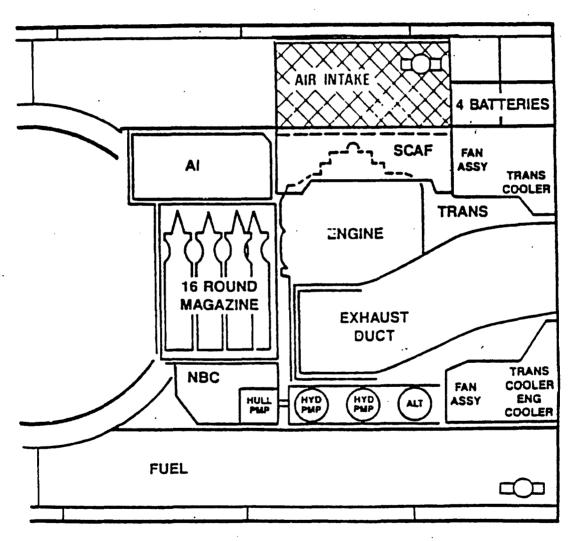




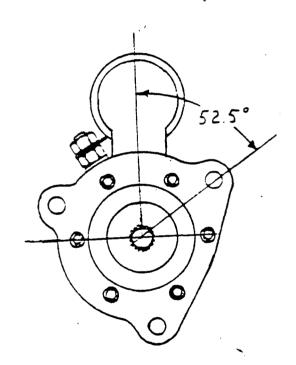
Present AGT-1500 Engine Arrangement

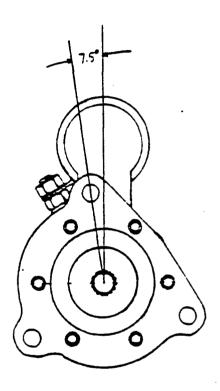


APTE AGT-1500 Engine Arrangement



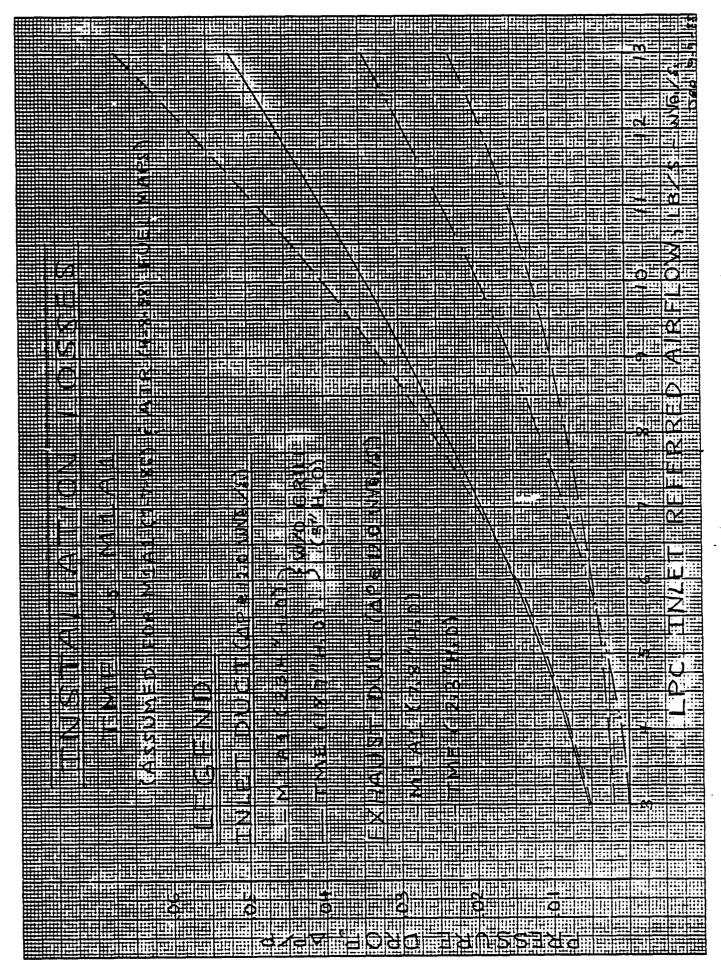
Lycoming TEXTRON





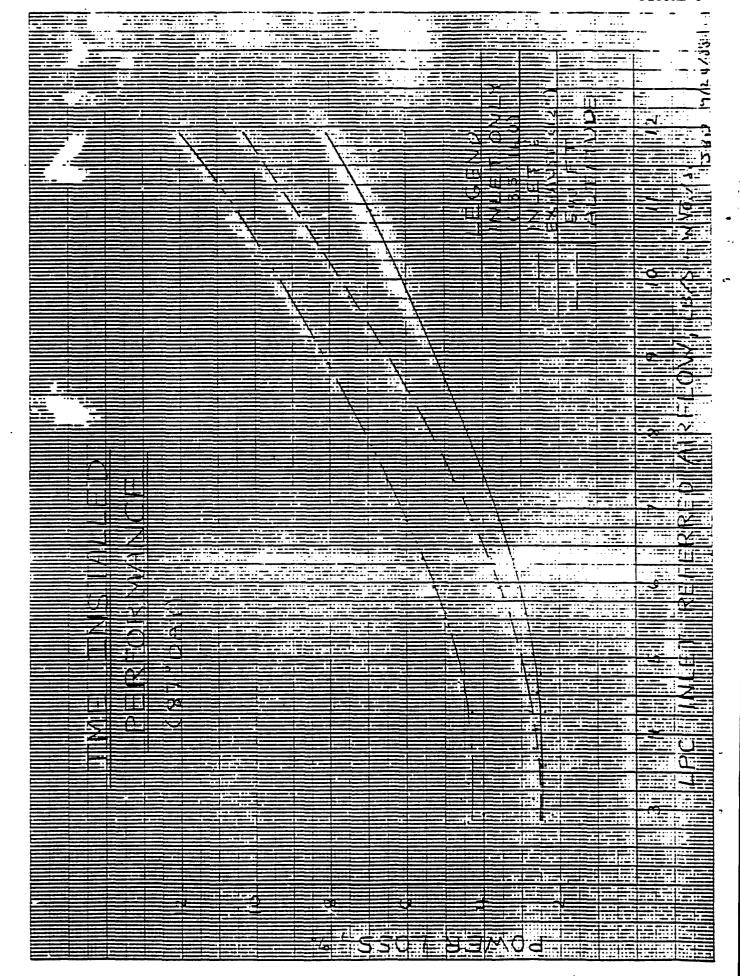
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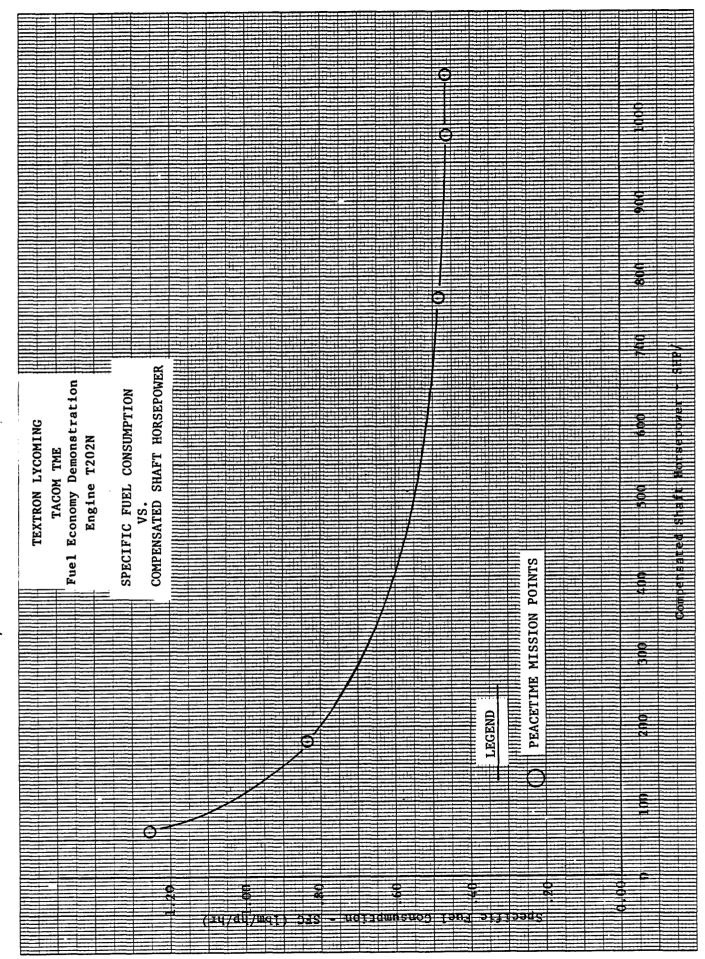
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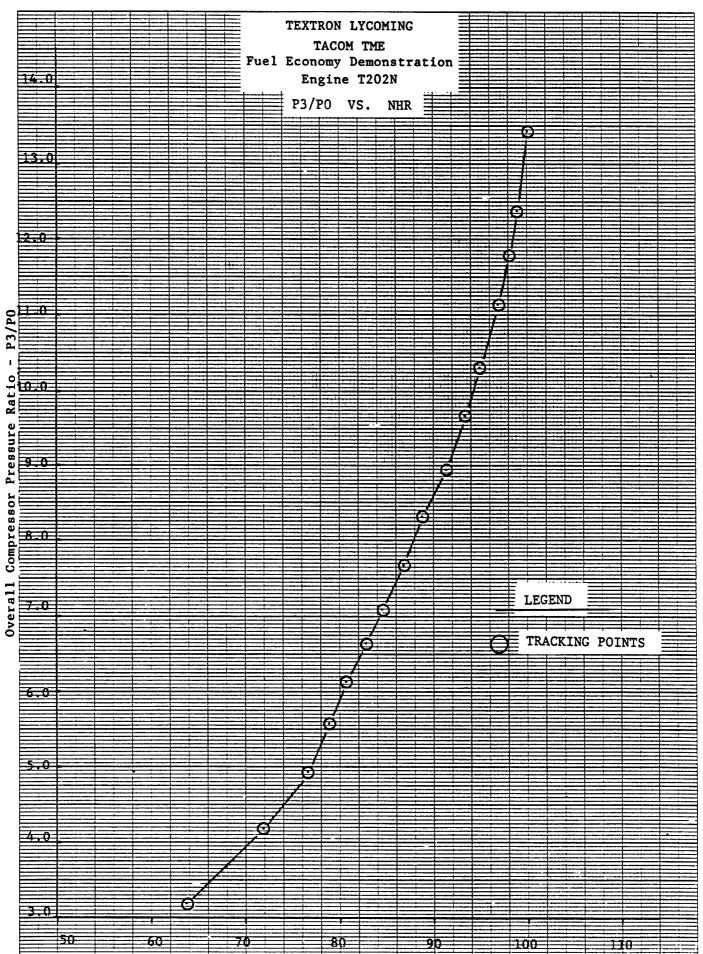


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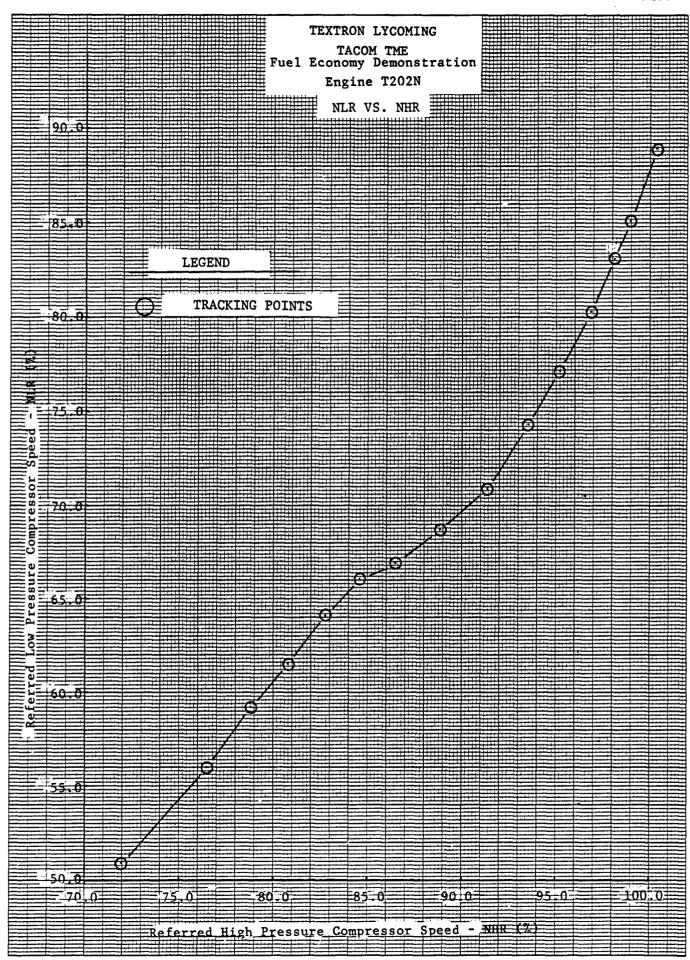
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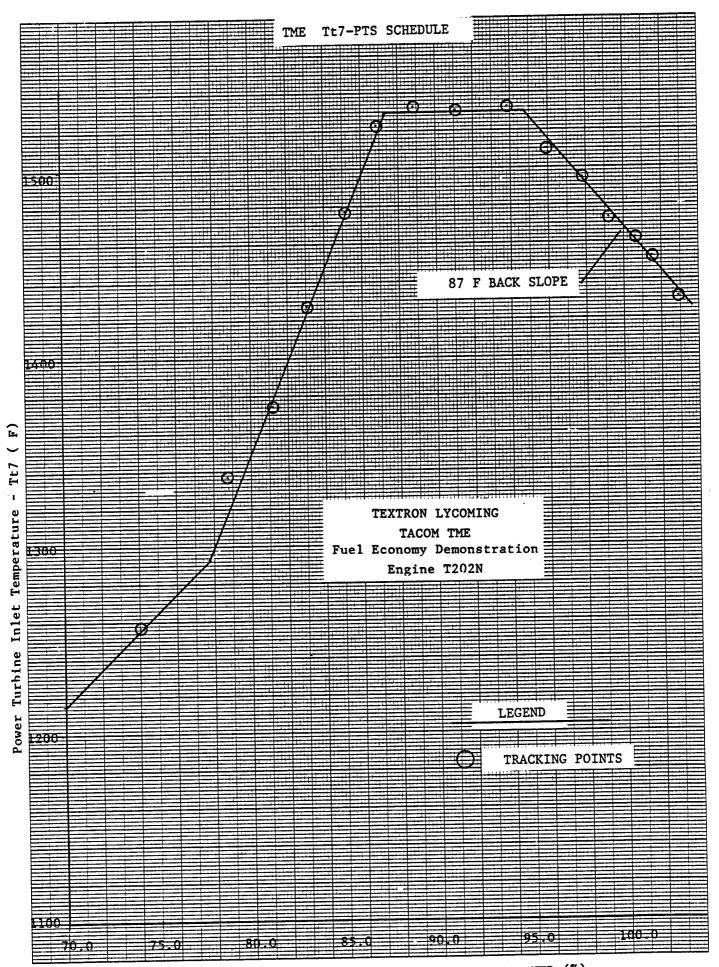


Referred High Pressure Compressor Speed- NHR(%)



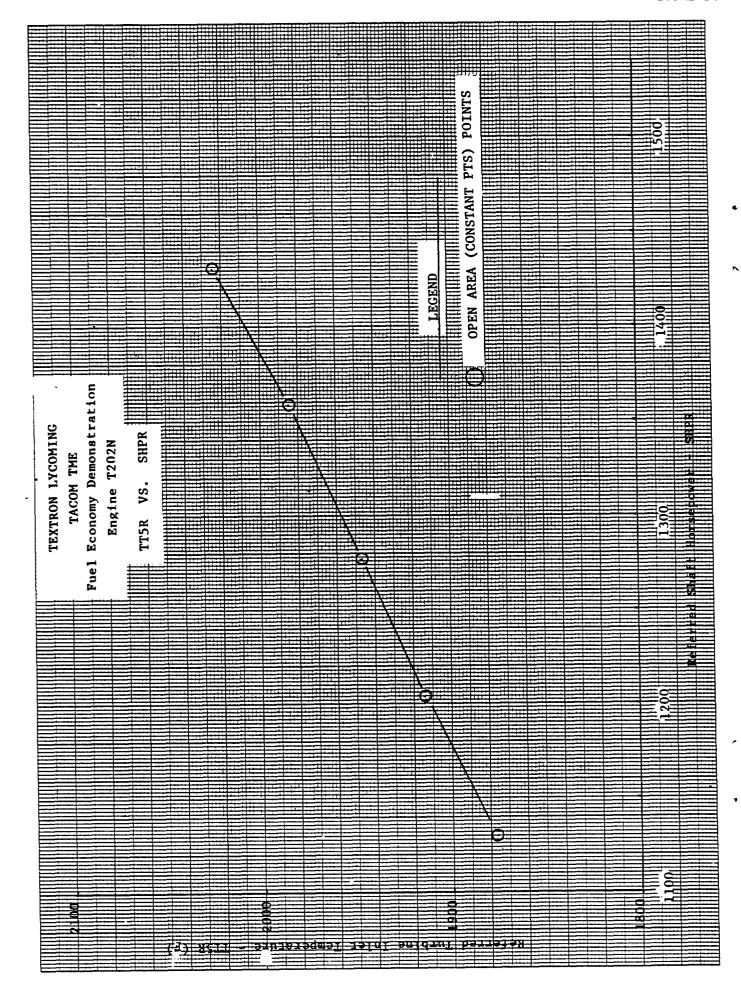
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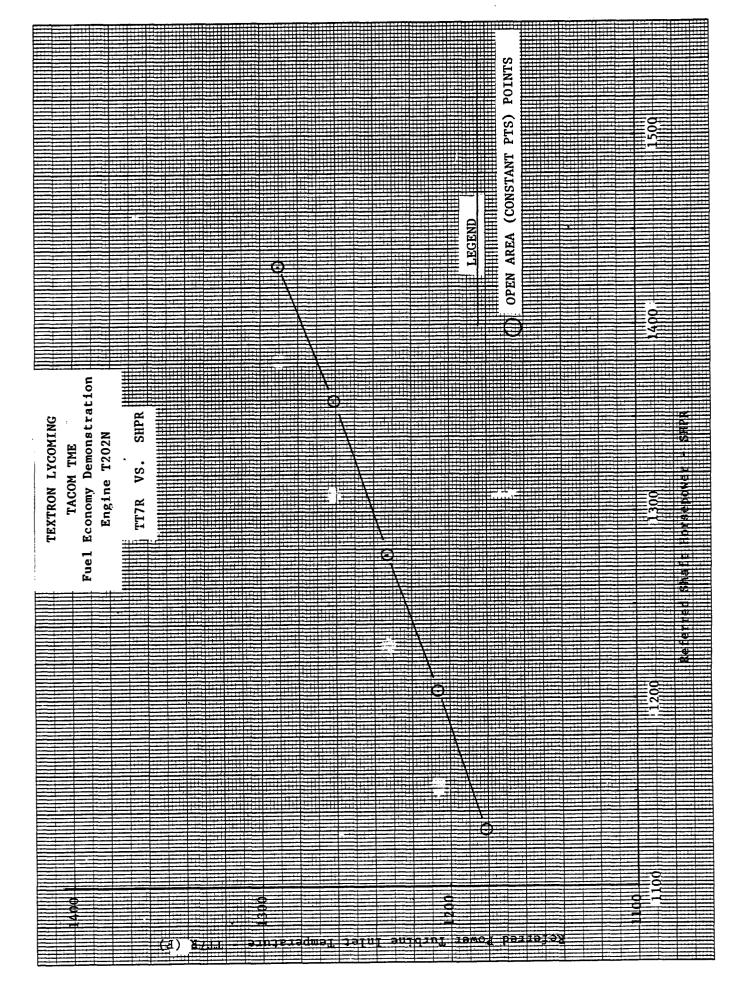
Referred Shaft Horsepower - SHPR



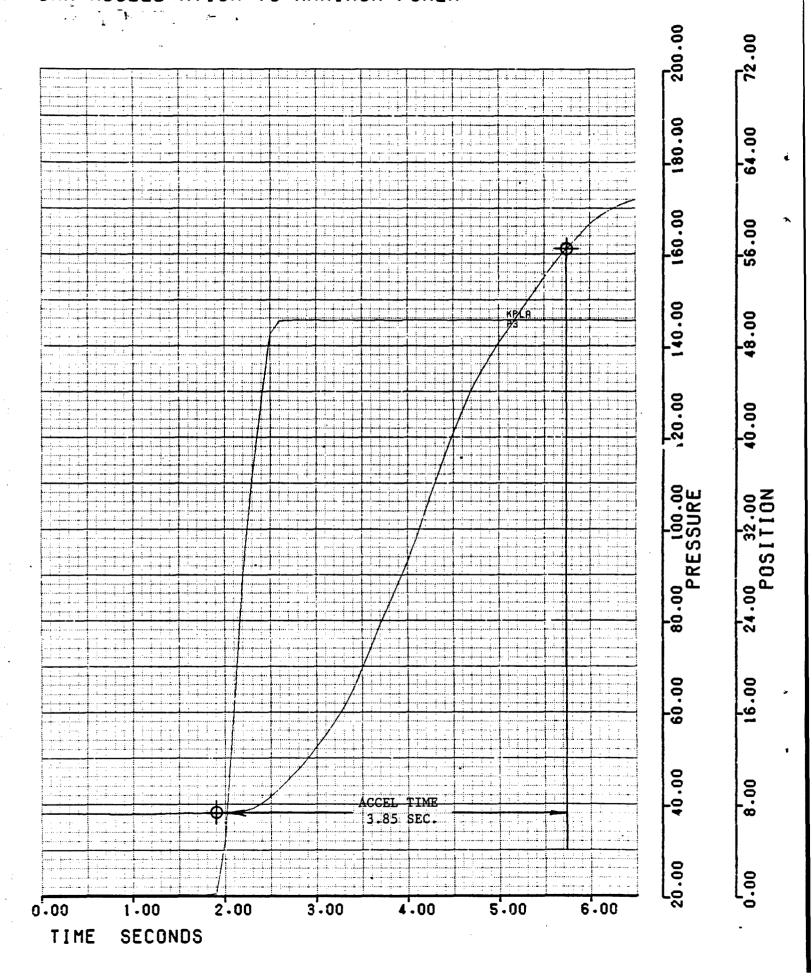
Referred High Pressure Compressor Speed - NHR (%)

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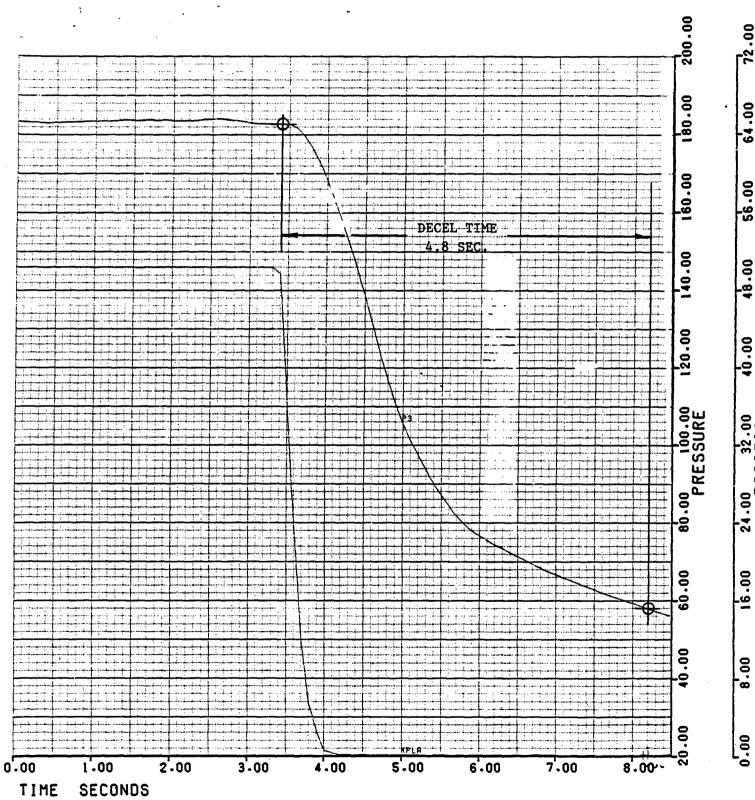




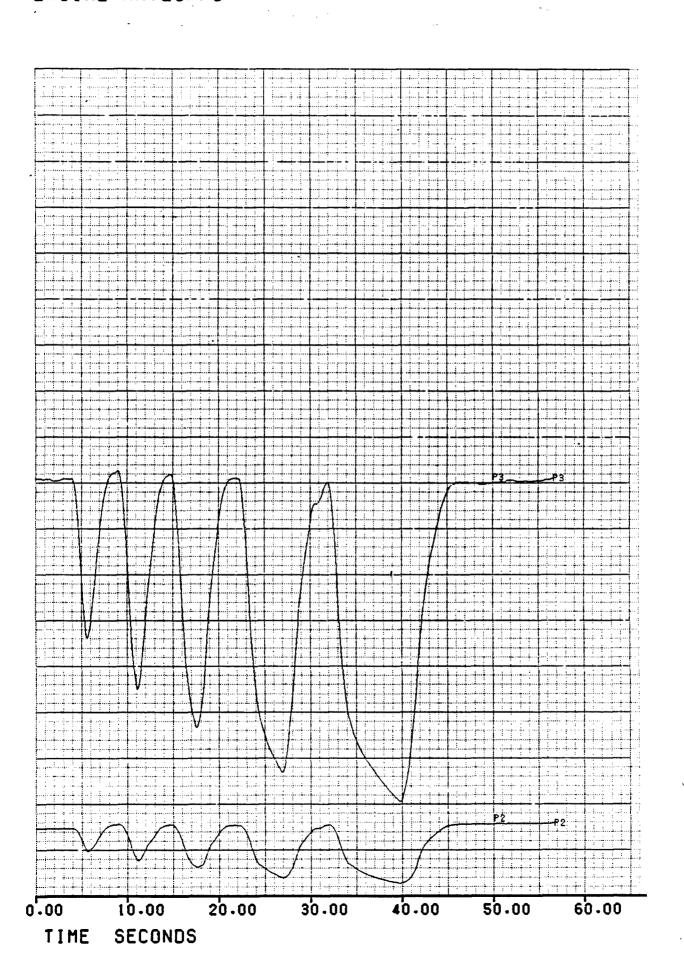
## TACOM FUEL ECONOMY DEMO JAM ACCELERATIOM TO MAXIMUM POWER



#### TACOM FUEL ECONOMY DEMO SNAP DECELERATION TO IDLE



## TACOM FUEL ECONOMY DEMO ENGINE WAVEOFFS



280.00 80.00 TABLES

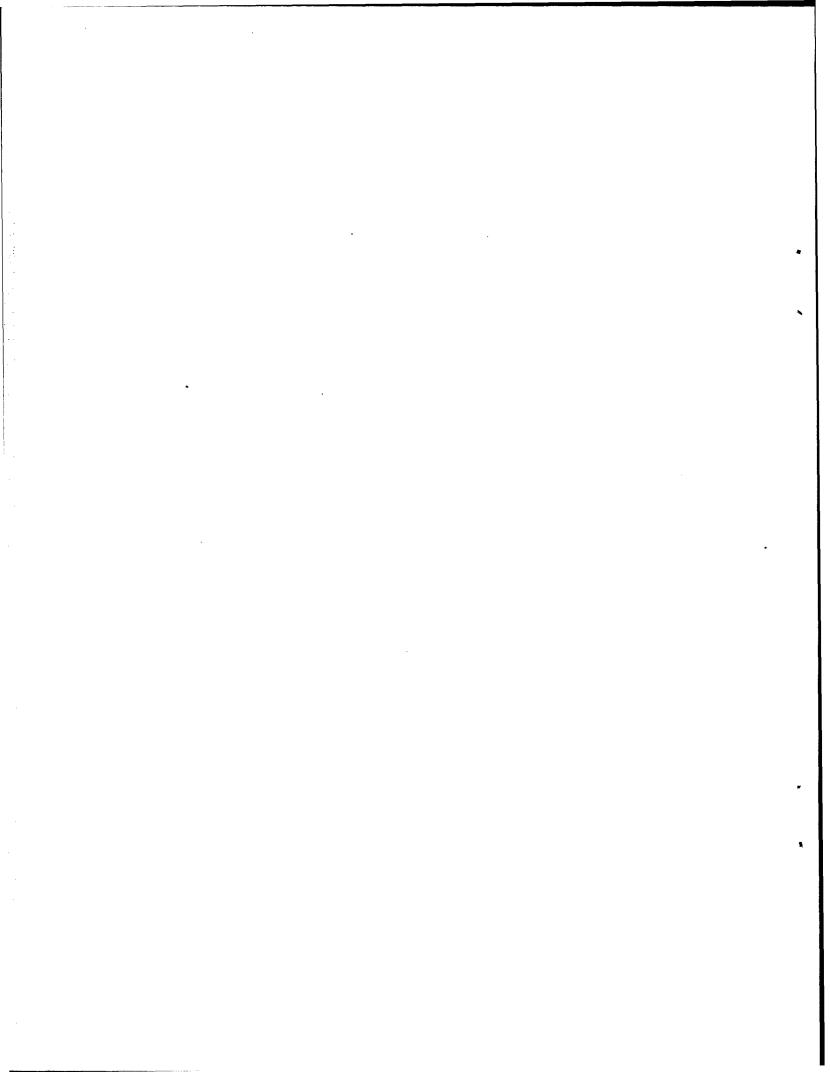


TABLE 1

TMEPS

PEACETIME MISSION/BATTLEFIELD DAY POWER POINTS

OPERATING CONDITION	VEHICLE SPEED (MPH)	ENGINE SPEED (RPM)	ENGINE OUTPUT (HP)
Tactical Idle	-	1300	58.10
Secondary Road	5.0	1174	174.00
Secondary Road	24.9	2389	728.20
Secondary Road	34.8	2379	991.00
Cross Country	16.8	2605	925.00

NOTE: Normal idle points are omitted because they are supplied by the TMEPS APU.

TABLE 2

TME FUEL ECONOMY DEMONSTRATION

PEACETIME FUEL CONSUMPTION, ANNUAL (NBC OFF)

			ENGINE	FUEL CONS (GAL) M1A1	SUMPTION LONS)
OPERATING CONDITION	TIME (HRS)	ENGINE SPEED (NPT)	OUTPUT (HP)	FAB SPEC BASELINE	T202N DEMO
Tactical Idle	50.0	1300	58	740	523
Secondary Road (5 MPH)	12.0	1174	174	271	248
Secondary Road (25 MPH)	14.9	2389	728	840	749
Secondary Road (35 MPH)	9.0	2379	991	672	594
Cross Country	11.3	2605	925	925	690
		· · · · · · · · · · · · · · · · · · ·	TOTAL:	3311	2804
		* R	EDUCTION:		-15.3%

NOTE: Normal idle points are omitted because they are supplied by the TMEPS APU.

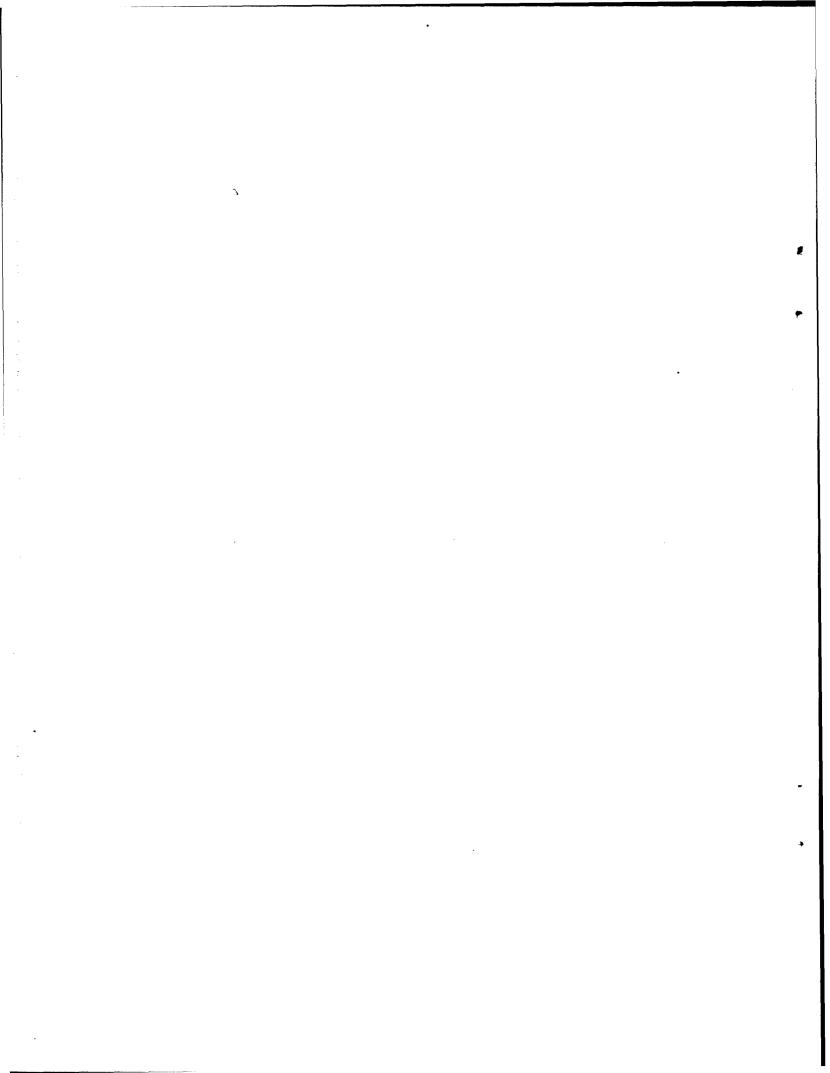
TABLE 3

TACOM TME FUEL ECONOMY DEMONSTRATION

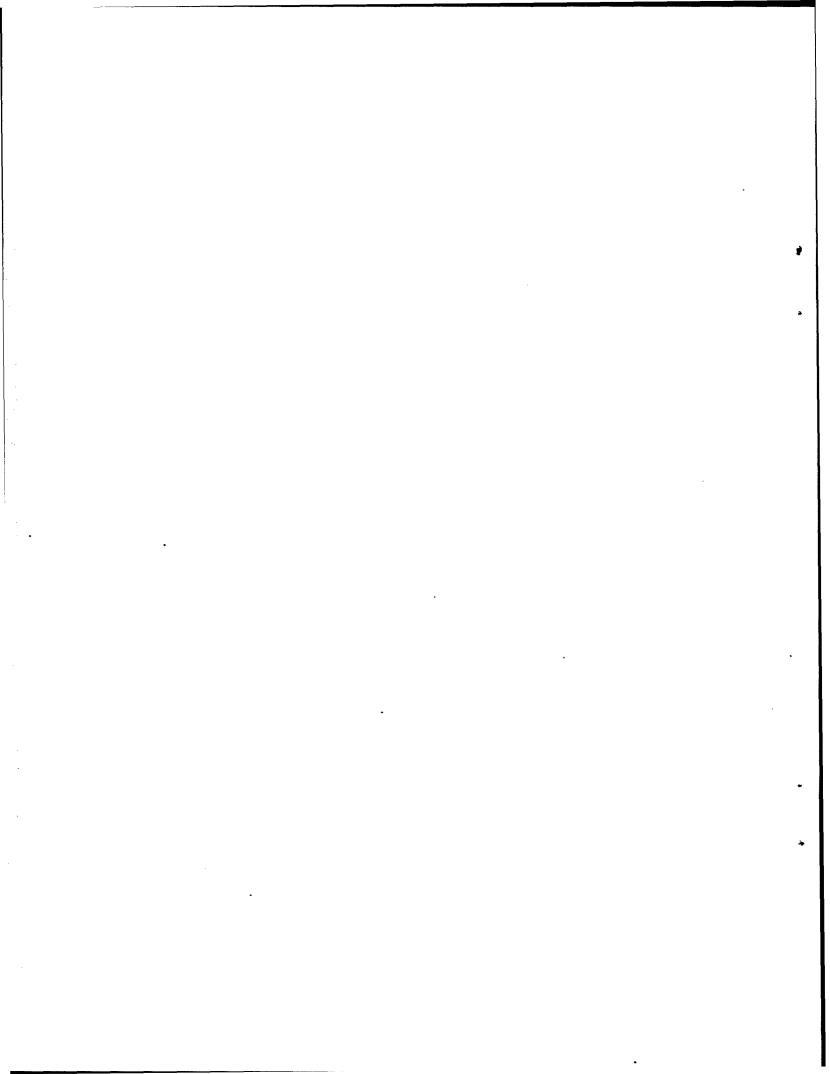
BATTLEFIELD DAY FUEL CONSUMPTION, ANNUAL (NBC OFF)

				FUEL CONSUMPTION (GALLONS)			
OPERATING CONDITION	TIME (HRS)	ENGINE SPEED (NPT-RPM)	ENGINE O OUTPUT (HP)	M1A1 FAB SPEC BASELINE	T202N DEMO		
Tactical Idle	.917	1300	58.1	13.6	9.6		
Secondary Road (25 MPH)	3.4	2389	728.2	191.7	170.9		
Cross Country	3.333	2605	925.0	232.3	203.6		
			TOTAL:	437.6	384.1		
		*	REDUCTION:		-12.2%		

NOTE: Normal idle points are omitted because they are supplied by the TMEPS APU.



APPENDIX



AGT 1500A

TME

FUEL ECONOMY TEST PLAN

LYC 88-28 03-P-902-88

Prepared by

R. Rusu

Sr. Dev. Test Engineer

Concurred by

DA Collings for Estubri

AGT 1500 Performance Manager

Approved by

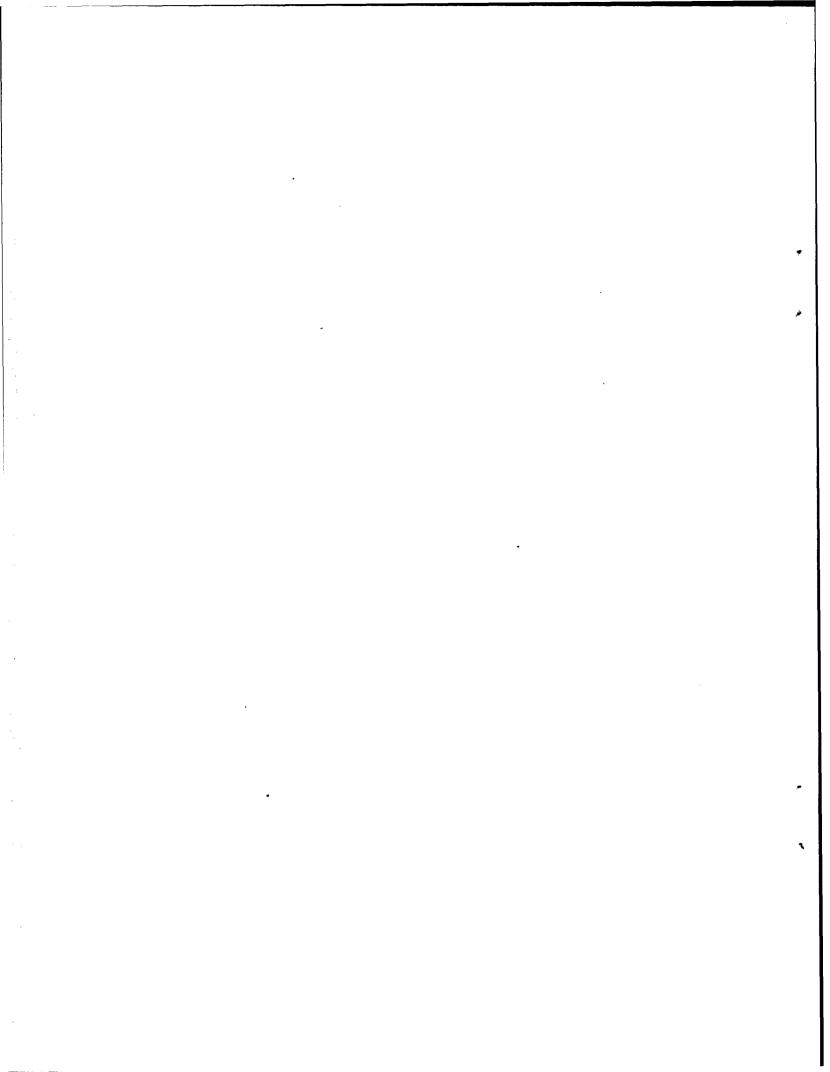
R. Hudson

AGT 1500 Dev. Test Manager

Approved by

R. Horan

Manager, TME Engineering



#### TME FUEL ECONOMY TEST PLAN

AGT 1500A Model Engine, Turbine, Automotive (Uninstalled-ASME Bellmouth Air Induction System and Cell Exhaust Duct)

#### 1.0 SCOPE

This Test Plan shall be the governing document for the execution of the Fuel Economy Test of the Transverse Mounted Engine (TME) demonstrator to be performed on the model AGT 1500A automotive turbine engine. The TME Fuel Economy Test will be conducted in accordance with the specific provisions presented in this test plan while assuring the compliance with TME ETP 1500A.

#### 1.1 Objective

The test objective is to demonstrate the ability of the Transverse Mounted Engine (TME) demonstrator to comply with the fuel economy requirements of TME contract No. DAC100001.

#### 1.2 Applicability

This plan shall apply for the Fuel Economy Demonstration Test of TME, AGT 1500A, S/N T202 (LE89752).

#### 2.0 GENERAL COMPOSITION AND ORDER OF THE TEST

The TME demonstrator Fuel Economy Test shall be accomplished in uninstalled configuration using the standard AGT 1500 ASME bellmouth air induction system and cell exhaust duct at 87±5°F ambient temperature and prevalent barometric pressure.

The TME installation losses will be analytically applied to the uninstalled test cell engine performance level per curves used to generate the TME fuel map (See Attachment 1).

The TME Fuel Economy Test shall prove a minimum 10% weighted fuel savings as compared to the current M1A1 Production Fabrication Specification E2180C based on ATR (63 ton) peacetime annual usage duty cycle (See Attachment 2), installed with NBC off, corrected and adjusted to 87±5°F/500 feet altitude ambient conditions.

During the TME Fuel Economy Demonstration Test the engine performance level shall also be compared to the performance goals stated in Appendix A of TME S.O.W. (See Attachement 3).

The TME fuel economy test results will be presented in an in-house test report. The report will include as a minimum: Summary, Background, Method of Test, Test Results, Conclusions and Recommendations.

#### 3.0 GENERAL PROVISIONS

#### 3.1 Data Accuracy

All instruments and apparatus shall be calibrated before the TME Final Acceptance Test to ensure that reported data shall have a static accuracy within 2% of the values obtained at the maximum rating of the engine, except for fuel flow, which shall be accurate within 1.0% and RPM and torque which shall be accurate within .5% of the values obtained at maximum rating of the engine.

#### 3.2 Data Correction

For purpose of on line data analysis, readings of shaft horsepower, RPM, air flow rate, fuel flow rate, specific fuel consumption, gas pressures, and gas temperatures, will be referred to sea level, standard day, as defined in U.S. Standard Atmosphere, 1962 (ASTIA Document 401813).

#### 3.3 Engine Tests

#### 3.3.1 Test Conditions

The TME Fuel Economy Demonstration Test shall be conducted at 87±5°F ambient temperature and prevailing barometric pressure.

#### 3.3.2 Test Apparatus

#### 3.3.2.1 Test Equipment

The following equipment will be used to facilitate conducting test.

#### 3.3.2.1.1 Power Absorption

A waterbrake will be used to absorb the engine output shaft powers. The brake is supported from the engine by an adapter having four beams which are strain-gaged for torque sensing. The TME accessory gearbox does not have provisions for customer power extraction.

#### 3.3.2.1.2 Starting System

A standard electric starter energized by batteries or a motor generator will provide engine starting power.

#### 3.3.2.2 Data Acquisition Equipment

The following apparatus shall be used to measure and record the required data. Where such equipment is available, an automatic data acquisition system may supplement or supplant the indicating devices noted below.

#### 3.3.2.2.1 Output Shaft Torque

The support beams of the waterbrake mounting adapter are equipped with calibrated strain gages which sense the output torque. Conversion of the strain gage signal to torque indication will be accomplished by a suitable signal convertor.

#### 3.3.2.2.2 Rotor Speeds

Magnetic pulse generators with suitable signal conditioning and readout equipment will measure and display engine main rotor speeds.

#### 3.3.2.2.3 Airflow

A calibrated inlet nozzle, with ASME recommended geometry will utilize throat static pressure and entry total pressures to measure airflow during performance calibrations.

#### 3.3.2.2.4 Pressures

Calibrated Bourdon tube gages and/or transducers will measure hydraulic and aerodynamics pressures.

#### 3.3.2.2.5 Temperatures

Temperatures will be measured by I.C. and/or C.A. thermocouples. Indication will be by means of Self-balancing Precision Brown Potentiometers or by digital indicators in conjunction with appropriate signal condition equipment.

#### 3.3.2.2.6 Fuel Flow and Oil Flow

Calibrated turbine elements with associated signal converters, amplifiers, and readouts will be used for fuel and oil flow measurements.

#### 3.3.2.2.7 Engine Control Equipment

The engine power lever and waterbrake load will be manually controlled by the test operator from the control room. The engine normal operation will be monitored by hardware alarms in case any critical parameter exceeds the pre-established limits.

#### 3.3.2.2.8 Transient Recorders

Transient recording apparatus will be used during the TME Fuel Economy Test to provide continuous monitoring of the engine. The following variables vs. time will be continuously recorded during the test: low pressure spool speed (NL), high pressure spool speed (NH), power turbine speed (NPT), high pressure spool outlet pressure (PT3), fuel flow (WF), measured power turbine inlet temperature (TT7), and throttle position (PLA).

#### 3.3.2.2.9 Vibration Measurement

The vibration measuring equipment will consist of Trig-Tek vibration meters Model No. 203J in conjunction with velocity-displacement pick-ups Model No. 4-128MA or equivalent. The meter will have incorporated in its system appropriate high pass filters. Displacement values (peak to Peak amplitude in mils) will be recorded by a minimum of two vibration pick-ups mounted on the engine.

#### 3.3.3 Operating Conditions

#### 3.3.3.1 Oil Pressure

The operating oil pressure is measured at the engine oil pump manifold (POMA). Operating condition shall be as described in E2147B with the exception of low idle pressure which must be readjusted according to the corresponding lower NH speed at TME low idle.

#### 3.3.3.2 Oil Servicing

The oil system will be drained and filled with new, MIL-L-23699 oil, Mobil 254 (containing anticoking additives), at the start of the test.

The oil filters shall be inspected and the filtration elements replaced and the oil changed if required. All oil additions and oil changes shall be noted and the chart of oil consumption prepared. Oil samples, chemical and spectro shall be taken prior to the beginning of the TME Fuel Economy Test and also after the end of the test. All samples will be subjected to laboratory analysis.

#### 3.3.3.3 Accessory Drives

The fuel control, the oil pump, and the electric starter will be the engine accessories installed and run during the TME Final Acceptance Test. The newly designed TME Accessory Gearbox does not have provisions for the electrical generator or the hydraulic pump, therefore, no accessory load will be extracted from the TME.

#### 3.3.3.4 Barometer Readings

The barometric pressure shall be read, corrected for the actual ambient temperature, and recorded at intervals not exceeding three hours.

#### 3.3.3.5 Measured Gas Temperatures

Gas temperatures shall be determined by a thermocouple harness located at the inlet of the power turbine.

#### 3.3.3.6 Fuel

The TME Fuel Economy Demonstration Test will be run on DF-2 diesel fuel conforming to Federal Specification VV-F-800. Fuel samples will be taken prior and after the Final Acceptance Test. All samples will be subjected to laboratory analyses.

#### 3.3.3.7 TME Fuel Economy Test Repeat

In case an engine component problem should occur during the TME Fuel Economy Test which requires engine disassembly, the test will be interrupted for appropriate corrective actions. The Fuel Economy Test will be reinitiated from the beginning and completed.

#### 3.3.4 Method of Test

#### 3.3.4.1 Procedure

The TME Fuel Economy Test shall be conducted in accordance with the specific provisions stipulated in this test plan assuming the compliance with TME ETP 1500A. Sufficient data shall be obtained to establish the 500 ft,  $87\pm5^{\circ}F$  day, installed performance requirements required by the TME S.O.W. As a minimum, the data described in Paragraph 3.3.4.2 shall be acquired at the power levels described in Table I. The power levels shall be maintained for a sufficient period (not less than 3 minutes duration) to allow the engine stable functioning (stabilized engine parameters). The compressors may be cleaned with B & B 3100 solution and water.

The TME transient acceptability shall be evaluated versus the TME S.O.W., Appendix A, Paragraph D (See Attachment 3).

The TME Fuel Economy Test will be conducted in the uninstalled configuration with an ASME bellmouth mounted at the engine inlet. The fuel to be used is DF-2 which conforms to Federal Specification VV-F-800.

Throughout the test, power levels will be established using the power lever which establishes the amount of fuel delivered by the fuel control to the engine and the corresponding power turbine load to be applied through the power absorption system (waterbrake) setting the required power turbine speed.

The TME Fuel Economy Test should confirm that the PTS is properly optimized and based on maximum allowable high pressure turbine inlet temperature, establish the engine maximum power capability at 87±5°F, sea level, uninstalled.

#### 3.3.4.2 Data

During the TME Fuel Economy Test the following data shall be recorded on each long reading (full scan):

- Time of Day
- L.P. Spool Speed, % RPM
- H.P. Spool Speed, % RPM
- Power Turbine Speed, % RPM
- Variable Inlet Guide Vane Position, Volts
- Shaft Horsepower, SHP (calculated based on measured output shaft torque)
- Torquemeter Reading, LBFt (measured output shaft torque)
- Fuel Consumption Rate (LB/HR)
- Engine Inlet Air Temperature, °F
- Bellmouth Static Air Pressure, inches H<sub>2</sub>O
- Bellmouth Total Air Pressure, inches H<sub>2</sub>O
- Barcmetric Pressure, PSIA

- LP Compressor Total Discharge Temperature, °F
- LP Compressor Total Discharge Pressure, PSIA
- HP Compressor Total Discharge Temperature, oF
- HP Compressor Total Discharge Pressure, PSIA
- Recuperator Total Discharge Temperature, °F
- Oil Inlet Temperature, °F
- Scavenge Oil Discharge Temperature, oF
- Fuel Pressure at Fuel System Inlet, °F
- Measured Gas Temperature, TT7, °F
- Fuel Temperature, °F
- Throttle Position, Volts
- Vibration Displacements, MILS

#### 3.4 Criteria

#### 3.4.1 Performance

It is required that TME AGT 1500A, S/N T202 (LE89752) will demonstrate as a minimum 10% mission fuel savings when compared to M1A1 annual usage duty cycle at 87±5°F and 500 feet altitude installed conditions (See Attachment 2).

It is required that the engine pass the ETP 1500A requirements during the TME Fuel Economy Test.

#### 3.4.2 Mechanical Integrity

In the event an engine component fails, affecting the engine performance and/or the engine mechanical integrity, the test will be interrupted for immediate corrective action and then the TME Fuel Economy Test must be reinitiated again.

#### 3.5 Test Monitoring

#### 3.5.1

The test controller (General Dynamics Land Systems Division) shall be involved in the decision to stop, fix, and rerun the TME Fuel Economy Test due to the incidents described in 3.4.2.

#### 3.5.2

The test controller shall be informed of any action taken to correct hardware deficiencies.

#### 3.5.3

All incidents shall be reported to the test controller as they occur during the testing. Reporting will be accomplished by telephone with a written report to follow. A close-out report will be provided when the problem is resolved.

#### TABLE I

#### TME FUEL ECONOMY DEMONSTRATION TEST

The following tabulation presents the TME test conditions to be set, where the engine performance should be collected and evaluated.

NOTE: Tempered air of 87±5°F shall be provided.

#### I. The Peacetime Power Points

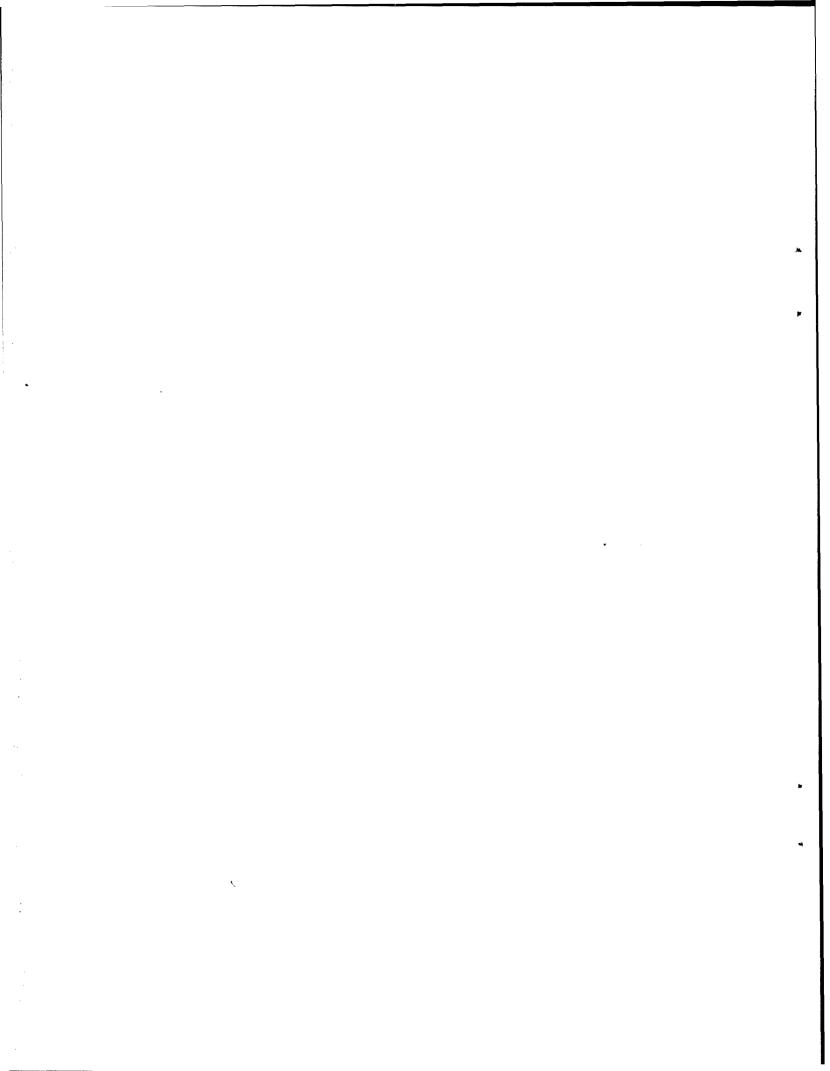
Operating Condition	NPT	SHP
Cross Country	86.83	989
35 MPH	79.30	1063
25 MPH	79.63	771
5 MPH	39.13	181
Tac Idle	43.30	60.9

#### II. The optimum PTS O/A Calibration

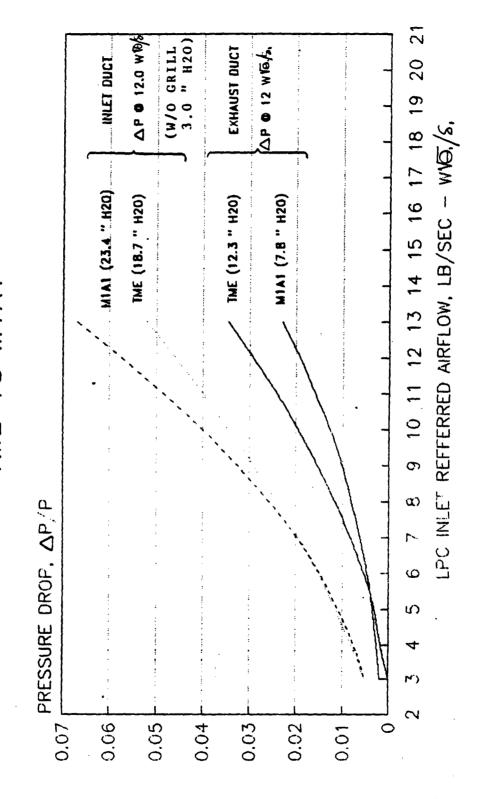
Acquire 5 data scans at that open area PTS which provides 1505 SHP/ $\delta\sqrt{\theta}$  (1545 referred to 87°F) at minimum T5/ $\theta$ .

### III. The TME TT7 - PTS Tracking Calibration (At 87°±5°F ambient temperature)

Acquire data scans at normal idle, tac idle, and subsequently at 100 SHP increments to max of 1545 SHP/ $\delta$  at optimum NPT speeds. Normal idle = 40 SHP, NPT = 900 RPM, Tac Idle = 55 SHP, NPT = 1300 RPM.



# INSTALLATION LOSSES TME VS M1A1



TME FUEL ECONOMY TEST PLAN LYC-88-28

ATTACHMENT 1

## PEACETIME ANNUAL FUEL USAGE NBC OFF

					1		
(03)	SPEED CUTPUT FUEL USED (RPM) SHP (GALLONS)	499.0	253.2	745.0	593.1	684.8	2775.1
IMEPS (7-SPEED)	SPEED OUTPUT FUEL (RPM) SHP	58.1	174.0	728.2	991.0	925.0	
TWEP	ENGINE SPEED (RPM)	1300	1174	2369	2379	2605	
ED)	SPEED OUTPUT FUEL USED (RPM) SHP (GALLONS)	740.0	271.1	840.3	671.9	787.6	3310.9
M1A1 (4-SPEED)	ENGINE OUTPUT SHP	55	172	675	1018	859	
MIA	ENGINE SPEED (RPM)	1300	1095	1855	2593	1853	
	VEHICLE SPEED (MPH)		5.0	24.9	34.8	16.8	
	TIME SPEED (HOURS) (MPH)	50.0	12.0	14.9	9.0	11.3	
	OPERATING CONDITION	TAC IDLE	SECONDARY ROAD	SECONDARY ROAD	SECONDARY ROAD	CROSS	TOTAL

GVW = 63 TONS
87 F, 500 FT., INSTALLED
5KW ELECTRICAL POWER USAGE
FUEL DENSITY = 7.05 LBM/GALLON
FUEL LHV = 18,500 BTU/LBM

ATTACHMENT 2 TME FUEL ECONOMY TEST PLAN LYC 88-28

#### Attachment 3

## TME FUEL ECONOMY TEST PLAN LYC 88-28

#### Engine Performance Goals

Unless otherwise specified the performance goals stated in the following subparagraphs not include installation losses. The inlet is configured with as ASME bellmouth. Fuel is DF-2 which meets Military Specification VV-F-800 (reference lower heating value 18,500 BTU/pound).

In general the type of engine performance characteristics which follow are typical of those quoted MI/MIA1 engine configuration.

A. The engine shall develop the minimum corrected brake horsepower (BHP) at the applicable shaft speed specified below when operated at an ambient temperature of 87°F with barometric pressure corrected to 29.92 in. Hg.

Nominal Speed (RPM)	ВНР
1800 ± 50	1295
2000 ± 50	1371
2600 ± 50	1500
3000 ± 50	1500

B. The engine specific fuel consumption shall meet or be less than the values provided below. Note: Ambient temperature 87°F at 29.92 in Hg. Actual SFC calibration data shall be provided at the points listed below.

Gross Brake HP	RPM		SFC (LBM/HP-HR)
nr	KFII	=	SEC (LDM/HE-HK)
600	2748 ± 60		.50
900	$3000 \pm 60$		. 45
1200	3000 ± 60		. 44
1500	$3000 \pm 60$		. 45
Idle			•
(neutral) (40 Hp)	900 ± 25		1.48
Tc Idle (55 Hp)	$1300 \pm 25$	•	1.28

C. Maximum and minimum governed speed.

0	Drive and reverse range	0	Low Idle					
	1300 SHP 3075 + 75 RPM -5 RPM		100	SHP	900	±	50	RPM
	1200 SHP 3075 ± 75 RPM -5 RPM		0	SHP	930	±	50	RPM

Reference Appendix A of TME S.O.W.

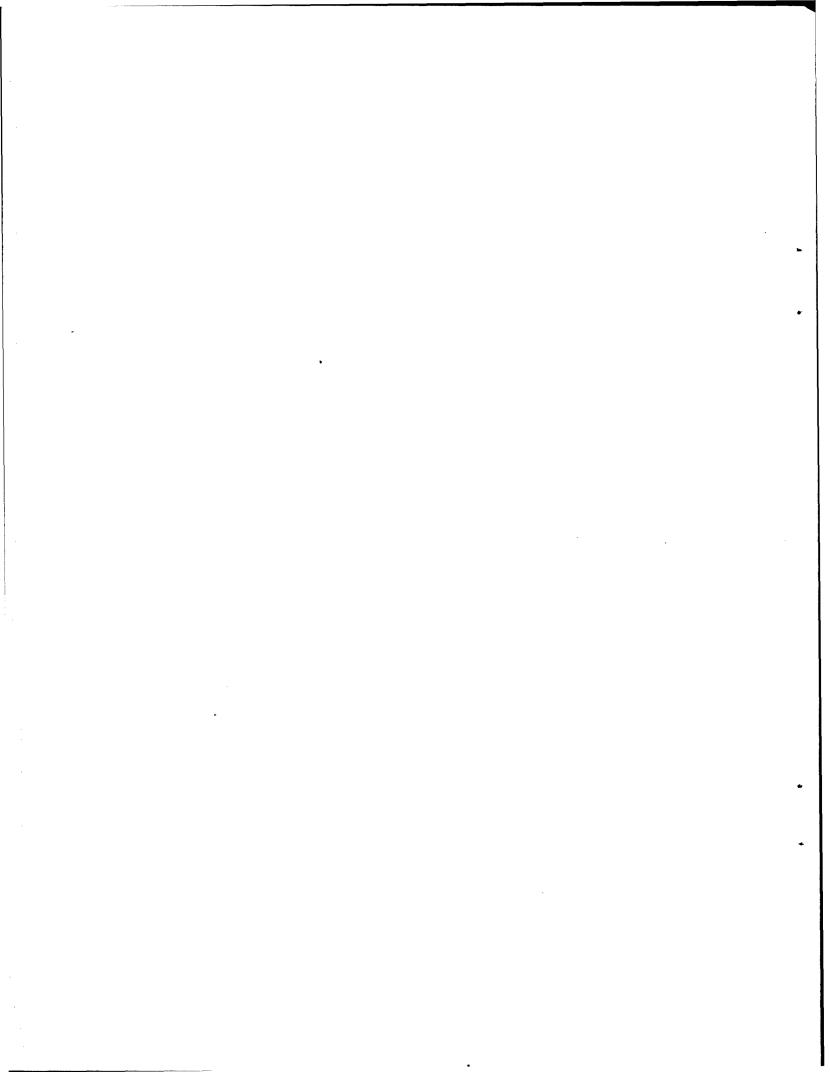
#### ATTACHMENT 3

## TME Fuel Economy Test PLan LYC-88-28

#### Engine Performance Goals (con't)

- D. <u>Transient Response</u>. Verification of conformance to transient response requirement shall be accomplished as follows:
- a. Set the power condition from 300, (+25, -0), power turbine speed (Np) at 63  $\pm$  2%. Stabilize for 3 minutes. Note high pressure compressor discharge pressure (PT3) at 300 shp.
- b. Accelerate to 90 percent of compensated gross horsepower obtained from Figure 4 with 20 ± 5 hp extracted at the power takeoff pad. Note PT3 at this horsepower.
- Set compensated gross horsepower per Figure 4, (+25, -0), with 20 ± 5 hp extracted at the power takeoff pad. Power turbine speed shall be 100 ± 2%. Run for 3 minutes.
- d. Snap decelerate to minimum power lever angle, leaving output shaft power absorption device preset. Time deceleration to PT3 determined in "a". Deceleration time shall not exceed 5.0 seconds. Stabilize at idle for 3 minutes.
- e. Jam throttle to maximum power. Time acceleration to PT3 determined in "b". Response time not to exceed 4 seconds. Stabilize for 3 minutes.
- f. Wave off engine in 5% high pressure compressor speed (Nh) increments until 75 percent Nh is reached. Waveoffs shall be surge-free.

APPENDIX III



PO. NO. E225693

FINAL REPORT FOR THE DEVELOPMENT OF A ROTATING ELEMENT SELF-CLEANING AIR FILTER (RESCAF)

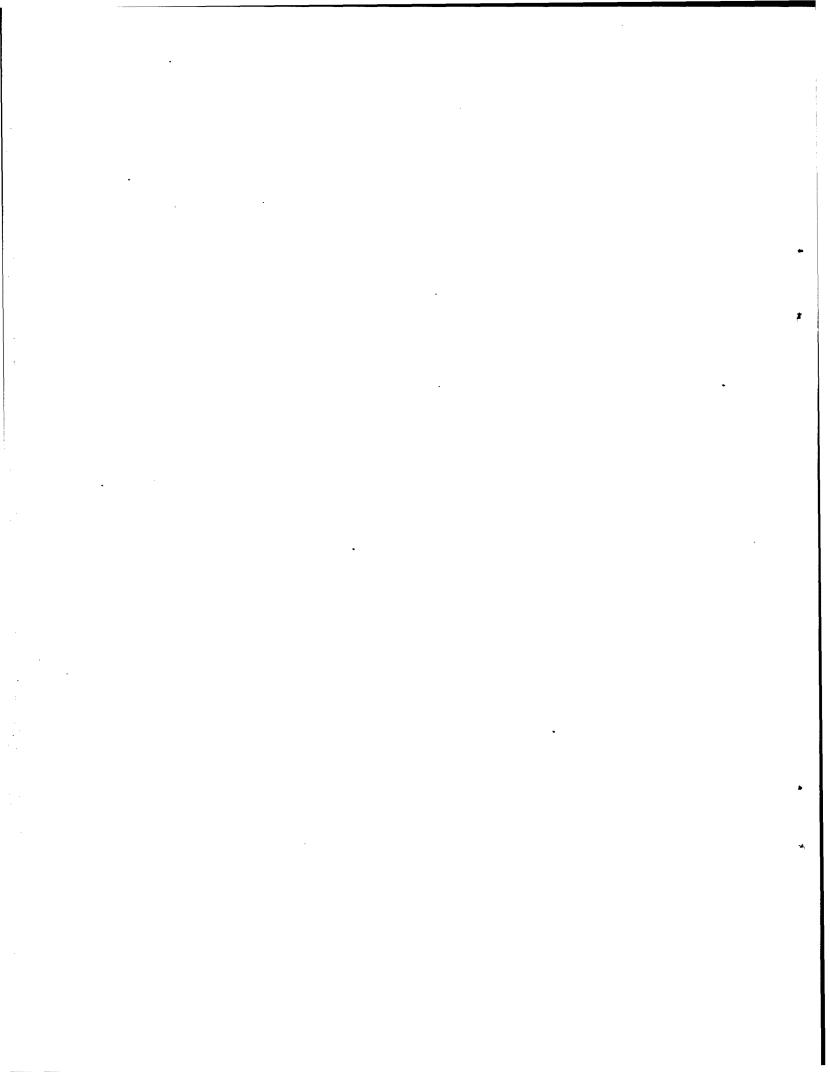
**JULY 1989** 

PREPARED FOR:

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MINNEAPOLIS, MN 55440



#### donaldson



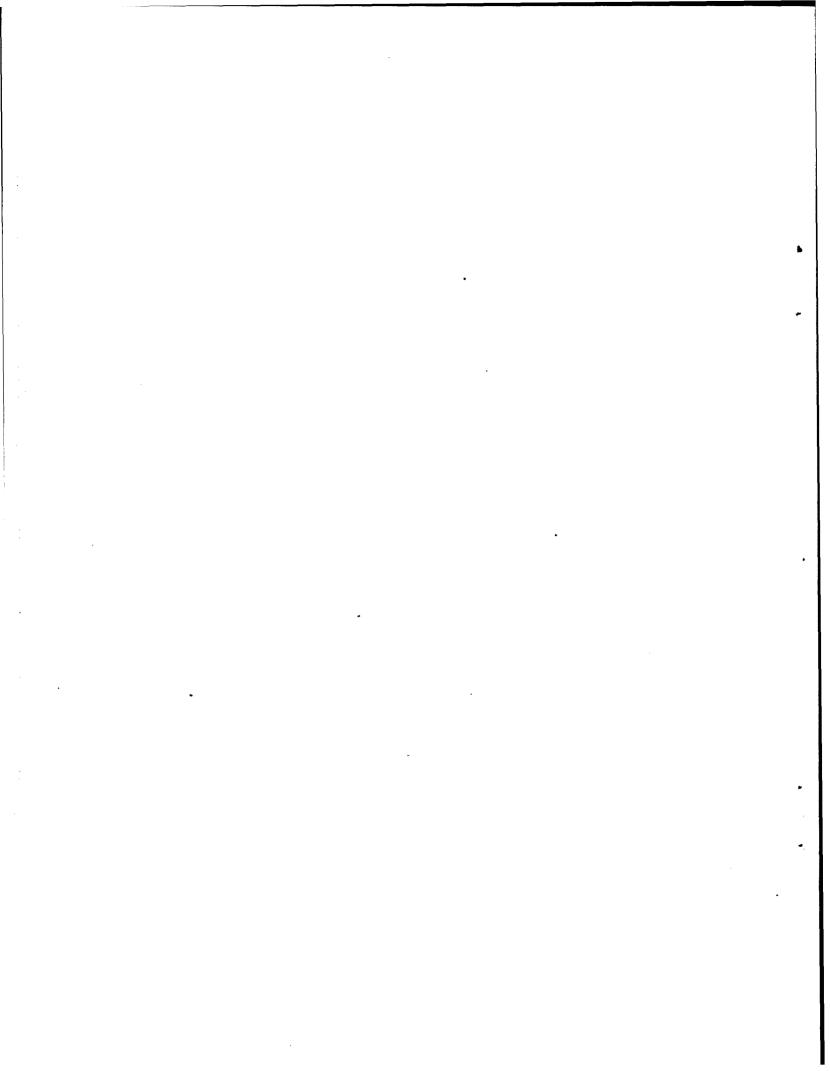
Donaldson Company, Inc. Defense Products 1400 West 94th Street Minneapolis, Minnesota

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Dennis J. Grigal Sales Manager

Defense and Aerospace Products



#### LIMITATIONS

It should be understood that Donaldson Company's reservations regarding these RESCAF systems is based upon the fact that an adequate level of component maturity testing has not been accomplished on these units and long term field tests will quite likely result in component failures. Additional component maturity testing is planned for FSED RESCAF systems. Problems occurred on two occasions with the RESCAF drive motors and a fix to one of the problems remains to be implemented (a larger straight keyway machined on the drive motor output shafts.) Problems also occurred with the RESCAF to engine mounting strut pins during shock testing which are not yet addressed at the time of this publication.

In addition, there are critical components, not developed by Donaldson Company, Inc. under this contract, that also must perform properly or performance will be substandard, these include:

#### Air Cleaner Scavange Blower & Ductwork

Inadequate scavange of the RESCAF, which can result from component problems (e.g., scavange blower) or improper sizing of ductwork, will severely reduce the RESCAF system dust capacity because a common duct is used for both cleaning mechanism and precleaner scavange.

#### RESCAF Air Compressor and Pressure Regulation Equipment

If compressed air is supplied to the RESCAF at inadequate (low) pressure, a reduction in cleaning performance will occur which reduces system dust capacity. If compressed air is supplied to the RESCAF systems at too high a pressure, the filter medium may rupture and the engine will be endangered.

A "safe" performance pressure band exists which must be maintained.

#### CONCLUSIONS

Under this contract a Rotating Element Self-Cleaning Air Filter (RESCAF) was designed, fabricated, and successfully laboratory tested. The RESCAF system met performance objectives by demonstrating in laboratory tests resistance to shock, vibration, and dust life under zero visibility conditions in excess of 200 hours.

Pressure drop characteristics of the RESCAF system met system requirements and are shown on figure 1.

An overall system dust collection efficiency of 99.999 was demonstrated during the life test of the RESCAF. Efficiency of the filter element at all times exceeded 99.9%. These efficiency levels exceed all Government and Textron Lycoming requirements.

The RESCAF system was subjected to shock and vibration testing and survived in operating condition. Dust testing after shock and vibration tests indicated that a satisfactory level of performance was maintained. Results of these tests are discussed in sections 5.4 and 5.5.

#### RECOMMENDATIONS

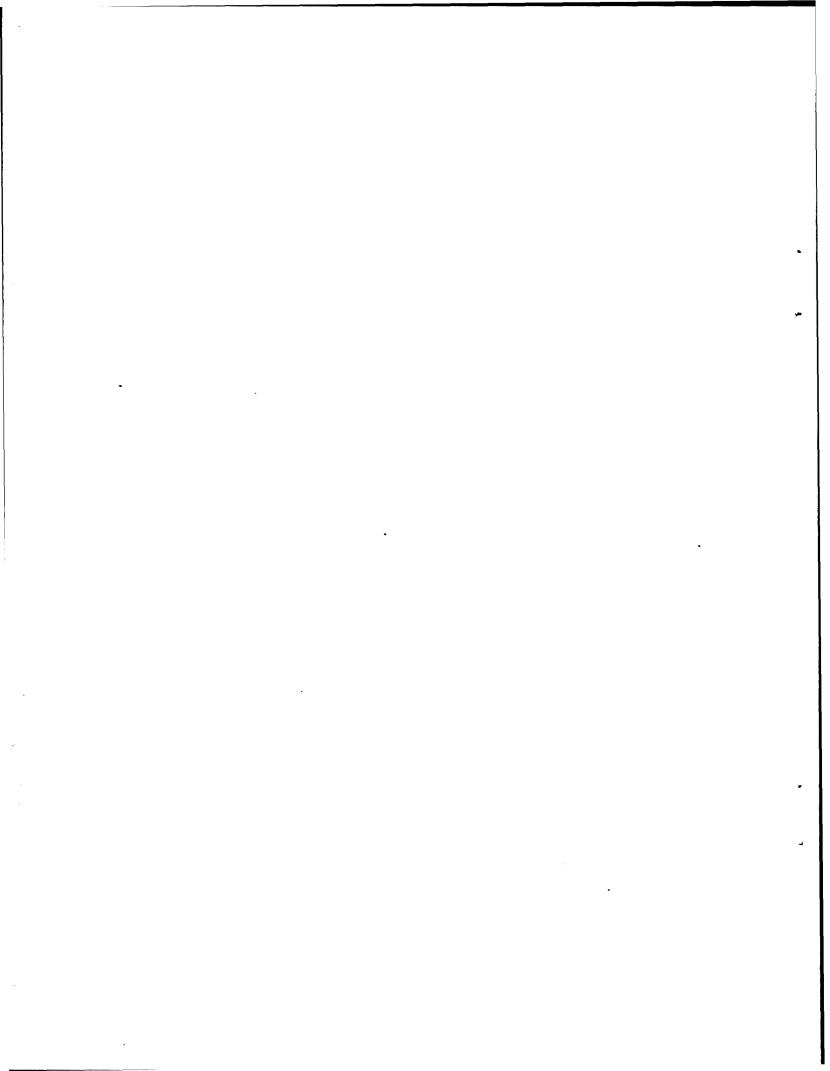
The RESCAF system demonstrated satisfactory performance during laboratory tests, and may be installed on an engine for limited field evaluation.

Additional recommendations for next generation systems are discussed in Section 6.4. Limitations discussed at the beginning of this report apply.

# DISCUSSION

# HARDWARE

The RESCAF system operation, mechanical description, and laboratory test summary is presented in this section. The RESCAF assembly is shown on figure 2 and with major removable components disassembled on figure 3.



#### SUMMARY

This report covers the development of a Rotating Element Self-Cleaning Air Filter (RESCAF) for a transverse mounted Textron Lycoming AGT 1500 gas turbine engine (TME). The results of laboratory testing is included. Shock, vibration, and simultaneous vibration and dust testing were successfully completed. Additional maturity testing will be required on FSED RESCAF systems, however, the RESCAF systems shipped under this contract are ready for installation in a main battle tank for limited field test evaluation.

The RESCAF systems supplied under this contract to Textron Lycoming are subject to the limitations discussed in the next section.

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#### 1.0 INTRODUCTION

This final technical report, prepared by Donaldson Company, covers work performed under contract with Textron Lycoming, Stratford, Connecticut, for development of a Rotating Element Self-Cleaning Air Filter (RESCAF). Work conducted on several purchase orders is covered in this report.

#### 2.0 OBJECTIVES

The overall objective of the program, summarized in this report, was to design and laboratory test a prototype RESCAF system for the Textron Lycoming transverse mounted AGT 1500 engine (TME). The laboratory test results are used as an indicator of the readiness of the hardware for installation on an engine for vehicle tests. The design objective was to build a RESCAF system which would fit the available space envelope. This envelope changed and became more defined as work progressed. The performance objective was to exceed the performance level demonstrated by the M1 Abrams tank SCAF and standard two-stage air cleaner. To meet this objective the RESCAF had to exceed the following environmental resistance characteristics.

- a. Operating Temperature Exposure to ambient air temperatures ranging from -65°F to +200°F for 8 hours minimum.
- b. Humidity Ambient relative humidity to a maximum of 100 percent, with air temperatures to a maximum of 95°F for 48 hours minimum.
- c. Chemicals Exposure to the vapors of, and contact with, the following materials for durations up to 48 hours:
  - Fuel Per VV-F-800, Mil-T-5624, MIL-G-3056, and
     MIL-F-16884
  - Hydraulic fluid Per MIL-H-46170

- Cleaning agents Per P-C-437
- Lubrication oils Per MIL-L-2104, MIL-L-23699, and MIL-L-7808
- d. Basic Shock Three 40  $\pm$  4.0 accelerations due to gravity (g), 18  $\pm$  0.02 milliseconds (ms) half sine wave shocks applied in each direction along the three mutually perpendicular axes, for a total of 18 shocks.
- e. Gun Firing Shock Three 55 + 5.5 g,  $2.5 \pm 0.02$  ms half sine wave shocks applied in each direction along the three mutually perpendicular axes, for a total of 18 shocks.
- f. Operational Shock Three 55  $\pm$  5.5 g, 0.5  $\pm$  0.1 ms half sine wave shocks applied in each direction along the three mutually perpendicular axes, for a total of 18 shocks.
- g. <u>Nondestructive Ballistic Shock</u> Shock impulses as specified in table 1 at the assembly mounting interface. Three shock impulses in each direction of the specified axis (six shocks per axis) shall be imposed.
- h. <u>Vibration</u> Sinusoidal vibrations in each of the three mutually perpendicular axes at the frequencies and accelerations specified in the following table. Vibration frequencies shall be applied at a logarithmic sweep rate of 20 minutes per sweep cycle from 5 Hz to 500 Hz to 5 Hz. Total vibration time, shall be 120 minutes in each axis. Unless otherwise specified, the conditions of specification MIL-STD-810D, Method 514.1 shall prevail (see table 2).

Table 1. Nondestructive Ballistic Shock Conditions

Axis	Level (g)	Duration (ms)		
Latitudinal	200 <u>+</u> 20	0.5 ± 0.1		
Vertical	200 <u>+</u> 20	0.5 ± 0.1		
Longitudianl	550 ± 55	$0.5 \pm 0.1$		

Table 2. Vibration Levels

Axis	Frequency (Hz)	Level (g)
Vertical	5 to 25	1
	25 to 57	0.030 inch DA
	57 to 500	5
Laterial and	5 to 25	1
Longitudinal	25 to 44	0.020 inch DA
	44 to 500	3

# i. Flame Resistance - Filter media

- Filter media materials must be self-extinguishing after burning
   inch or less during upward flame propagation, with no spark, sputter, or drip of flaming particles.
- 2. Downward flame propagation rates of filter media shall not exceed 0.3 inch per second with no spark, sputter, drop or transfer of solid mass during burning.
- 3. Filtering materials shall offer no evolution of flashing vapors below 100°F or evidence charring, self-sustained combustion, or pyrolysis at less than 150°F.
- j. Storage Temperature Ambient storage temperatures within the range of -70°F to +300°F for extended durations.
- K. <u>Fungus</u> Inoculation of external and internal surfaces and components of the assembly with spore suspension as defined by the Specimen Inoculation paragraph of MIL-F-13927 followed by exposure to ambient air temperatures between 80°F and 84°F at relative humidity between 96 to 100 percent for a 28-day duration.

#### 3.0 CONCLUSIONS

Under this contract a Rotating Element Self-Cleaning Air Filter (RESCAF) was designed, fabricated, and successfully laboratory tested. The RESCAF system met performance objectives by demonstrating in laboratory tests resistance to shock, vibration, and dust life under zero visibility conditions in excess of 200 hours.

Pressure drop characteristics of the RESCAF system met system requirements and are shown on figure 1.

An overall system dust collection efficiency of 99.999 was demonstrated during the life test of the RESCAF. Efficiency of the filter element at all times exceeded 99.9%. These efficiency levels exceed all Government and Textron Lycoming requirements.

The RESCAF system was subjected to shock and vibration testing and survived in operating conditions. Dust testing after shock and vibration tests indicated that a satisfactory level of performance was maintained. Results of these tests are discussed in sections 5.4 and 5.5.

#### 4.0 RECOMMENDATIONS

The RESCAF system demonstrated satisfactory performance during laboratory tests, and may be installed on an engine for limited field evaluation.

Additional recommendations for next generation systems are discussed in Section 6.4. Limitations discussed at the beginning of this report apply.

## 5.0 DISCUSSION

#### 5.1 HARDWARE

The RESCAF system operation, mechanical description, and laboratory test summary is presented in this section. The RESCAF assembly is shown on figure 2 and with major removable components disassembled on figure 3.

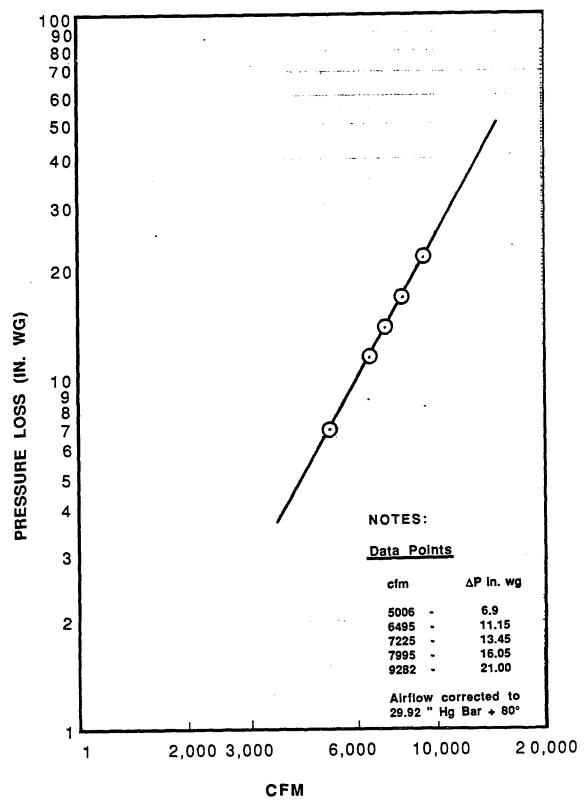


Figure 1. RESCAF Pressure Drop Characteristics

Figure 2. RESCAF Assembly

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## 5.1.1 Description of Operation

The RESCAF system, designed for use on a main battle tank, uses a two-stage process to provide a clean air supply to the engine. The first stage of the system is a precleaner which uses an inertial separation mechanism to reduce the quantity of dust reaching the second stage. The second stage contains four cylindrical segment barrier filters to collect the remaining dust and are automatic cleaning mechanism for filter element rejuvenation. Figure 4 is a schematic of this nozzle receiver type of self-cleaning air filter. The following paragraphs describe the RESCAF operational sequence.

- a. Dust-laden air enters the system after passing through the ballistic grills located on the left sponson. A stainless steel inlet screen, mounted on the inlet of the ballistic grill intercepts leaves, stones, and large debris.
- b. The dust-laden air then enters the precleaner where it is drawn through STRATA® tubes. Here the heavier dust particles are separated from the airstream by inertial forces. The separated dust is conveyed by a scavenge airstream and discharged overboard. This first stage of the system has an efficiency of 92 to 94 percent on AC Coarse test dust.
- c. The air and remaining dust next enters the four cylindrical segment barrier filters. These filters collect a high percentage of the remaining dust through the traditional filtration mechanisms. After leaving the filters (99.9 percent efficient on AC Coarse test dust), the clean air passes directly into the engine inlet.
- d. As dust continues to collect on the barrier filters, airflow to the engine becomes restricted. When airflow restriction reaches a preset level, a pressure sensor activates the automatic cleaning process. The filter elements are rotated by a small servo motor past a cleaning nozzle which back flushes thru the filter elements compressed air from a small auxillary compressor.

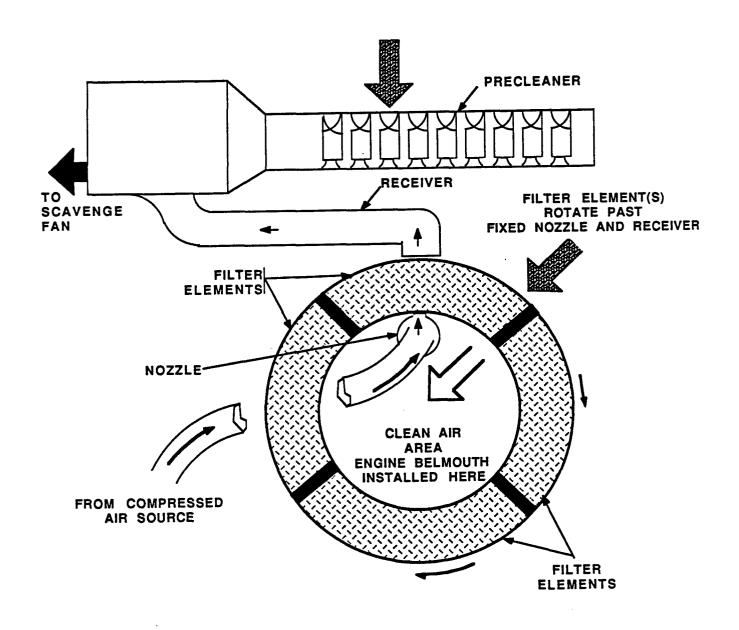


Figure 4. RESCAF Assembly Schematic

The dislodged dust is vacuumed by the receiver and conveyed into the precleaner scavenge system. The self-cleaning process stops after one complete cleaning cycle, lasting about 30 seconds. This process is reactivated only when airflow restriction again reaches the preset level or if manual override actuation is desired. Airflow to the engine remains uninterrupted during the automatic cleaning process.

#### 5.1.2 Description of Hardware

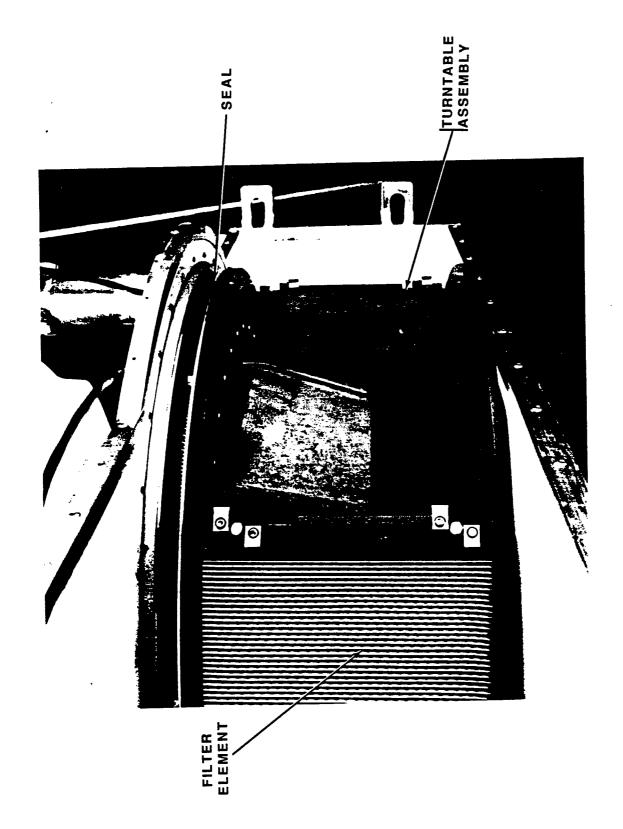
The physical size and shape of the RESCAF was influenced by the mechanical layout of the propulsion system components, the availability of access for service, and the application of appropriate RESCAF components. The resulting configuration, shown on figure 5, provides top access for filter service. The paragraphs on the following pages describe the major assemblies that make up the 9800P0200 RESCAF.

#### 5.1.2.1 Precleaner Assembly

The RESCAF precleaner assembly was designed to provide the following functions:

- a. Effectively remove a high percentage of dust entering the engine air cleaner inlet, thus reducing the quantity of dust which must be collected on the second stage barrier filters. A separation efficiency of 93 percent on AC Coarse test dust was the performance objective.
- b. Support internal ductwork for conveying dust cleaned from the second stage filter elements.

The precleaner functional requirements combined with environmental requirements provided the basis on which component selection, materials and finishes were made.



Final RESCAF Configuration Showing Top Access For Filter Removal Figure 5.

The RESCAF precleaner contains inertial separator tubes pressed into aluminum alloy baffles. The inertial separator tubes are constructed as a tube assembly and are sealed into a welded frame constructed of aluminum alloy fabricated parts. The framework houses a gasket assembly providing the seal between the RESCAF body assembly and the precleaner. A Marman type connection is provided for attachment of scavenge ducting. Four captive bolt fasteners are provided to secure the precleaner to the body assembly.

## 5.1.2.2 RESCAF Body Assembly

The RESCAF body assembly is a 5052 alloy aluminum weldment, and a 356 alloy transition casting to which is attached the RESCAF filter rotating drum and cleaning nozzles. This assembly is shown on figure 6. The four filter elements are secured to the rotating drum with four retention bolts per filter.

The cleaning mechanism consists of two fixed cleaning nozzles located on the inner circumference of the drum and two fixed receivers flanking the nozzles on the outer circumference of the drum. The filters pass between the nozzles and receivers when the drum rotates.

The drum is bolted to a bearing which in turn is bolted to the body assembly outlet casting. A gear is cut on the bearing which is engaged with a fractional hp 28 Vdc gearhead motor. This motor provides the power to rotate the drum when a signal is received from the RESCAF control. Four redundant circumferential seals are provided between the drum and casting. All body assembly components are constructed of corosion resistant materials, stainless steel, or treated aluminum.

Figure 6. RESCAF Body Assembly

BODY

# 5.2 RESCAF System Specifications & Weight

Max Rated Airflow		9221 cfm (11.3#/sec) (corr to 29.92 "Hg bar. and 80 F				
Scavange	Airflow		692 cfm @ 17.5" wg. (.847#/sec) corr to 29.92			
Compresse	d Air Requirement		175 SCFM @ 8 psig			
Electrica	1 Power Requirement		2.5 amps @ 28 Vdc			
Efficiency (AC Coarse test dust)			99.9% Filter 99.99% System			
Maximum Compressed Air Temperature		230°F at Cleaner Supply Port				
RESCAF We	ights:					
	Drive motor		7.5#			
	Filter element (4 required)		13.5# ea (54# total)			
	Body assembly (less turntable outlet casting)	e and	73.5#			
	Precleaner APU casting Turntable and outlet casting Misc gaskets, bolts, nuts		56.5 # 15.25# 173 # 6.75#			
	Tot	al Weight	386.75#			
Coordinates of RESCAF CG STA ("as shipped" configuration with WL filters installed) BL		30.25 44.75 31.30				

# 5.3 Dust Test Summary

Two aborted attempts to complete a 100 hour dust test were made (December 1988 and January 1989). Dust leakage through the filter medium were the cause. A decision was made to use the proven filter medium developed for the MIAI SCAF system after the two aborted attempts. Filters were constructed and this third (successful) test was started 16 February 1989.

The test plan is summarized for dust testing on figure 7. A summary of the pressure loss vs time (i.e., life characteristics) is presented on figure 8. A summary of efficiency data gathered during this test is presented on table 3. The test setup is shown on figure 9.

#### 5.4 Vibration Tests

Vibration testing is summarized in Appendix A. During vibration testing a turntable motor gearbox failed. The cause of this failure was traced to an interference between the pre-cleaner and turntable which occurred during vibration. The pre-cleaner inlet frame was reinforced to prevent further interference. The interference caused the load on the motor to increase beyond acceptable limits.

#### 5.5 Shock Tests

Shock testing is summarized in Appendix B. During shock testing, the woodruff key connecting the drive gear to the turntable motor sheared. This key will be changed to a straight key (for increased contact area) on FSED units. A copy of the motor manufacturer failure report is attached to this report.

#### Figure 7. Dust Test Plan

Item to be Tested: TME Self Cleaning Air Filter System (pn 9800P200) (Testing with actual compressor, compressor ductwork,

and pressure regulation equipment to be determined.)

Test Objective:

The objective of this test is to demonstrate that the SCAF system designed for the Transverse Mounted AGT 1500 gas turbine (TME) meets or exceeds the design dust capacity and efficiency levels currently considered necessary for effective and efficient operation of the AGT 1500 propulsion system in an M1 main battle tank. Since specific performance goals are not stated in the current contract, the following performance guidelines are considered by Donaldson Company to be minimum acceptable levels:

Overall System Efficiency (accumulative) . . . 99.99% when challenged with SAE coarse test dust

Filter element efficiency (accumulative . . . . 99.90% on precleaned dust.

Dust capacity - the total pressure loss characteristic from the inlet of the RESCAF pre cleaner to the outlet of the RESCAF filter elements should not rise more than twenty (20) inches of water gage (w.g.) when challenged with SAE coarse test dust at a concentration of .025 gm/ft<sup>3</sup> zero visibility for 20 hours, when tested at 80% of max engine airflow (air cleaner outlet air flow).

The test airflow rate for the TME SCAF was determined to be 7223 cfm (corrected to 29.95" Hg bar and 80°F) by Mr. Richard Horan, Textron Lycoming and agreed to by Mr. Harry Camplin, Donaldson Company, Inc. This airflow rate is based upon the engine max rated airflow being 10.7# sec. installed X .8 (based upon government requirement) plus a 239 cfm allowance for the NBC system.

#### Test Procedure:

The self-cleaning air cleaner system will be set up and prepared for dust testing with test equipment, fixtures, and ducting defined on the attached test setup sketch (figure 9).

The following test parameters will be set.

- Qs (scavenge airflow . . . . . . 655 cfm (corrected to 29.92" Hg. Barand 80°F) 80#/sec.

# Figure 7. Dust Test Plan (Cont)

Scavenge airflow is 7.5% of primary airflow at 100% rated power. Distribution of this airflow internal to the SCAF is 5% for the precleaner and 2.5% for the receivers.

Pn (pressure inside cleaning nozzles) . . . . 8.0 psig.

Dust Type . . . . . . . . . . . . . . . . SAE coarse test dust

During the test a periodic record will be kept (and corrections made to test procedures as required to assure set levels stay constant) of the following:

Ambient temperature (°F)

Barometric pressure (" Hg)

Relative humidity (%)

Upstream Mercury (UpsHg) changes affecting flow correction on all flow meters not automatically controlled.

On nozzle flow rate (cfm)

T comp outlet (nozzle compressed air temperature, °F)

A continuous record will be kept of all changes in filter element pressure drop. (Via a strip chart recorder or appropriate data recording device)

The test duration will be 100 hours.

The RESCAF system will be subjected to vibration for the duration of the test per the following:

Vibrator Setting: .00875 inc. DA with 10-100-10 Hz sweeps continuously with total sweep time of 10 minutes.

Absolute filters will be changed and overall system and filter element efficiency will be measured at the following intervals:

- 5 hours after the start of testing.
- 10 hours after the start of testing.
- 50 hours after the start of testing.
- 100 hours after the start of testing.

Schedule: This test is scheduled to be conducted November 5 - 15, 1988

Table 3. Life/Efficiency Summary Sheet

	System Effy-Cum	99.9992	99.9993	99.9993	99.9992	99.9992	99.9992	99.9992
	System Effy Es, 1	99.9992	99.994	99.9992	1666.66	99.9991	99.9992	99.9991
	Filter Effy-Cum	99.988	99.990	99.989	99.988	99.987	99.987	99.987
	Filter Effy Ef, 4	99.988	99.991	99.988	99.986	99.985	99.988	99.986
	Preciner Effy (e) Ep, %	93.5	93.5	93.5	93.5	93.5	93.5	93.5
	Cum Feed Preciner Rate Effy (e) x 0 vis Ep, %	1.82	1.95	1.87	1.87	1.90	1.91	1.92
rae 7878 6.90	Feed C Rate x 0 vis	1.82	2.05	1.79	1.87	1.97	1.98	1.95
5 F C	Dust Cum 1bs	332.00	785.50	1415.00	2365.50	3212.00	4016.50	5003.50
Project # 9260P Task # Customer: Textron Dust Type: SAE Coarse Inlet Airflow, cfm: Initial Filter dP: 6	Dust Fed Wd, 1bs	332.00	453.50	629.50	950.50	846.50	804.50	987.00
	Abs . Cum	1.14	2:40	4.55	8.61	12.25	15.09	19.03
	Abs . Gain Wa, gms	1.14	1.26	2.15	4.06	3.64	2.84	3.94
To JAE :or: HRC :m: RESCAF of 1	Filter delta P in H20	13:0	15.8	17.8	19.8	20.5	22.2	23.5
Date: To Tester: JAE Originator: HRC Test Item: RESCAF Page lofl	Segment Time Hours	7	8.5	13.5	19.5	16.5	15.6	19.4
	Cum Time Hours	7	15.5	29	48.5	65	90.6	100

 Absolute weight gains are measures of dust which passed through the filter and in a vehicle installation would be ingested by the engine.

NET STATIC PRESSURE DROP ACROSS FILTER ELEMENT (BOTTOM CYCLE) - IN. WG.

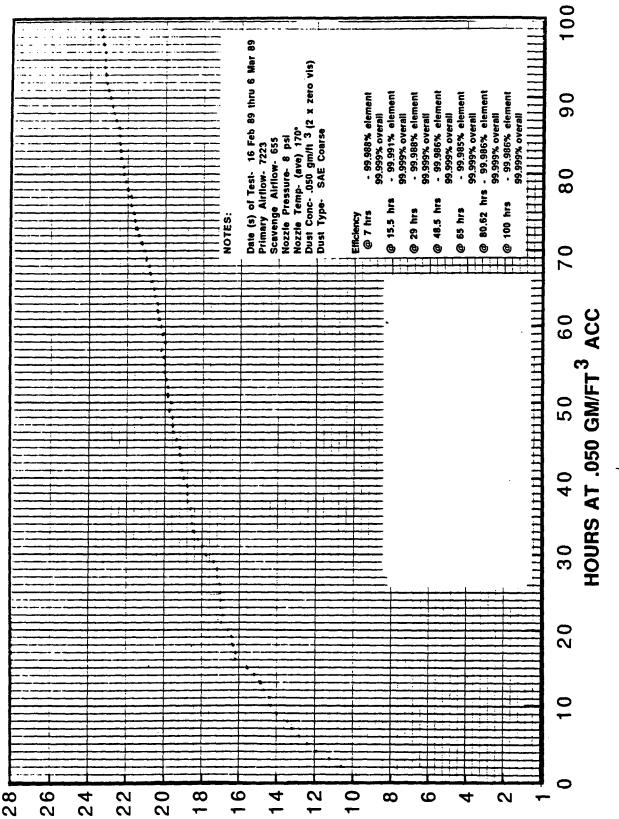


Figure 8. TME RESCAF Life Characteristics (9800P200 Air Cleaner Assembly)

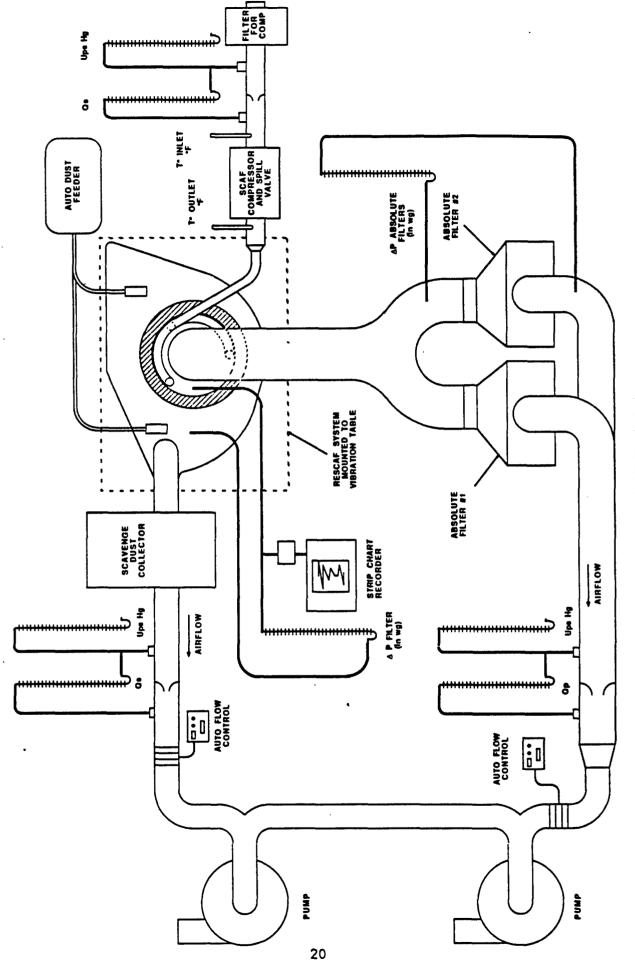


Figure 9. TME SCAF Test Setup

# 6.0 RESCAF Development Issues

## 6.1 Design Changes

During development of the RESCAF System, changes were made to RESCAF hardware to correct unanticipated problems that occurred. A complete list of these changes is as follows:

#### Change

Spacers added to increase gap between precleaner and body assembly by 3/4 inch.

Filter medium change to "Yuma" M1A1 SCAF medium.

Reinforcement of precleaner inlet frame.

Cutout added to precleaner to provide relief for APU duct.

Lock nuts welded in place (instead of Loctite) on element retainer bolt captive devices.

# Reason for Change

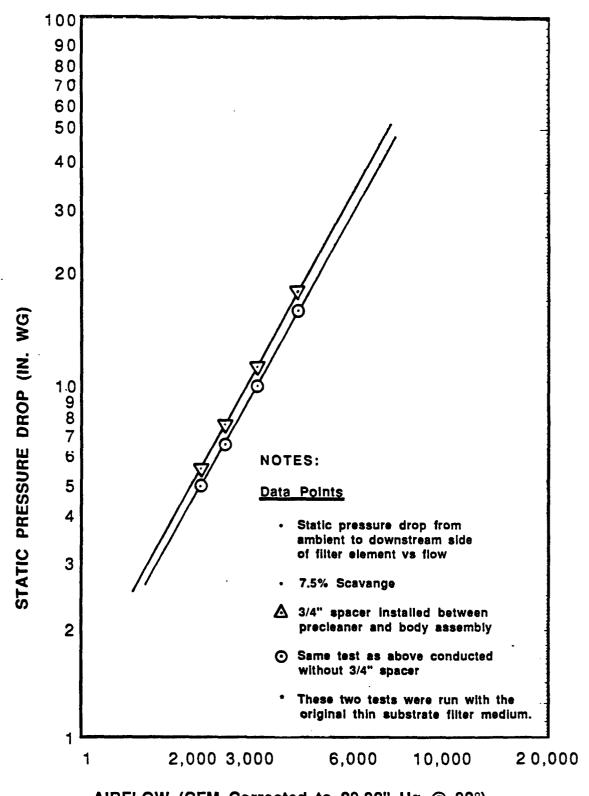
As a result of development test #5, it was shown that the pressure drop could be significantly reduced by adding a 3/4 inch spacer between the precleaner and body assembly (See figure 11).

Problems occurred with original thin substrate filter medium, and Donaldson Co. Inc. recommended a change to a "proven" medium. The first test of the RESCAF with the "Yuma" M1A1 SCAF medium also failed, due to a different (QA) problem. The second test with the "Yuma" medium was successful and data from this test is included in this report (See figure 8). The change to the thicker Yuma medium resulted in raising the pressure loss of the system. The result is quantified on figure 11.

During vibration testing, the precleaner distorted and came into contact with the filter turntable. This change was made to stiffen and prevent precleaner distortion.

This change was added to correct a design interference which occurred due to a misinterpretation of blueprints by the APU casting vendor.

Loctite did not hold when removing filters, which resulted in bolts not staying captive.



AIRFLOW (CFM Corrected to 29.92" Hg @ 80°)

Figure 10. Pressure Drop of RESCAF System with Thin Substrate Medium, With and Without 3/4" Spacer

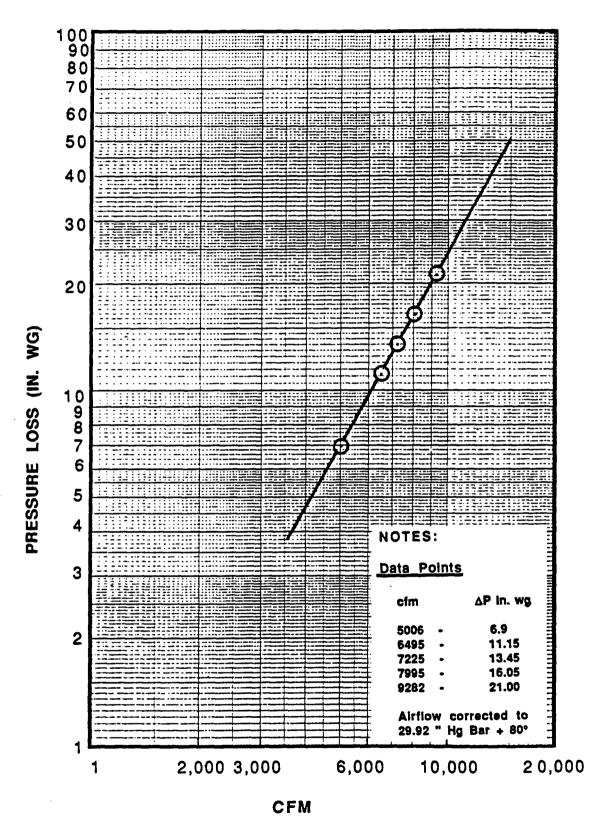


Figure 11. Pressure Drop of RESCAF System with Yuma Filter Medium and 3/4" Spacer

Secondary seal level changed on transition casting.

Motor gear material changed to hardened steel.

Secondary seal interfered with primary seal (design error).

Excessive wear noted after first dust test on soft gear.

# 6.2 Tests Run on RESCAF

The following tests were completed on RESCAF for developmental reasons, or at the request of Textron Lycoming, per contract agreement. Data from tests contracted by Lycoming is provided in this report.

NOTES: Data from tests conducted for developmental reasons is proprietary and is not part of this report.

	TEST	DATE	OBJECTIVE
1)	Static-pressure loss characterization of RESCAF, inlet to filter element and inlet to clean side of filter element.	6/28/88	Define system, pre- cleaner, and filter element pressure loss.
2)	Static-pressure loss characterization - clean air side of casting - control box, upstream element - receiver, upper - receiver, lower (7 1/2% scavange rate) (filter elements in and out)	6/28/88	Define filter element loss characteristics and calibrate receiver circuits (developmental).
3)	Static-pressure loss characterization (same data as test 2 except scav. rate at 10%).	6/28/88	Define SCAV. depression characteristics of RESCAF (developmental).
4)	Static-pressure loss characterization of RESCAF with precleaner removed.	6/28/88	Define precleaner distortion/duct losses (developmental).
5)	Static-pressure loss with precleaner raised 3/4".	6/30/88	Determine if pre- cleaner distortion level can be corrected if space added at element interface (developmental).
6)	Static-pressure loss characterization with precleaner raised 1 1/4".	7/5/88	Determine effect of increasing gap by 3/4" (developmental).

7)	Static-pressure loss characterization with precleaner raised 1/2".	7/5/88	Determine effect of decreasing gap by 1/2" (developmental).
8)	Static-pressure loss characterization with tapered gap 3/4" - 0.	7/11/88	Determine effect of running with tapered gap (developmental).
9)	Nozzle pressure vs. flow characteristics - hot nozzle (from M1A1 SCAF compressor) - nozzle 1 and nozzle measured - temperature gradient mounted	10/18/88	Determine nozzle pressure vs. flow characteristics at varying Temperatures. (developmental).
10)	Static-pressure loss characterization - new elements constructed with "proper" spacers - precleaner on and off	11/10/88	Determine effect of "proper" pleat spacing on system P with original precleaner spacing (developmental).
11)	Static-pressure loss characterization - same as test 10 except pre- cleaner raised 3/4"	11/14/88	Determine total effect of "proper" pleat spacing and 3/4" spacer (developmental).
12)	RESCAF rig test (8" x 8' filter) - test run for 20 hours at 10 x 0 vis.	11/22/88	Verify test parameters for full scale dust test on rig test (developmental).
13)	RESCAF rig/precleaner efficiency	11/18/88	Calibrate rig pre- cleaner to simulate full scale RESCAF.
14)	Full scale RESCAF dust test	12/16/88	Demonstrate life and efficiency characteristics of full scale RESCAF. (Test ran 51 hours and was shut down due to thin substrate medium failure.)
15)	Nozzle flow distribution test	12/19/88	Determine level of nozzle jet distortion on RESCAF.

16)	Full scale RESCAF dust test	1/16/89	Demonstrate life and efficiency characteristics of full scale RESCAF. (Test ran 18.87 hours and was shut down due to medium failure, traced to QA failure of supplier.)
17)	RESCAF nozzle and spill valve loss characterization	3/7/89	Characterize nozzle flow and pressure from spill valve to nozzle with Lycoming supplied spill valve manifold.
18)	Full scale RESCAF pressure drop test	2/16/89	Determine pressure loss characteristics of RESCAF with "Yuma" M1A1 medium.
19)	Full scale RESCAF dust test	2/17/89	Demonstrate life and efficiency characteristics of full scale RESCAF (test successful).
20)	RESCAF vibration test	3/27/89	Demonstrate vibration resistance.
21)	RESCAF shock test	5/12/89	Demonstrate shock resistance.
22)	RESCAF post shock and vibration efficiency test	6/15/89	Demonstrate RESCAF efficiency after shock and vibration tests.
23)	GDLS APU duct test	6/16/89	Determine GDLS manifold pressure loss characteristics.

#### 6.3 RESCAF Teardowns

The RESCAF system was disassembled three times during the laboratory test program. The purpose of these teardowns was to perform a complete post test inspection of the RESCAF system and determine if failures had occurred. The following paragraphs describe these teardowns and what was found.

## Teardown 1

The first teardown occurred immediately after the first dust test failure in December, 1988. The following conditions were noted and corrective action taken:

- Excessive wear was noted on the drive motor spur gear. This problem was corrected by changing requirements to specify a hardened drive motor spur gear and replacing the gear.
- Interference was noted between the primary and secondary turntable seals. This interference with the seal bolt heads resulted from a tolerance stack up. The gap was increased between these seals (drawing change), and the seals were reinstalled. No further problems were found on subsequent tests.
- The lock nuts on the filter element retainer assembly came loose when removing the filters. These nuts were welded in place after assembly for subsequent tests. This fix worked well and the change was incorporated in the design.

#### Teardown 2

The second teardown occurred immediately after the third successful dust test, in February, 1989. The RESCAF was found to be in good condition throughout, with only normal wear noted.

#### Teardown 3

The third teardown occurred after completion of the dust test which followed shock testing. The RESCAF was found again to be in good condition. The turntable seal manufacturer inspected the critical seals and figure 12 is an excerpt from the manufacturers letter report.

## 6.4 Recommendations for Next Generation Systems

For next generation systems Donaldson Company, Inc. recommends use of thin substrate filter medium (after successfully passing development tests) because of the lower system pressure loss and longer life benefits it would provide. An onboard diagnostics system is also recommended because it would reduce trouble-shooting time in the event of a problem.

Donaldson Co. Inc. recommends that an alternate air compressor for RESCAF be developed. This recommendation is made because the compressor chosen is considerably larger and capable of performance in excess of that required for this application. Donaldson Co. originally recommended (for the 4 speed RESCAF) a compressor with pressure capability of 22.5 psig and flow capabilities of 280 SCFM. Development work during the summer of 1988 indicated adequate performance could be achieved with a compressor capable of only 12 psig and 175 SCFM. Recommended pressure in the cleaning nozzles dropped from 15 psig to 8 psig during this period. This drop in pressure is necessary to prevent filter medium failure which may occur at higher pressures. In addition, a lower operating pressure should result in reduced compressor size and cost.

Donaldson Co. Inc. also recommends that an alternative RESCAF drive motor be developed. This is recommended because some problems with the motor occurred during RESCAF development.

Donaldson Co. Inc. would recommend the development of a composite body assembly to reduce system weight; however, since additional changes to the body are planned, and tooling costs are high, this change should only be implemented when the design frozen.

Donaldson Co. also recommends that the larger 1 1/2 inch diameter separator tubes continue to be used in the pre-cleaner for this RESCAF and with subsequent versions. This recommendation is made because of recent experience with plugging of smaller 3/4 inch diameter tubes with stones and vegetation at Yuma Proving Ground. The larger air passages of the 1-1/2 inch diameter tubes make them less prone to this plugging problem and, it is well established in the current M-1, MBT that the larger diameter tubes are more efficient at removing moisture from the intake airstream.

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Below are excerpts from seal manufacturers report on the condition of the seals after shock, vibration, and dust testing was completed.

- 1) Teflon member showed little wear. Approximately 0.002/0.003 (inches) teflon lost from sealing bead which equals less than 2 percent of total seal life. Teflon members machined shape and formed angle remained constant after deflection from installation and testing. Although spliced spring joints did not show excessive wear there was evidence of inconsistent wear at joints.
- 2) Seal mating (hardened running surface) showed no sign of overheating, or wear although there had been more polishing from teflon member than expected; I believe this was a result of grind marks across running surface. This can be corrected by Blanchard grinding the hardened running surface so all grind marks lay in circumferential direction.

# Finding on secondary seal

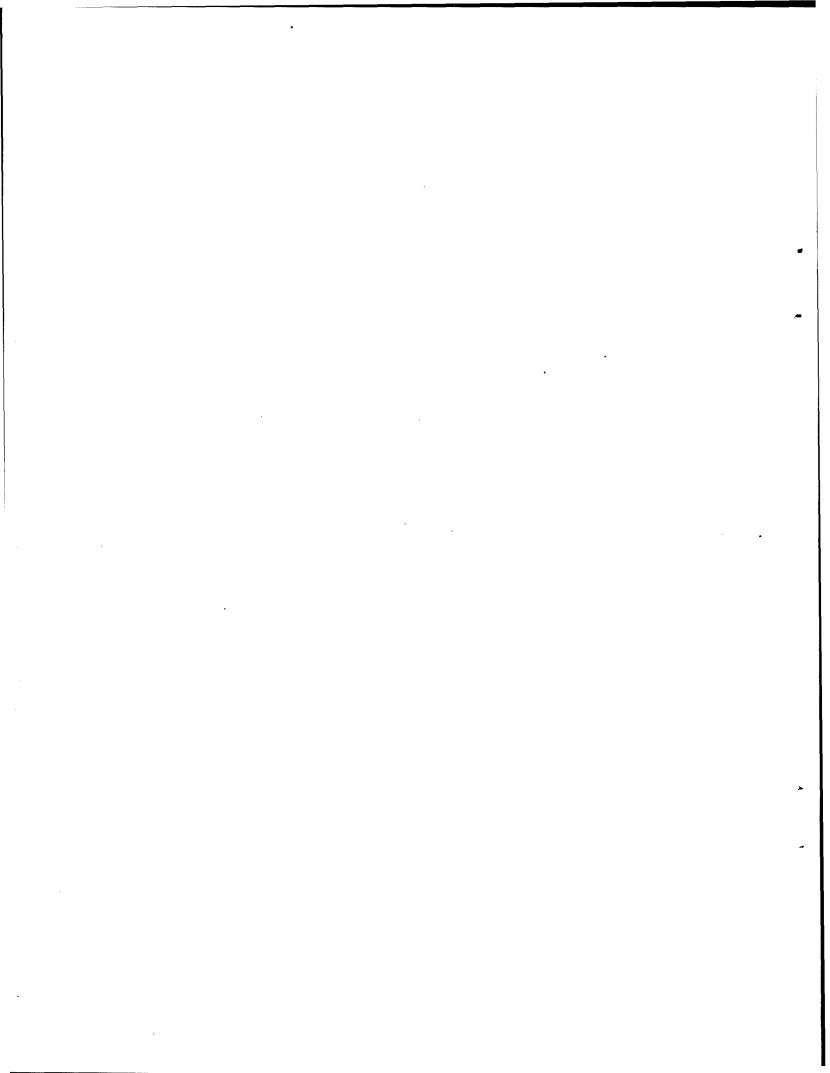
- 1) Seal element and mating surface looked excellent. Very little wear was noted on element. The seal interface looked as if only toe contact is being made at this time, and with continued running the contact pattern will increase with element wear.
- 2) The inside mating surface which the element runs against looked as if the seal had moved. Note there were two contact patterns made by the element. Each of which appeared approximately 0.040 (inches) in width and 0.125 apart. There were only a few number of hours on the first interface. Although both interface patterns appeared the same the initial running is at an accelerated rate until break-in has seated.

In conclusion, the overall performance appeared to be excellent.

Figure 12 (cont.)
Report on condition of the critical seals after teardown No. 3. These seals were used for all testing.

# APPENDIX A

VIBRATION TESTING



#### ENGINEERING REPORT Honeywell ວີປີ3-27-89 AVIONICS COPYLIST: PAGE PEC 2 48 8 1 X X 4 4 6 6 7 - 2760 3 DEFENSE SYSTEMS OF CONTRACT NUMBER SSUDIE Environmental Lab D7800-920 R Leach MN50-4100 G Hedstrom UNIT TESTED: MN11-1352 One Donaldson Corporation M1 tank air filter (SCAF), P/N S Nisbet 9800P0200, was vibration tested per Mil-Std-810D, Method Textron 514.2, Procedure VIII, Table 514.2-VI -- Sinusoidal Lycoming transportation vibration for tracked vehicles. See Table 3. (2) H Camplin Donaldson SCOPE: Corp. This report covers the procedures used for vibration testing, (2) D Swanson\* the response location descriptions from which data was MN50-4100 acquired during the vibration tests, and magnitude and phase Uniterm File\* plots of the resultant data. MN50-4100 D&E Files **SUMMARY:** MN50-4100 One Donaldson air filter was vibration tested per customer specifications from Mil-Std-810D. Testing was performed on an Unholtz-Dickie TA-4000 electrodynamic vibration machine. The air filter was instrumented at thirteen locations as specified by Donaldson Corp. and Textron Lycoming customers. (See Figure 1 and Table 1.) Vibration was induced in three axes and data was acquired at select locations for each KEYWORDS: Donaldson Co. axis. SCAF PROBLEMS: | Vibration During both lateral axis and axial axis vibration, bolts were sheared on the air filter's supporting struts and torque arms. Since these had no direct bearing on the test they were replaced and testing was resumed. The second 1/3 of the lateral axis vibration and the final 1/3 of the axial axis vibration, however, were run at only 70% of the full level as directed by Scott Nisbet of Textron Lycoming. After completion of lateral axis vibration, it was noted that a bolt, which held the gear box to the AGT 1500 tank engine (used as a fixture for the SCAF), had broken and a decision was made to run without the gear box for the vertical and ATTACHMENTS: axial axes. Tables 1 - 3 Figure 1 Appendices A, B, and C

DATE BOOK NUMBER	5 <del>0</del> −153	T3-Maf-8	)	15-Mar-85	1
RECHASTED BCampl	in <b>68</b> 1		wallan Kenny	Jum K	
DE DOHATUSON C	corporation		*Robert Leach	84 Cearl	3/28/89

HE-44A REV 12/78

#### PROCEDURE

The Donaldson Corp. Air Filter (SCAF) was mounted to an AGT 1500 Ml tank engine and coupled to a base plate for vibration testing. The filter was instrumented at thirteen different locations as shown in figure 1. Testing was performed per Mil-Std 810D, Method 514.2 Procedure VIII, Transportation vibration for Tracked Vehicles. The sinusoidal test profile was:

5-5.5 Hz. 1.0" D.A. 5.5-30 Hz. 1.5 G pk. 30-50 Hz. 0.033" D.A. 50-500 Hz. 4.2 G pk.

This profile was used for all three axes of vibration.

During vibration all thirteen response locations and the control response were recorded to analog tape and were also recorded real time on a Gen Rad 2515 analyzer. Control of the test was performed on a Gen Rad 2511 Vibration Control System.

The test sequence was randomly selected as:

1: Lateral Axis
2: Vertical Axis

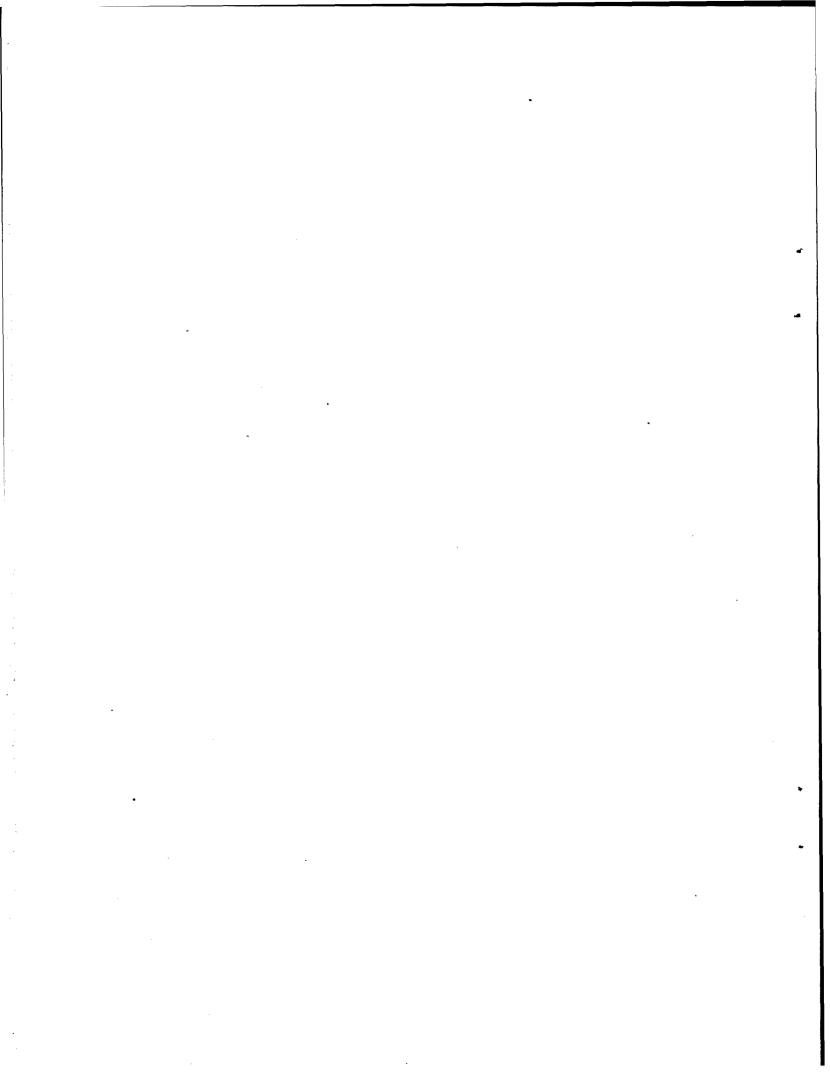
3: Axial Axis

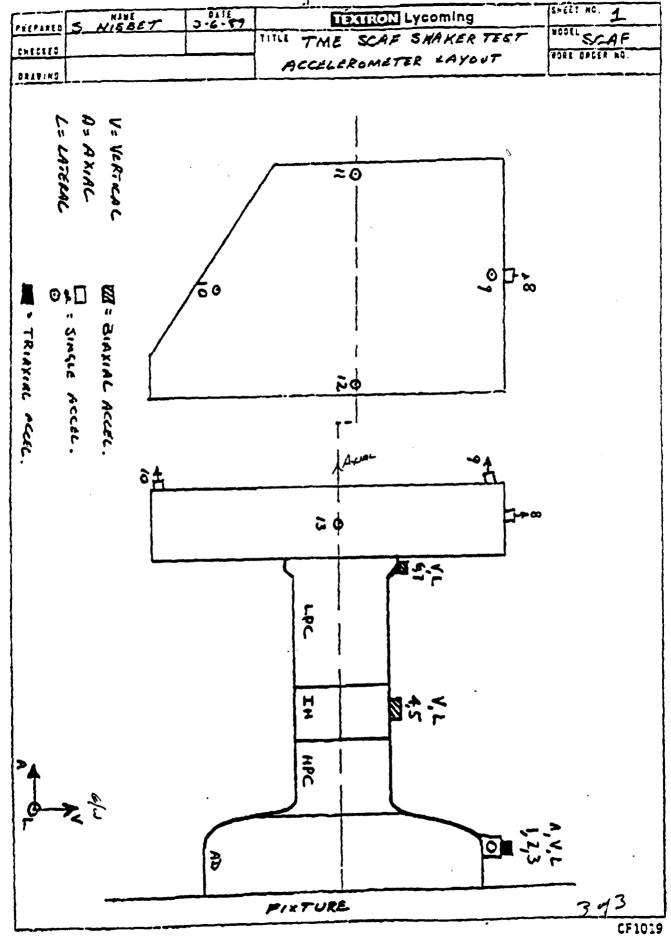
During vibration in the Lateral Axis, after 1/3 of the test had completed, a bolt sheared on a strut/torque arm supporting the air filter. The bolt did not have a direct effect on the focus of the test so it was replaced and the test was resumed at 70% of the full test level, as decided by the Textron Lycoming customer representative. After 2/3 of the test had completed the bolt sheared again and a decision was made to change from a three strut configuration to a four strut configuration. The remainder of the lateral axis and both the vertical and axial axis vibration runs were performed using this four strut configuration. Following the lateral axis vibration, it was noted that the gear box attachment to the AGT 1500 engine was loose. The gear box was removed and, since extensive rework would have been required to secure the gear box, the customer decided to run in the vertical and axial axes without the gear box attached. Vertical axis vibration was performed without problems. Axial axis vibration also saw the bolt shearing problem on the torque arms. After completing 1/3 of the vibration and again after completing 2/3 of the test run, a bolt sheared on one of the supporting struts. Again the bolt was replaced each time and the test resumed until it was completed. No other failures occured.

During each axis, vibration response signals of interest to the customer were acquired and are included here in Appendices A-C (see also figure 1 and tables 1 and 2 for locations). The response locations of concern were:

-Lateral axis: Locations 3,5,7,11,12,13 -Vertical axis: Locations 2,4,6,8,9,10,11,12 -Axial axis: Locations 1,6,7,9,10,11,12 For each response location a magnitude, a transfer function magnitude, and a transfer function phase plot were generated. The relative axis of orientation of each response location is included in the calibration sheet (table 1).

The test was completed on Wednesday, March 15, 1989, to the customer's approval.

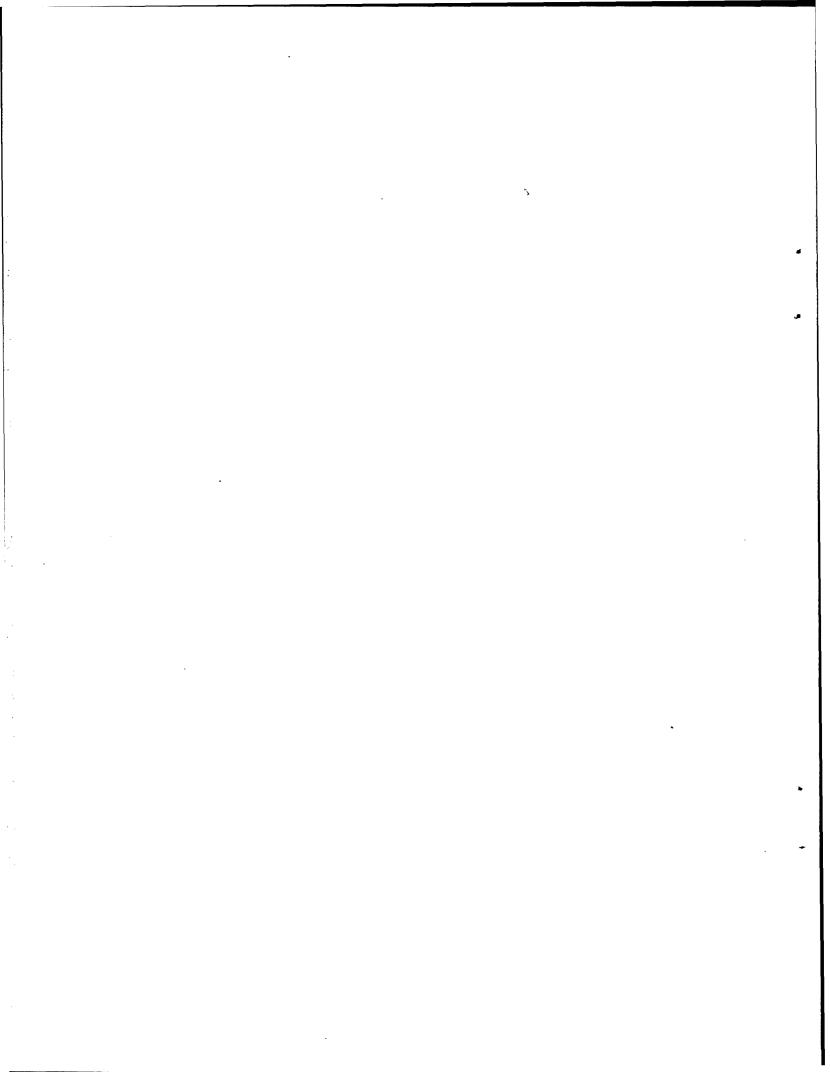


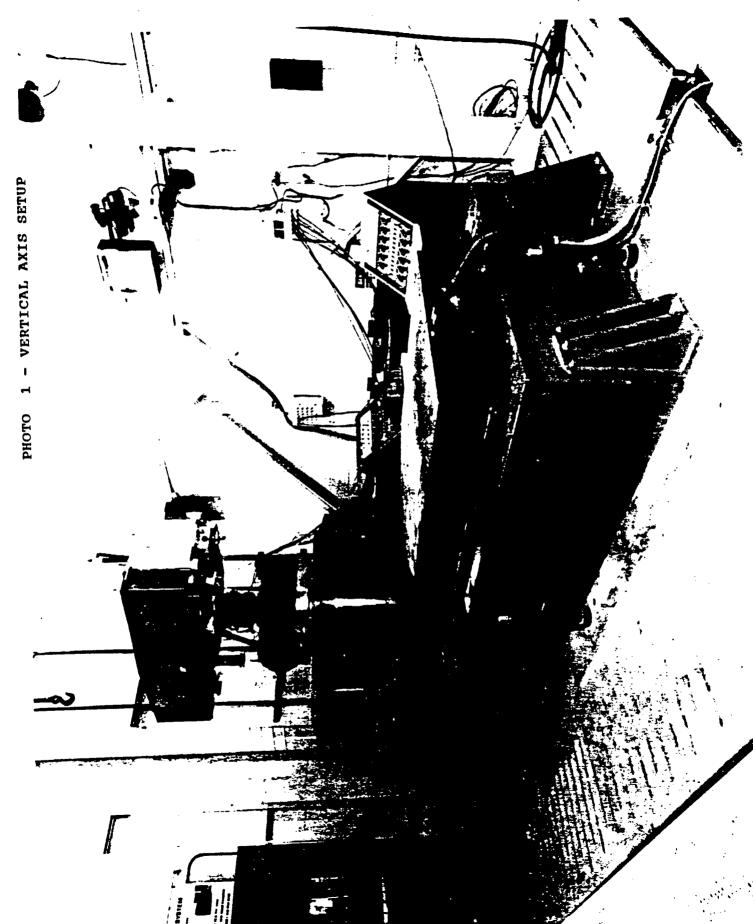


200 : 35 H a

FIGURE 1

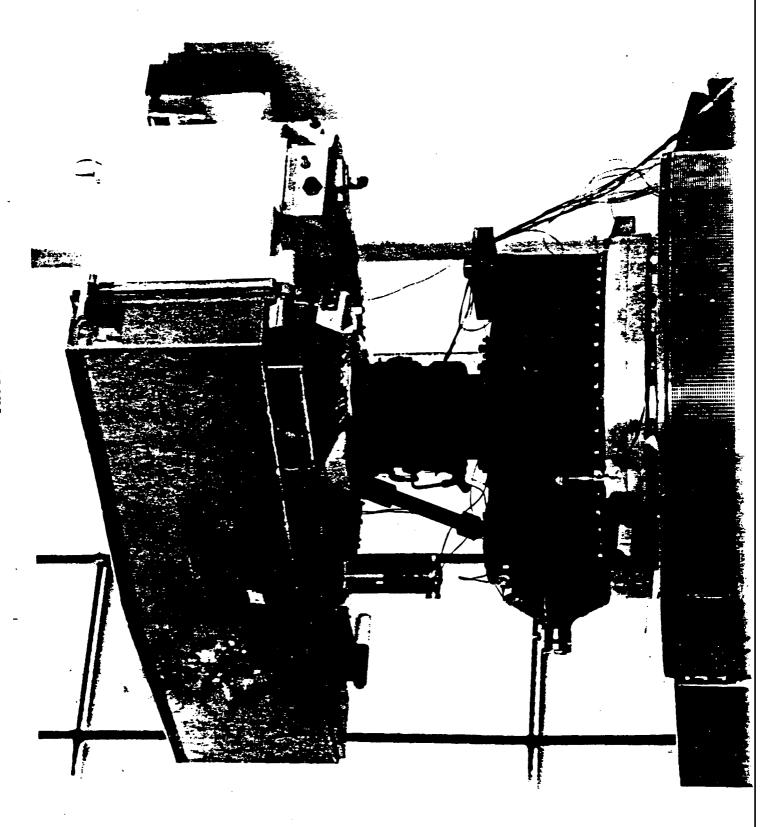
90:2: 69. 9 eta

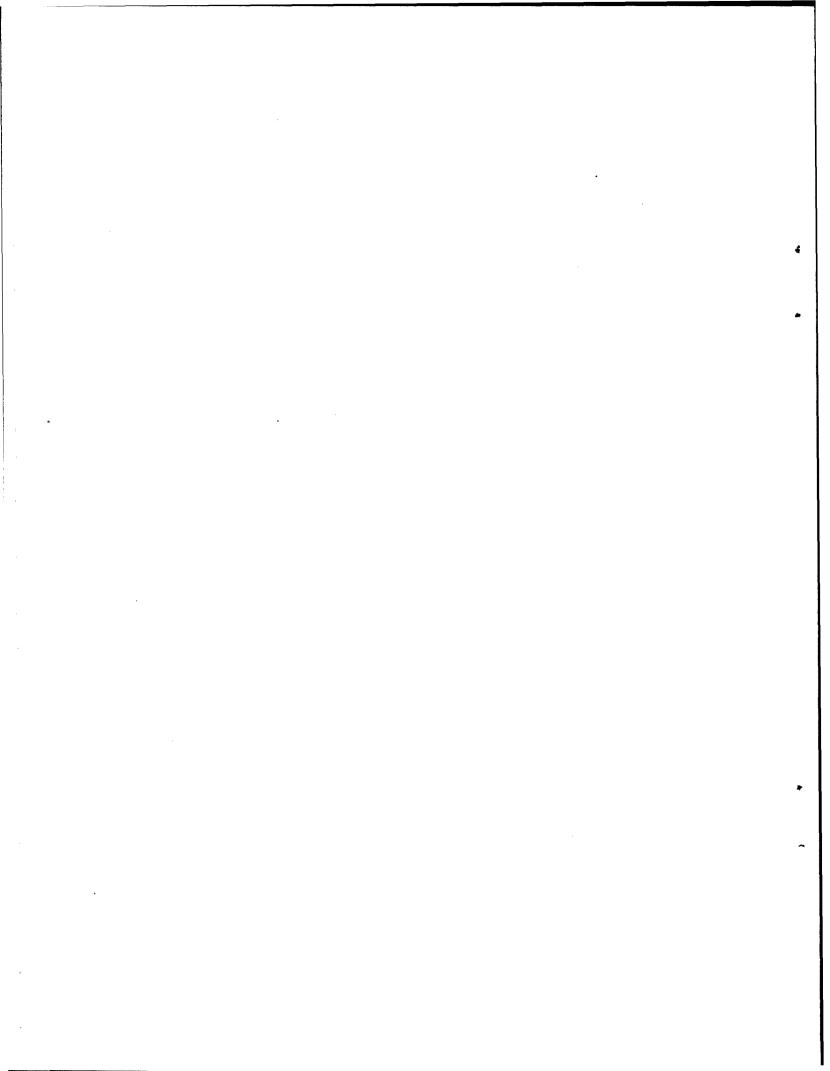






VERTICAL AXIS SETUP





TITLE Daycldson SCAF (MI)

TABLE 1

DEXM 30, 299 D

TALLY NO. G. 2068-11 0001 2760

	COMMENTS	Cal Bin - 12.90	ALOCE .	V 40.7 = 4.0 × 7 × 5		except KF49 are Enderen	Model 2229C's, KF49 .5	an Endevice 2002C.	Control Accel 1, 4 Dytran	recel solo Ag		-Not used - Space rostriction	replaced by KF49		AM Coxes are define	relative	RESPONSE AKES: (4 VI.RT - VERTICAL IS MANL - AK
	TAPE CHAN.	8	~	h	S	ی	7	8	6	10	1		hı	13	۳۱		7-
	F.S.	100/4	10~1/6	10 20/4	10.01/4	10.01/9	10mV/9	10	10	10 mV/g	10 20/9	10 mV/a	10~0/4	10~1/4	10~V/c	9.95 mV/4	1 Volt
(av) 660	CHG. ANP.		6	~	4	ζ	9	7	8	6	10	1.1	13	13	15	Dyiren 4131 Current Salvee	
	SENS.	3.215	2.885	3.30	3.19	3.23	3.09	3.04	3.26	3.30	2.84	2.78	3.46	3.085	1.43	,	
Patch Line	LINE *	~/ /_	3/3	4/2	5/	2/5	6/7	7/8	8	0/	10/01	=	he ci	13/14	61/11	15/	
	ACCEL.	8K69	BK 39	RKSY	BK63	BK65	BF83	BF83	TA63	BK70	iš B 66	BRS6	8D 56	B A 3.7	KF49	394	(Fre Synch)
RESP. /	AXIS/Lac	VERT 6	LAT / 7	VERT 8	A4196 9	AKIAL 10	AMOU II	AXIAL/ 12	AXIAL	YERT / 2	LAT / 3		LAT / S	LAT / 13	VERT 4	Dyna tolog	COLA

### DATA ACQUISITION RESPONSE LOCATIONS

#### REQUESTED BY TEXTRON LYCOMING

### LATERAL AXIS EXCITATION:

Response locations: 3,5,7,11,12,13

-Magnitude plots

-Transfer functions relative to Input

(Magnitude and Phase plots)

-Transfer Function of:

location 11 vs. location 12 (Magnitude and Phase plots)

### VERTICAL AXIS EXCITATION:

Response locations: 2,4,6,8,9,10

-Magnitude plots

-Transfer functions relative to Input

(Magnitude and Phase plots)

-Transfer Function of:

location 9 vs. location 10 (Magnitude and Phase plots)

-Magnitude plots of: locations 11 and 12

### AXIAL AXIS EXCITATION:

Response locations: 1,6,7,9,10,11,12

-Magnitude plots

-Transfer functions relative to Input (Magnitude and Phase plots)

-Transfer Function of:

location 9 vs. location 10 location 11 vs. location 12 (Magnitude and Phase plots)

### TME SCAF VIERATION TEST PROCEDURE

Ref.: MIL-STD-810C, Method 514.2

Equipment category f (equipment installed in ground vehicles)

Procedure VIII - proceed according to section 4.5.1.3

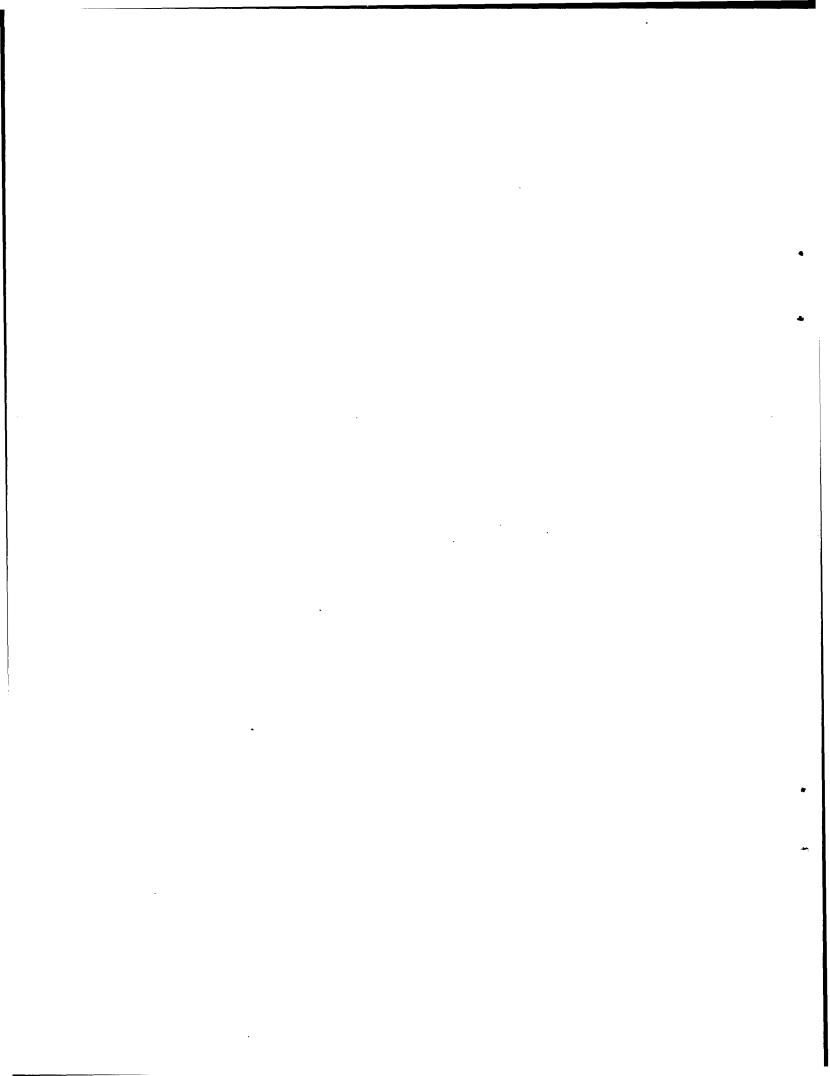
Applicable table: 514.2-VI Applicable figure: 514.2-6

4.5.1.3 Cycling. The test item shall be vibrated along each axis in accordance with the applicable test levels. frequency range, and times from the applicable tables and figures. The frequency of applied vibration shall be swept over the specified range logarithmically in accordance with figure 514.2-10. The specified sweep time is that of an ascending plus a descending sweep and is twice the ascending sweep time shown on figure 514.2.10 for the specified range.

From table 514.2-VI for tracked vehicles:
Sinusoidal cycling time, 30 minutes/1000 miles, not
to exceed 3 hours per axis.
Maximum cycling time, 3 hours per axis
Sweep time, 15 min for 5-500-5 Hz
Curve W

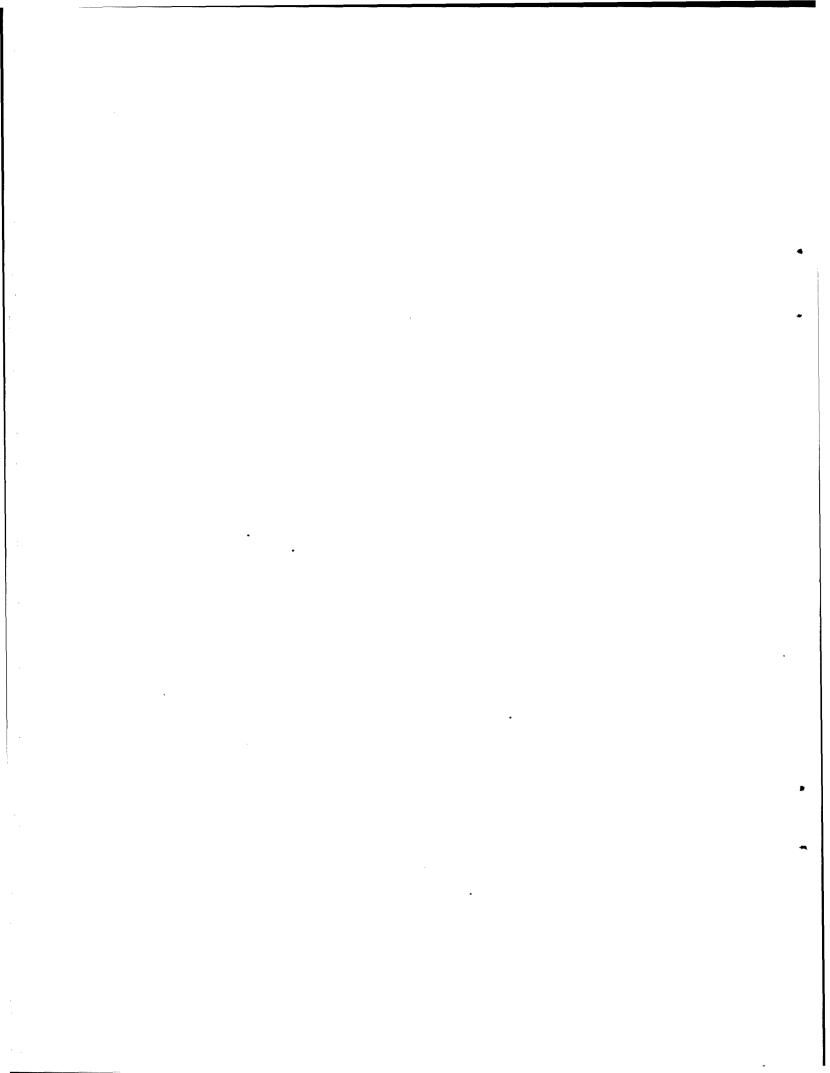
Vibration input, from figure 514.2-6, Curve W: 1.0 inch double amplitude from 5 - 5.5 Hz 1.5g constant acceleration from 5.5 - 30 Hz 0.033 inch double amplitude from 30 - 50 Hz 4.2g constant acceleration from 50 - 500 Hz

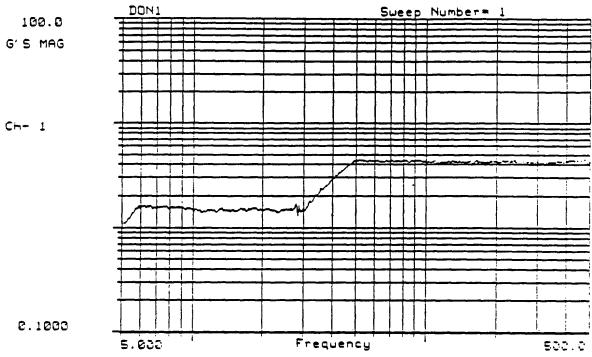
TABLE 3



# APPENDIX A1 LATERAL AXIS INPUT

13-MARCH-1989 3-strut Configuration





DON1

3/13/89 CH- 1: CONTROL DONALDSON SCAF TESTING

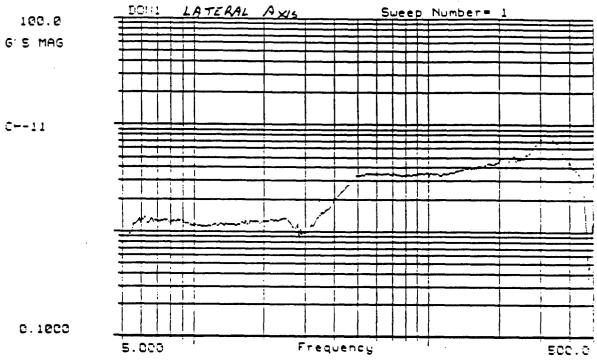
		١
ı	6	

### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	lateral	Unfiltered
•		Tape Chn. 1
Resp. Loc	CONTROL	Footage
Resp. Accel	394	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot .



DOM1

3/13/85 CH-11: LINE #10 loc #3

DONALDSON SCAP TESTING

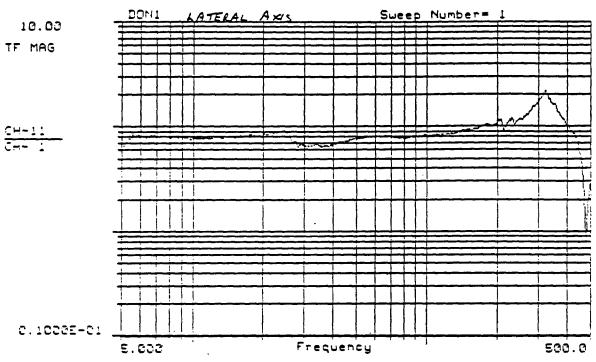
-

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
•		Tape Chn. 11
Resp. Loc	#3	Footage
Resp. Accel	8866	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



2001

CH-11: LINE #10 Loc #3

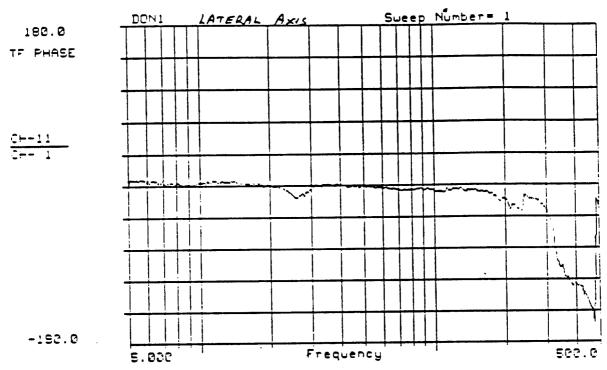
3/13/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,		Tape Chn. 11
Resp. Loc	#3	Footage
Resp. Accel	8866	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



1781

CH-11: LINE #10 400 #3

3/13/85 CH- 1: CONTROL DONALDSON SCAF TESTING

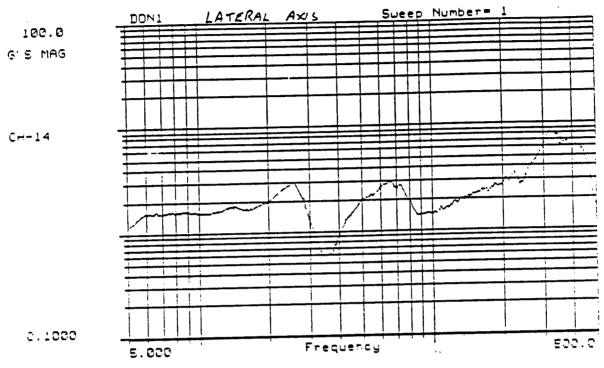
Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
•		Tape Chn. 11
Resp. Loc	#3	Footage
Resp. Accel	8866	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Transfer Function Phase



DOM:

3/13/89 CH-14: LINE #13 40c #5

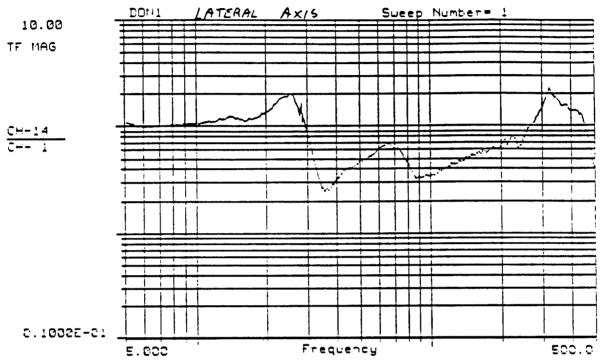
DOMALDSON SCAF TESTING

	Honeywell	Hopkins	Sine	Vibr	ation
OEXM #	30,2990	Te	est Da	ate	13-MAR-89

Input Axis	Lateral Lateral	Filtered YES Unfiltered
Resp. Axis	Percia.	Tape Chn. 14
Resp. Loc	#5	Footage
Resp. Accel	BD56	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



IO'-1

CH-14: LINE #18 LOC # 5

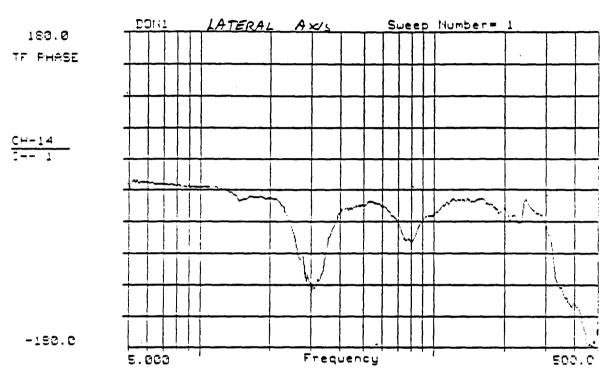
3/13/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 14
Resp. Loc	#5	Footage
Resp. Accel	8D56	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNÝ

Donaldson Air Filter (SCAF)



PON1

C--14: LINE #13 400 # 5

1/13/89 CH- 1: CONTROL IDMALDSON SCAF TESTING

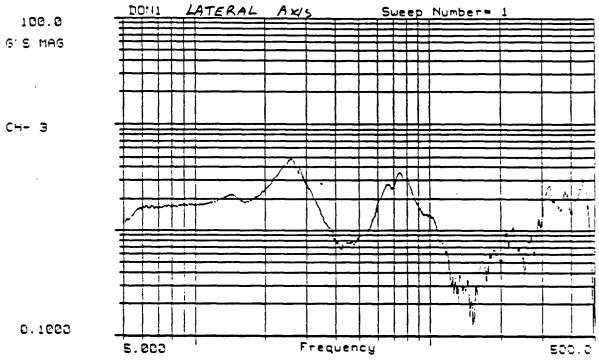
E

## Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 14
Resp. Loc	#5	Footage
Resp. Accel	BD56	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Transfer Function Phase



DONI

2/13/RS CH- 3: LINE #2 Loc # 7 DOMALDSON SCAF TESTING

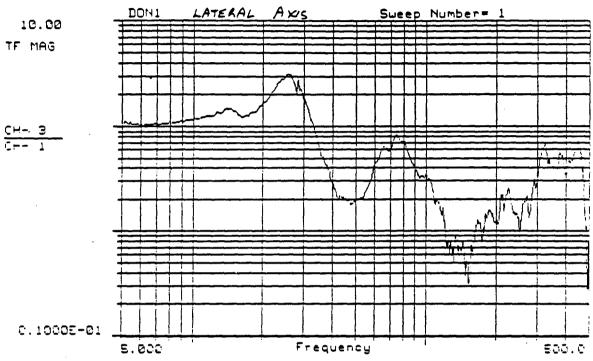
Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 3
Resp. Loc	<b>#</b> 7	Footage
Resp. Accel	BK39	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



DOM:

CH- 3: LINE #2 400 #7

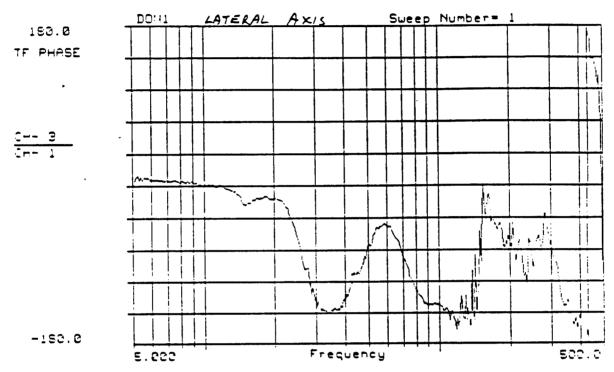
3/13/85 CH+ 1: CONTROL DOMALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 3
Resp. Loc	#7	Footage
Resp. Accel	BK39	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



E2:41

CH- 3: LINE #2 Loc #7

EVIEVES CH- 1: CONTROL ETHALDSON SCAF TESTING

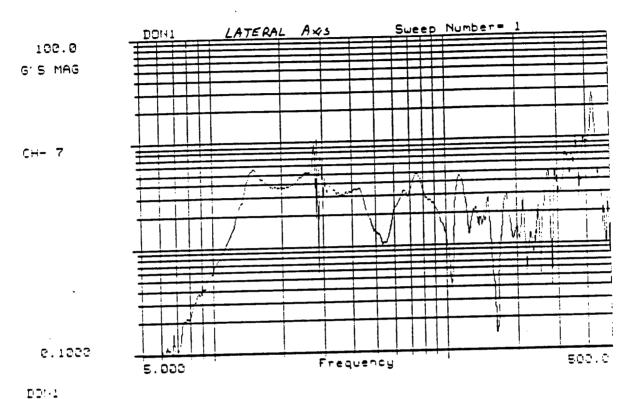
E

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral Lateral	Filtered YES Unfiltered
Resp. Axis	Fereign	Tape Chn. 3
Resp. Loc	#7	Footage
Resp. Accel	BK39	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Transfer Function Phase



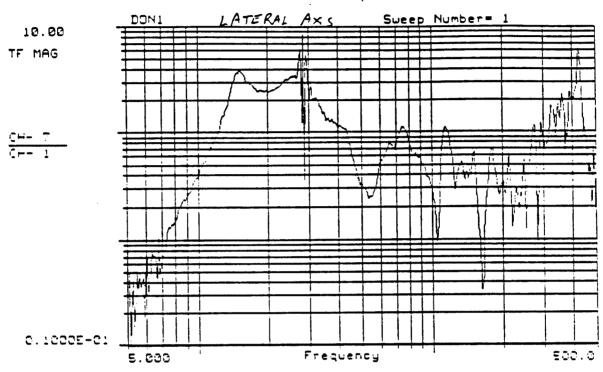
3/13/ES CH- 7: LINE #E Loc #/|
DONALDSON SCAF TESTING

Ε

E	Honeywell	Hopkins Sine Vibration
OEXM #	30,2990	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
D	#11	Tape Chn. 7 Footage
Resp. Loc Resp. Accel	• • •	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



E:0:41

CH- 7: LINE #6 Loc # 11

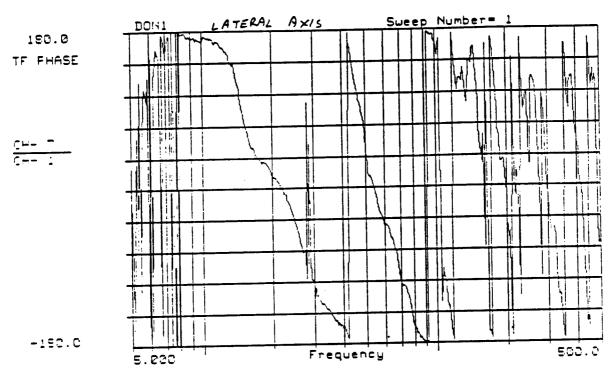
3/13/89 CH- 1: CONTROL DONALDSON SCAF TESTING

E

## Honeywell Hopkins Sine Vibration

30,299D	Test Date 13-MAR-89
Lateral	Filtered YES
Axial	Unfiltered
	Tape Chn. 7
#11	Footage
BF83	Filename Don1.swp
ROOM	Operator A. KENNÝ
	Lateral Axial #11 BF83

Donaldson Air Filter (SCAF)



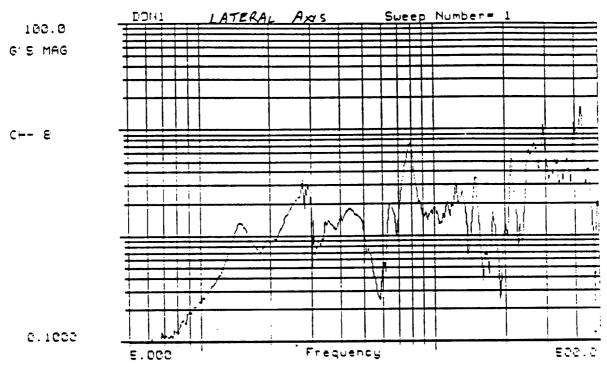
DORT CH- 7: LINE #8 40c #1

3/13/83 CH- 1: CONTROL DONALDSON SCAF TESTING

_	Honeywell	Hopkins Sine Vibration
OEXM #	30,299D	Test Date 13-MAR-89
Input Axis Resp. Axis	Lateral Axial	Filtered YES Unfiltered Tape Chn. 7
Resp. Loc Resp. Accel Test Temp	#11 BF83 Room	Footage Filename Don1.swp Operator A. KENNY

Donaldson Air Filter (SCAF)

Transfer Function Phase



ET'SI

EXISES CH- 8: LINE #7 Loc #12 DOMALDSON SCAF TESTING

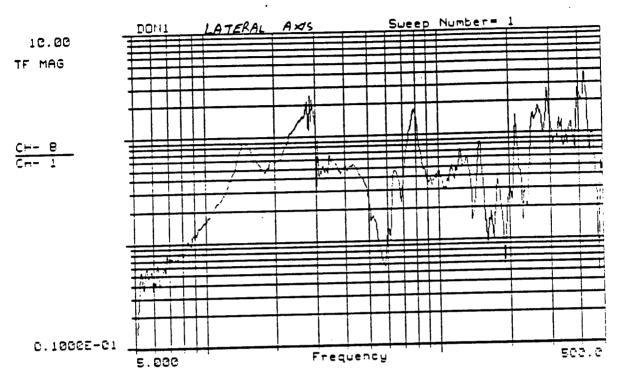
Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. B
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



DOM1

CH- E: LINE #7 Loc #12

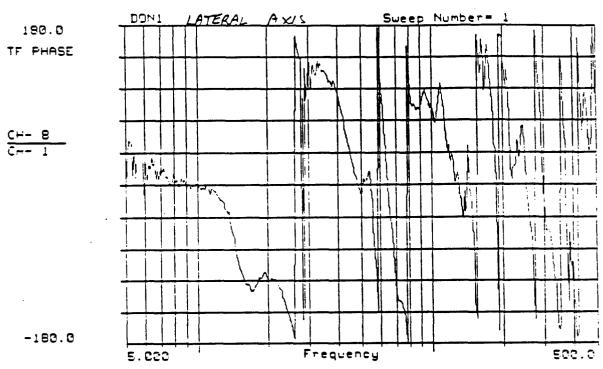
3/13/89 CH- 1: CONTROL DOMALDSON SCAF TESTING

E

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DOM:

CH- 8: LINE #7 Loc #12

EVIBURE CH- 1: CONTROL DOMALDSON SOAF TESTING

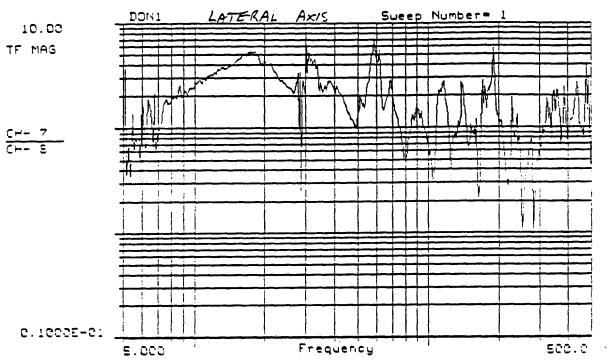
E

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
• .		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Transfer Function Phase



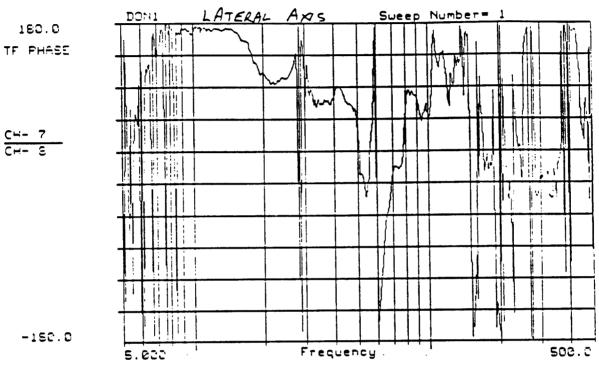
CH- 7: LIME #8 Loc # # SX18/88 CH- 8: LIME #7 Loc # /2 DONALDBON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
•		Tape Chn. 7/8
Resp. Loc	#11 VS. #12	Footage
Resp. Accel	BF83/BF82	Filename Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donalds. ... Filter (SCAF)



IDM1 CH- 7: LINE #8 LOCATION # 11

EV13/89 CH- 8: LINE #7 Loc # 12

PONALDEDN SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

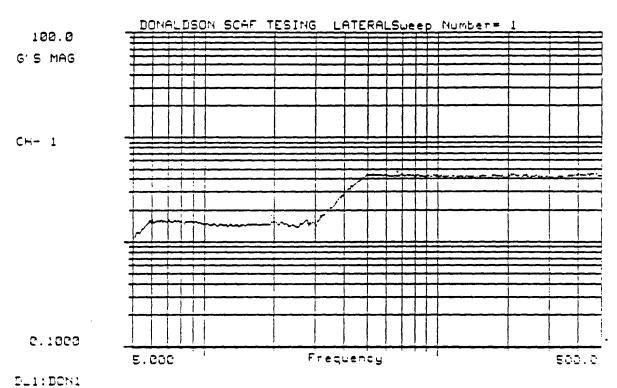
30,299D	Test Date 13-MAR-89
Lateral	Filtered YES
Axial/Axial	Unfiltered
	Tape Chn. 7/8
#11 VS. #12	Footage
BF83/BF82	Filename Don1.swp
ROOM	Operator A. KENNY
	Lateral Axial/Axial #11 VS. #12 BF83/BF82

Donaldson Air Filter (SCAF)

Transfer Function Phase

APPENDIX A2 LATERAL AXIS INPUT

14-MARCH-1989 4-strut Configuration



3/14/89 CH- 1: CONTROL

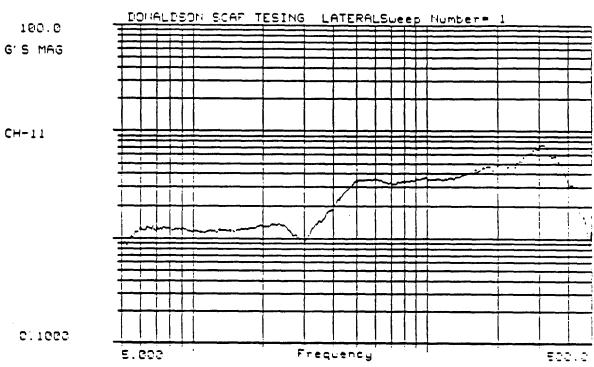
Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 1
Resp. Loc	CONTROL	Footage
Resp. Accel	394	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



3/14/83 CH-11: LINE #10 LOC#3

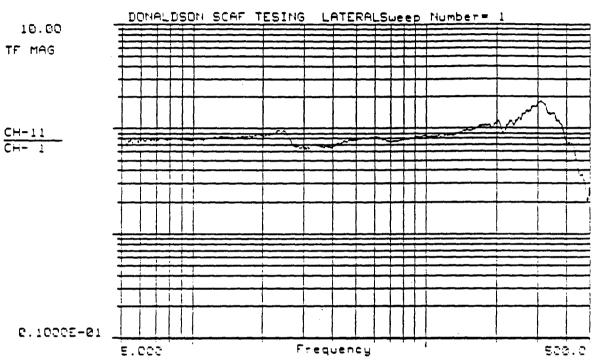
DONALDEON SCAF TESTING

#### Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
Resp. Loc	#3	Tape Chn. 11 Footage
Resp. Accel	<b>B</b> B66	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



CH-11: LINE #10 406#3

3/14/89 CH- 1: CONTROL DOMALDSON SCAF TESTING

E

# Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 11
Resp. Loc	#3	Footage
Resp. Accel	<b>8</b> 866	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

DONALDSON SCAF TESING LATERALSweep Number= 1

CH-11
CH-1:
CH

DL1: DON1

CH-11: LINE #10 Lec #3

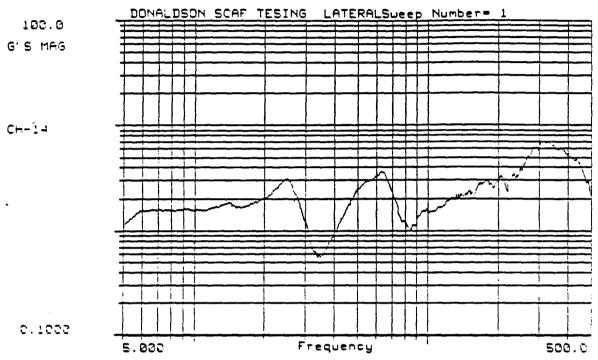
3/14/83 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
Resp. Loc	#3	Tape Chn. 11 Footage
Resp. Accel	8866	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DDN1

3/14/89 CH-14: LINE #13 LOC #5

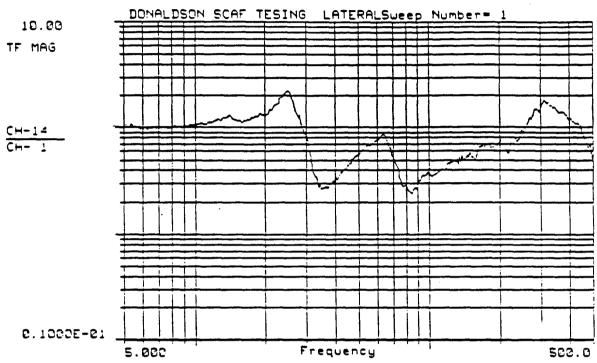
DONALDSON SCAF TESTING

#### Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
•		Tape Chn. 14
Resp. Loc	#5	Footage
Resp. Accel	BD56	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DCN1

CH-14: LINE #13 LOC#5

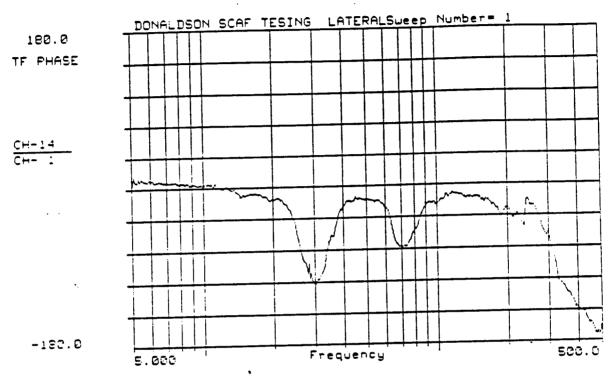
3/14/89 CH- 1: CONTROL BONALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
·		Tape Chn. 14
Resp. Loc	<b>#5</b>	Footage
Resp. Accel	BD56	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1: DON1 CH-14: LINE #13 404 #5

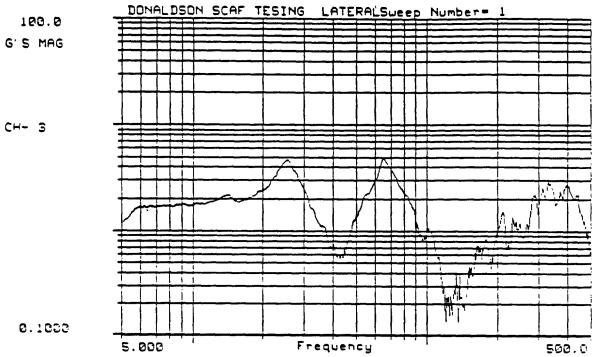
3/14/89 CH+ 1: CONTROL DONALDSON SCAF TESTING

E

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis Resp. Axis	Lateral Lateral	Filtered YES Unfiltered Tape Chn. 14
Resp. Loc Resp. Accel Test Temp	#5 BD56 ROOM	Footage Filename DL1:Don1.swp Operator A. KENNY

Donaldson Air Filter (SCAF)



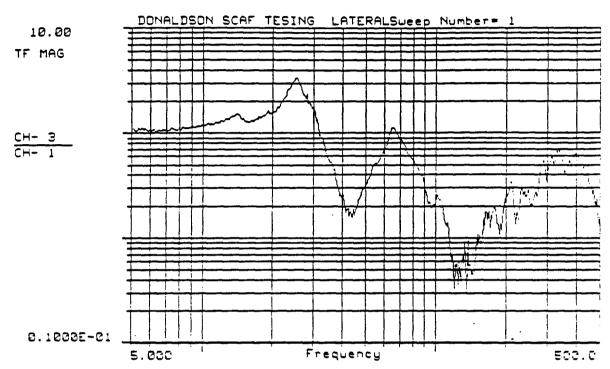
3/14/89 CH- 3: LINE #2 LOC #7
DDNALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 3
Resp. Loc	#7	Footage
Resp. Accel	BK39	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



CH- 3: LINE #2 Loc #7

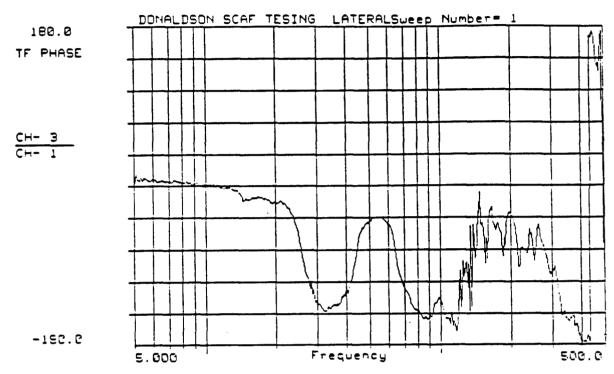
3/14/85 CH- 1: CONTROL DOMALDSON SCAF TESTING

E.

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 3
Resp. Loc	#7	Footage
Resp. Accel	BK39	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DON1 CH- 3: LINE #2 Loc #7

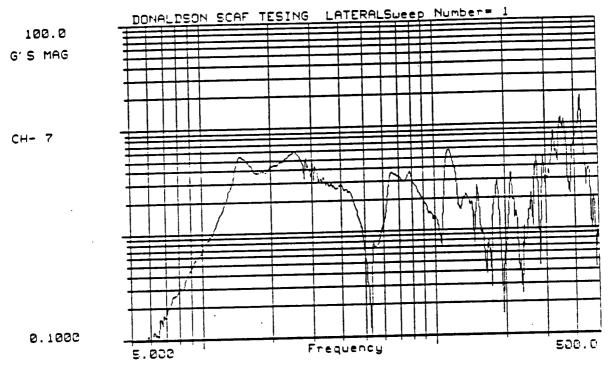
3/14/89 CH- 1: CONTROL 'DONALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 3
Resp. Loc	#7	Footage
Resp. Accel	BK39	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



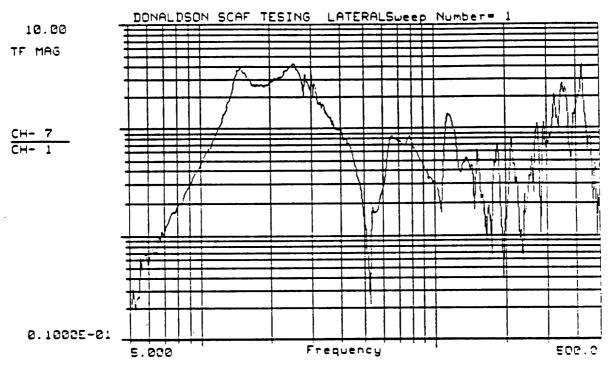
3/14/89 CH- 7: LINE #6 40c # N DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 7
Resp. Loc	#11	Footage
Resp. Accel	BF83	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DDN1 CH- 7: LINE #5 Loc ##

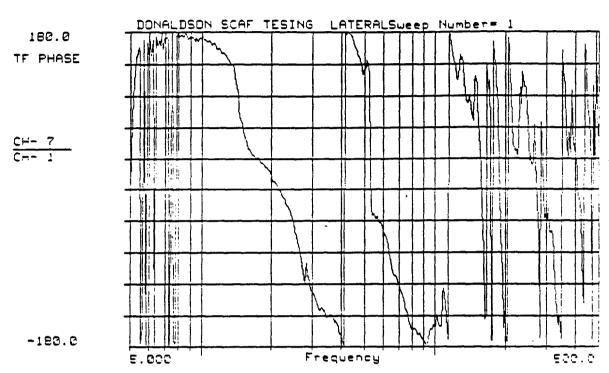
3/14/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 7
Resp. Loc	#11	Footage
Resp. Accel	BF83	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



CH- 7: LINE #6, LOC #11

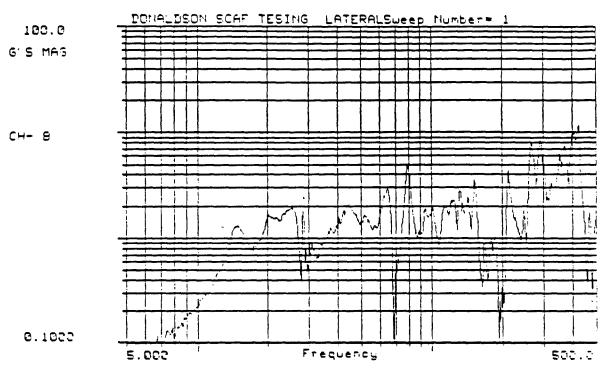
3/14/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 7
Resp. Loc	#11	Footage
Resp. Accel	BF83	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



3/14/83 CH- 8: LINE \*T 400 #17

DONALDSON SCAF TESTING

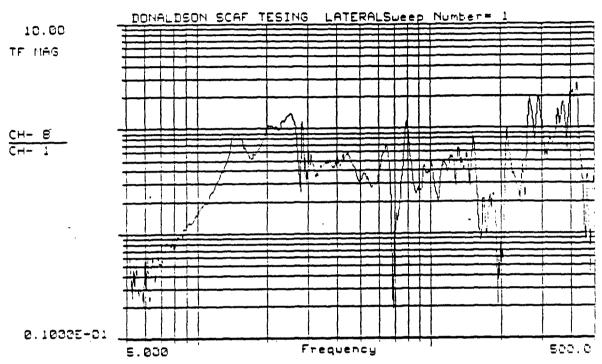
Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



CH- E: LINE #7 Loc # 12

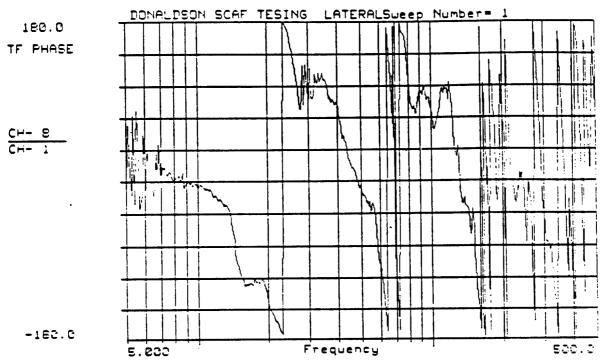
3/14/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date: 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF) .



DL1:DDN1

CH- E: LINE #7 LOC # 13

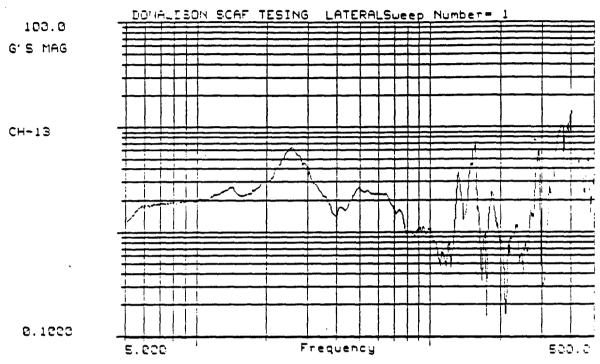
3/14/23 CH- 1: CONTROL DONALDSON SCAF TESTING

#### Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	<b>B</b> F82	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DOH1

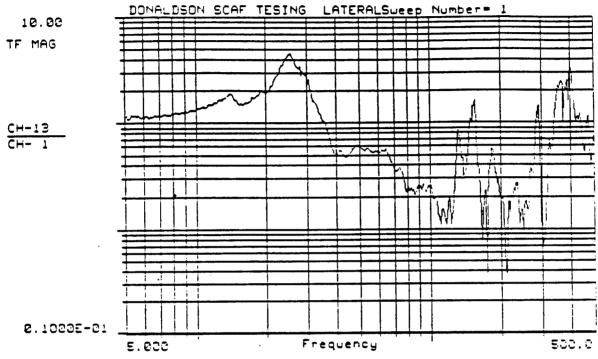
3/14/85 CH-13: LINE #12 LOC #13 BONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
•		Tape Chn. 13
Resp. Loc	#13	Footage
Resp. Accel	BA37	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



CH-13: LINE #12 LOC #13

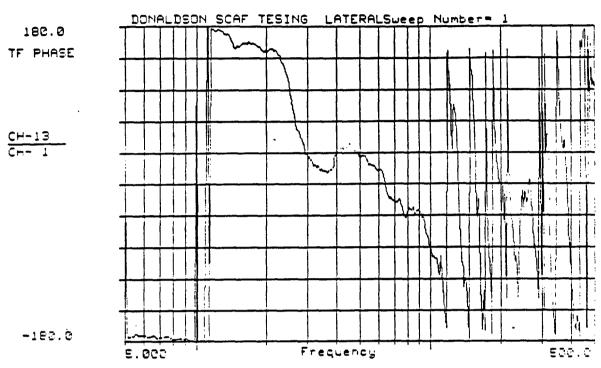
3/14/89 CH- 1: CONTROL DONALDSON SCAF TESTING

E

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 13
Resp. Loc	#13	Footage
Resp. Accel	BA37	Filename DL1:Don1.swp
Test Temp	COM	Operator A. KENNY

nonaldson Air Filter (SCAF)



CH-13: LINE #12 Loc #13

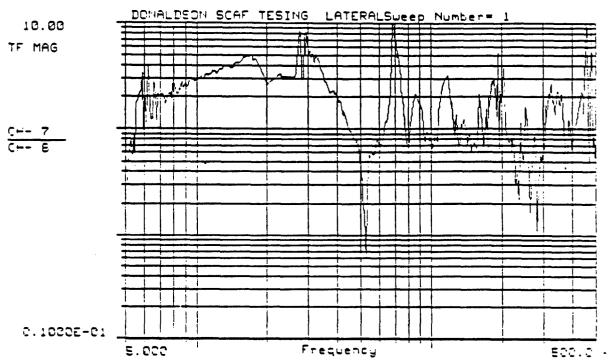
3/14/89 CH- 1: CONTROL DONALDSON SCAF TESTING

E

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Lateral	Unfiltered
·		Tape Chn. 13
Resp. Loc	#13	Footage
Resp. Accel	BA37	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1: BON1 CH- 7: LINE +8 Loc +1

3/14/89 CH- E: LINE #7 Loc #12

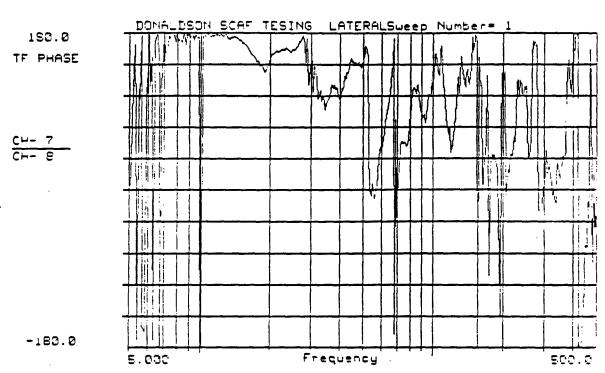
DONALDSON SCAF TESTING

#### Ε

### -Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
		Tape Chn. 7/8
Resp. Loc	#11 vs. #12	Footage
Resp. Accel	BF83/BF82	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DE1: DEN1 CH- 7: LINE #5 406 # 11

3/14/89 CH- E: LINE #7 Loc #12

DOMALDSON SCAF TESTING

Ε

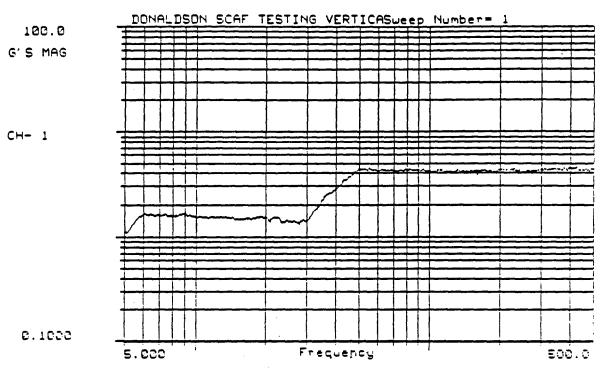
# Honeywell Hopkins Sine Vibration

OEXH #	30,299D	Test Date 14-MAR-89
Input Axis	Lateral	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
Resp. Loc	#44 #49	Tape Chn. 7/8
	#11 vs. #12	Footage
Resp. Accel	BF83/BF82	Filename DL1:Don1.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

### APPENDIX B VERTICAL AXIS INPUT

15-MARCH-1989 Gear Box Removed 4-strut Configuration



PL1:DDN2

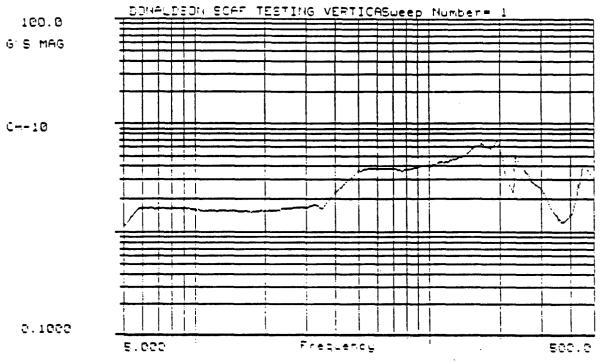
3/15/83 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	vertical	Unfiltered
		Tape Chn. 1
Resp. Loc	CONTROL	Footage
Resp. Accel	394	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



EL1:DDM2

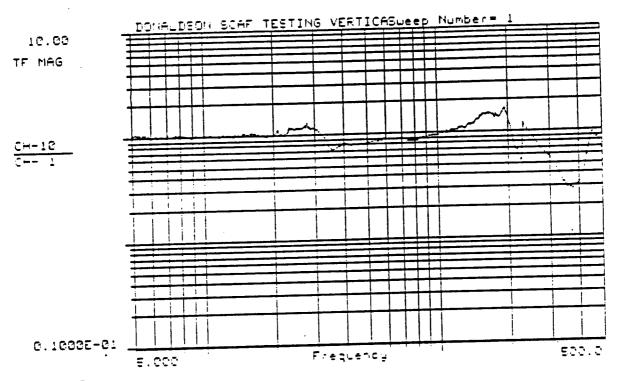
BY15/89 CH-10: LOC. #2 DOMALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 45-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
		Tape Chn. 10
Resp. Loc	#2	Footage
Resp. Accel	BK70	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



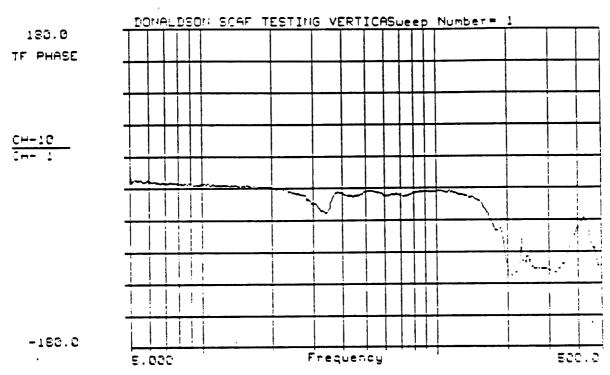
CH-10: LDI. #1 3/15/89 CH-1: CONTEDL DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
		Tape Chn. 10
Resp. Loc	#2	Footage
Resp. Accel	BK70	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



IL1:DON2

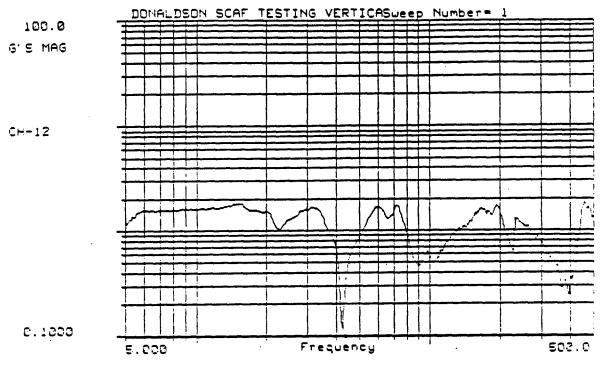
CH-10: LDC. #2 3/15/65 CH- 1: COMTROL DONALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
_		Tape Chn. 10
Resp. Loc	#2	Footage
Resp. Accel	<b>B</b> K70	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DCN2

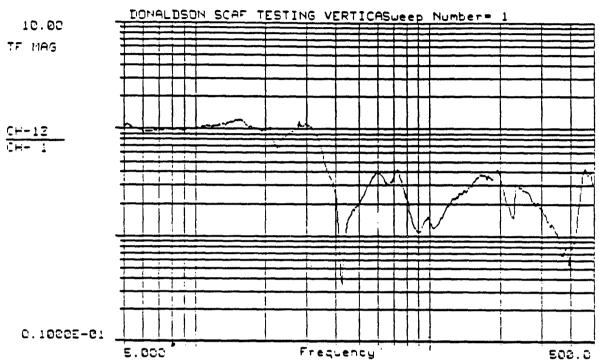
3/15/89 CH-12: LOC. #4 DONALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
·		Tape Chn. 12
Resp. Loc	#4	Footage
Resp. Accel	KF49	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



CH-12: LDC. #4

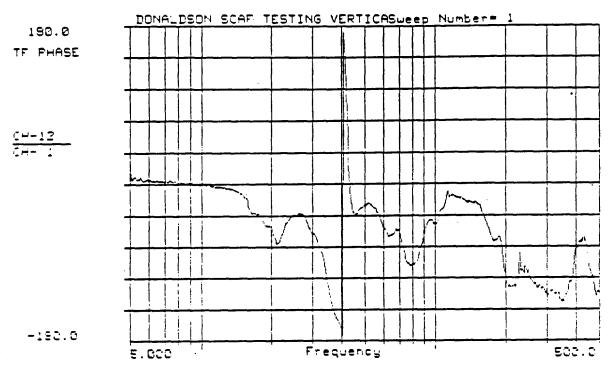
3/15/89 CH- 1: CONTROL BUHALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
		Tape Chn. 12
Resp. Loc	#4	Footage
Resp. Accel	KF49	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



PL1:DDN2 CH-12: LDC. #4 BX1EXES CH- 1: CONTROL

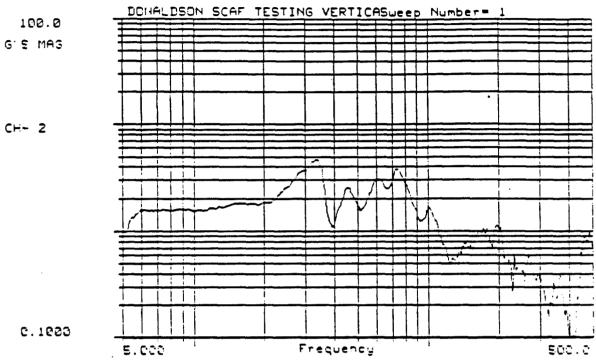
DONALDSON SCAF TESTING

#### Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
		Tape Chn. 12
Resp. Loc	#4	Footage
Resp. Accel	KF49	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DDN2

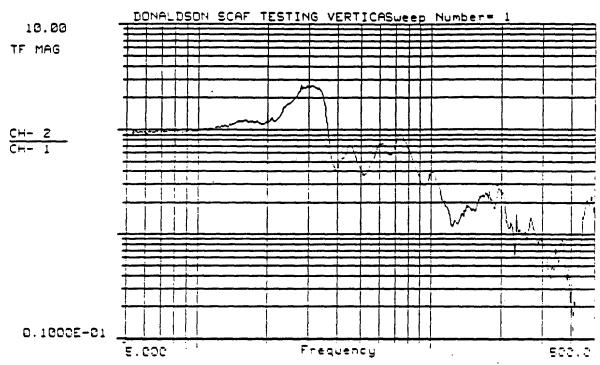
3/15/83 CH+ 2: LDC #5 DGHALDSON SCAF TESTING

E

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
·		Tape Chn. 2
Resp. Loc	#6	Footage
Resp. Accel	BK69	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



IL1:DON2

CH- 2: LGC \*6

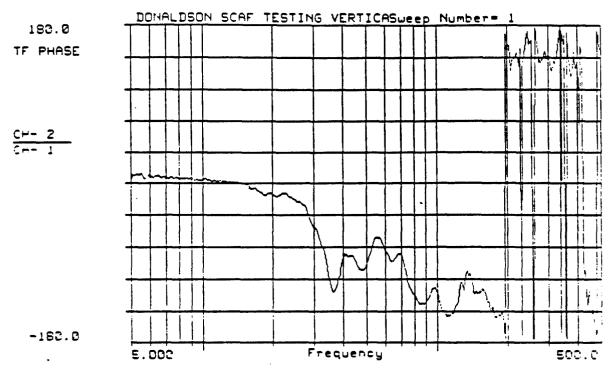
3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
·		Tape Chn. 2
Resp. Loc	#6、	Footage
Resp. Accel	BK69	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



CH- 2: LOC #5

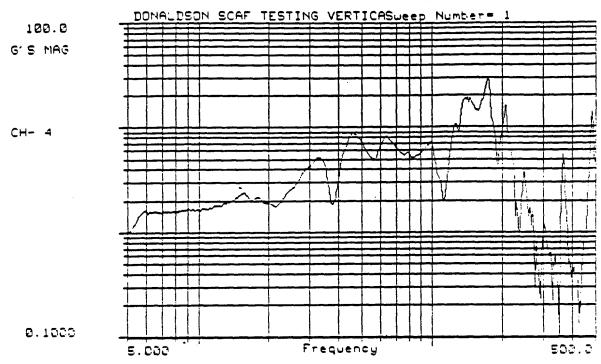
3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
		Tape Chn. 2
Resp. Loc	#6	Footage
Resp. Accel	BK69	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



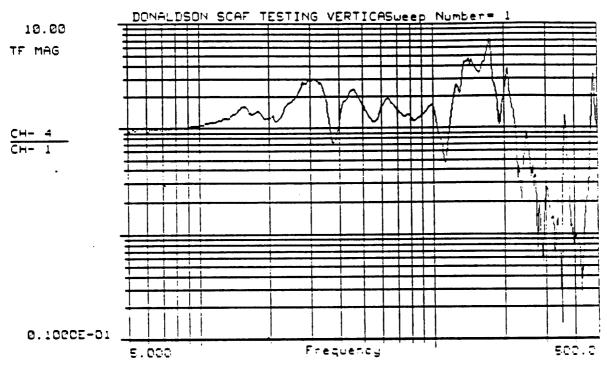
3/15/89 CH- 4: LOC. #8
DONALDSON SCAF TESTING

E

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
		Tape Chn. 4
Resp. Loc	#8	Footage
Resp. Accel	BK54	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DOM2

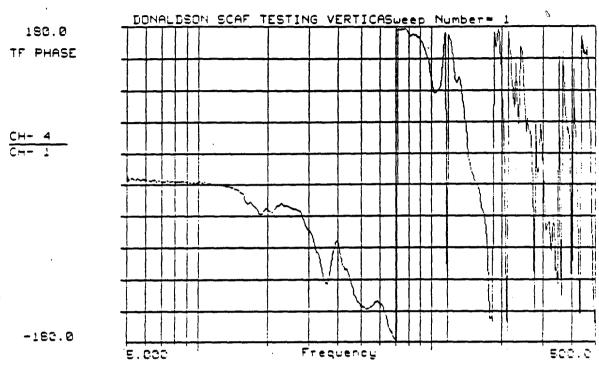
CH- 4: LDC. \*8
3/15/89 CH- 1: CONTROL
DONALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
•		Tape Chn. 4
Resp. Loc	#8	Footage
Resp. Accel	Brt	Filename DL1:Don2.swp
Test tomm	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



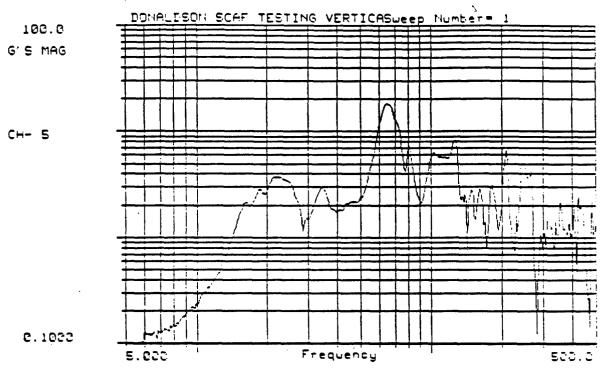
PL1:DOM2
CH- 4: LOC. #8
3/15/89 CH- 1: CONTROL
DOMALDSON SCAF TESTING

E

#### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Vertical	Unfiltered
·		Tape Chn. 4
Resp. Loc	#8 .	Footage
Resp. Accel	BK54	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DDN2

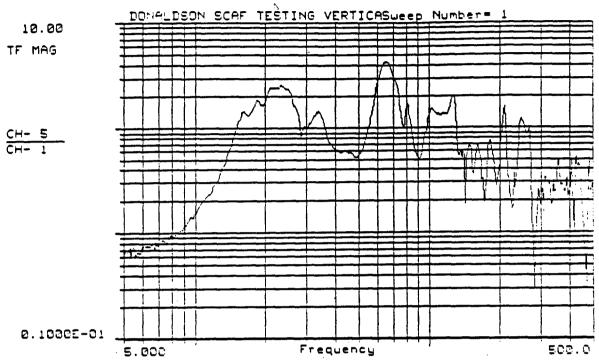
3/15/89 CH- 5: LCI. #5 DONALDSCH SCAF TESTING

E

# Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 5
Resp. Loc	#9	Footage
Resp. Accel	BK63	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



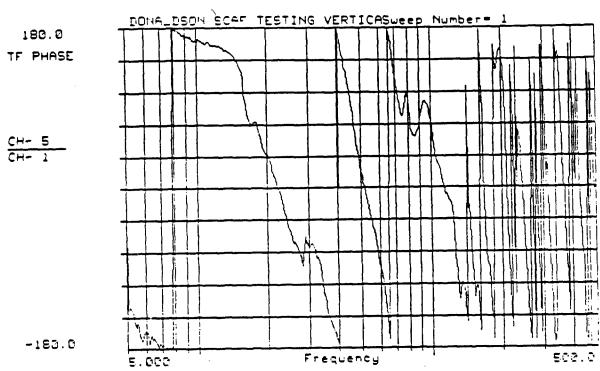
CH- 5: LDC. #9 3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 5
Resp. Loc	<b>#9</b>	Footage
Resp. Accel	BK63	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DDH2
CH- E: LDC. #5
3/15/89 CH- 1: CONTROL
DONALDSON SCAF TESTING

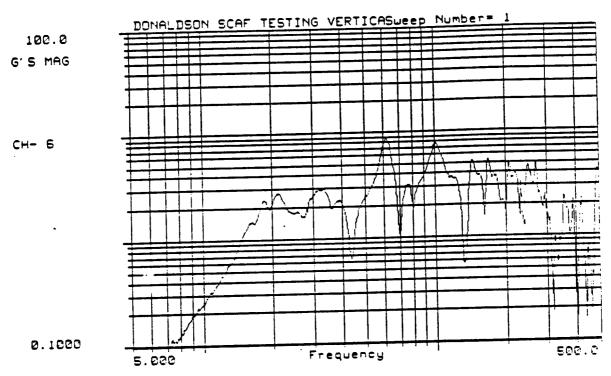
Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial	Unfiltered
•		Tape Chn. 5
Resp. Loc	<b>#</b> 9	Footage
Resp. Accel	BK63	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Transfer Function Phase



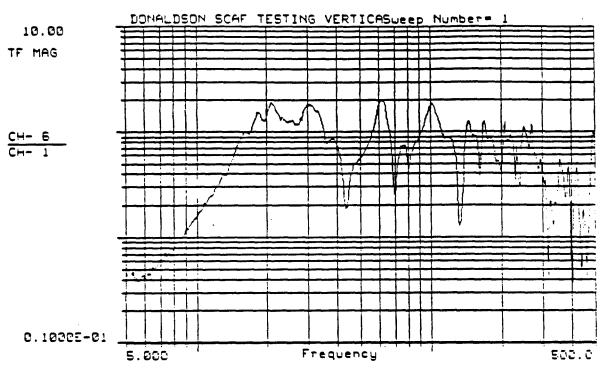
DL1:DCN2

3/15/89 CH- E: LOC. #10 DOMALDSON SCAF TESTING

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 13-114"
Input Axis Resp. Axis	Vertical Axial	Filtered YES Unfiltered Tape Chn. 6
Resp. Loc Resp. Accel Tast Temp	#10 BK65 Room	Footage Filename DL1:Don2.swp Operator A. KENNY

Donaldson Air Filter (SCAF)



CH- 6: LOC. #10

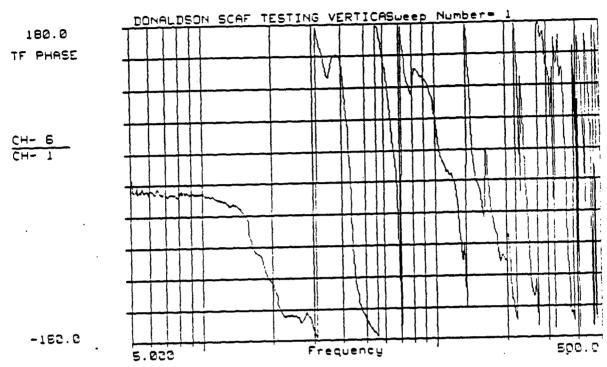
3/15/69 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial	Unfiltered
•		Tape Chn. 6
Resp. Loc	#10	Footage
Resp. Accel	8K65	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



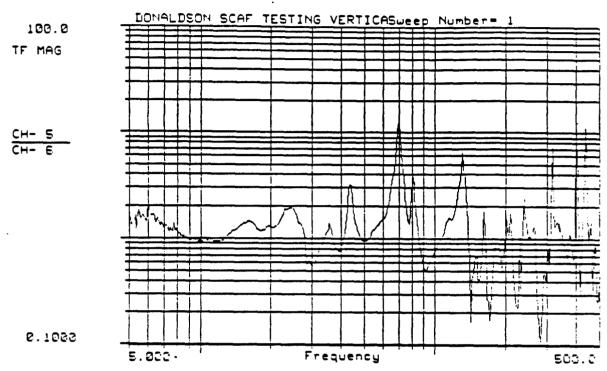
DL1:DON2 CH- E: LDC. #10 3/15/89 CH- 1: CDNTRDL DDNALDSON SCAF TESTING

E

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial	Unfiltered
•		Tape Chn. 6
Resp. Loc	#10	Footage
Resp. Accel	BK65	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



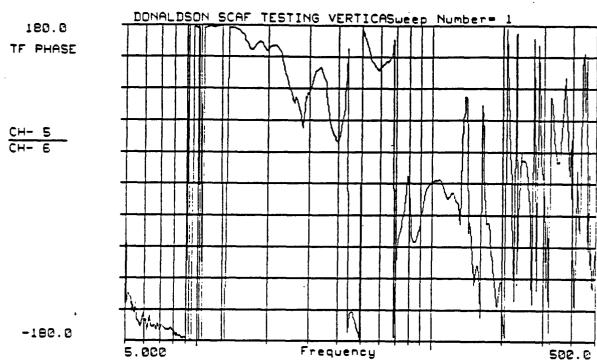
CH- E: LDC. #5
3/15/89 CH- 6: LDC. #10
DDNALDSON SCAF TESTING

E

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
		Tape Chn. 5/6
Resp. Loc	<b>#9 VS. #10</b>	Footage
Resp. Accel	BK63/BK65	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



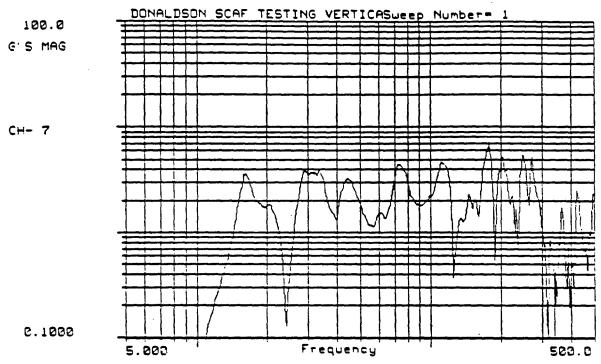
DL1:DON2 CH+ 5: LOC. #9 3/15/E3 CH- E: LOC. #10 DONALDSON SCAF TESTING

E

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
•		Tape Chn. 5/6
Resp. Loc	#9 VS. #10	Footage
Resp. Accel	BK63/BK65	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF) Transfer Function Phase



DL1: DDN2

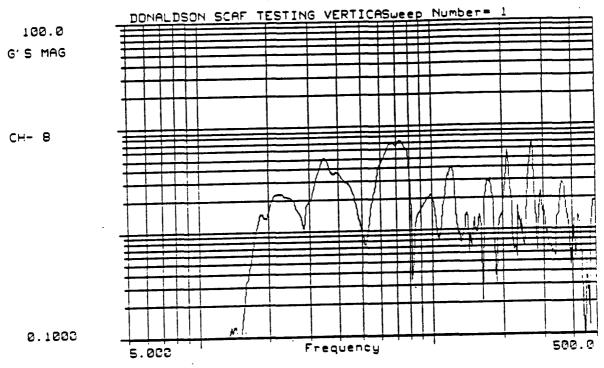
3/15/89 CH- 7: LOC. #11 DUNALDSON SCAF TESTING

п	

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 7
Resp. Loc	#11	Footage
Resp. Accel	BF83	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



3/15/89 CH- 8: LDC. #12 DONALDSON SCAF TESTING

E

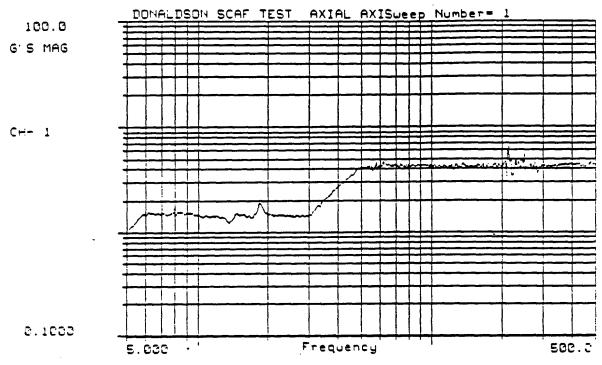
# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Vertical	Filtered YES
Resp. Axis	Axial	Unfiltered
Keepe III.		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename DL1:Don2.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

#### APPENDIX C AXIAL AXIS INPUT

15-MARCH-1989 Gear Box Removed 4-strut Configuration



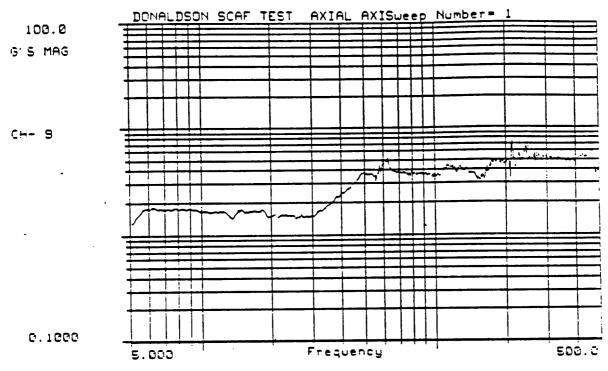
BUIENES CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	axial	Unfiltered
		Tape Chn. 1
Resp. Loc	CONTROL	Footage
Resp. Accel	394	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



IL1:DON3

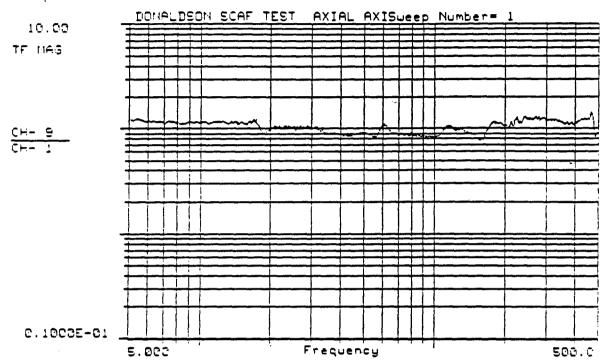
3/15/89 CH- 9: LOC. #1 DONALDSON SCAF TESTING

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 9
Resp. Loc	#1	Footage
Resp. Accel	TA63	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



DL1:DDN3 CH~ S: LDC. #1

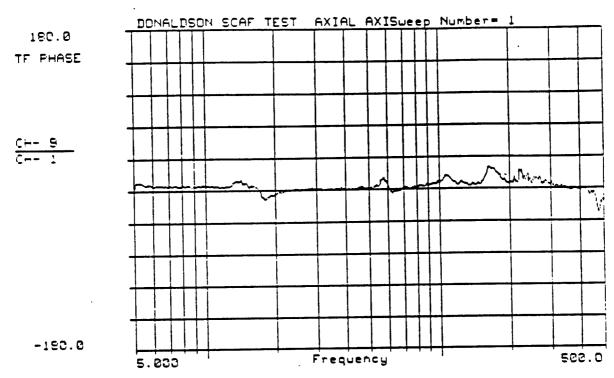
BY15/69 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 9
Resp. Loc	#1	Footage
Resp. Accel	TA63	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



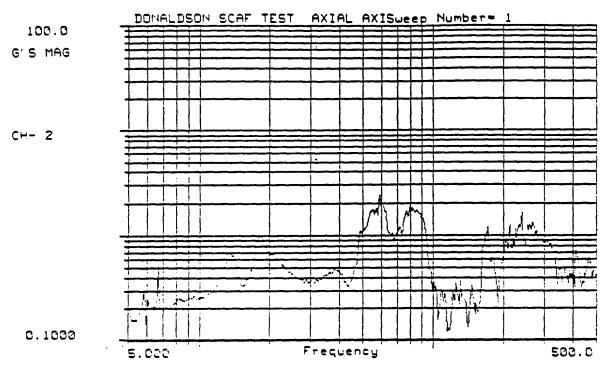
IL1: PDN3 CH- 9: LOC. #1 3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 9
Resp. Loc	₩1	Footage
Resp. Accel	TA63	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DOM3

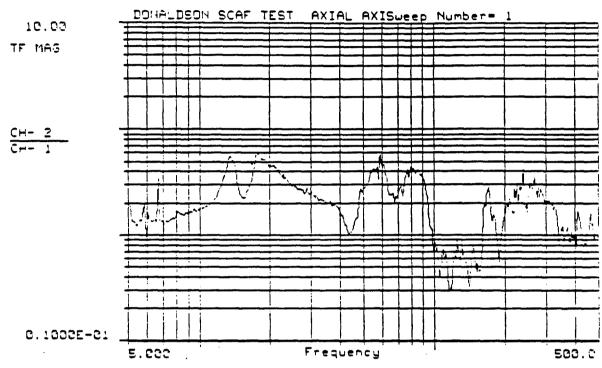
3/15/63 CH- 2: LDC. #6 DONALDSON SCAF TESTING

Ε

#### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Vertical	Unfiltered
•		Tape Chn. 2
Resp. Loc	#6	Footage
Resp. Accel	BK69	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



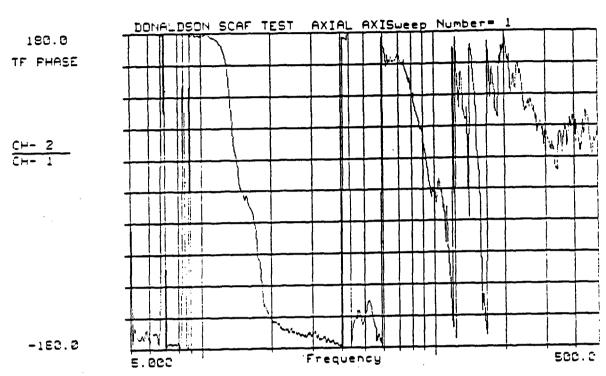
EL1:DONS
CH- Z: LOC. #5
3/15/69 CH- 1: CONTROL
DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Vertical	Unfiltered
•		Tape Chn. 2
Resp. Loc	#6	Footage
Resp. Accel	BK69	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



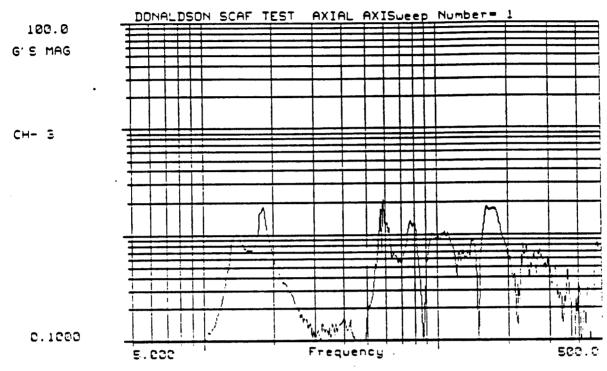
IL1:DON3 CH- 2: LOC. #6 3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,2990	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Vertical	Unfiltered
•		Tape Chn. 2
Resp. Loc	#6	Footage
Resp. Accel	BK69	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



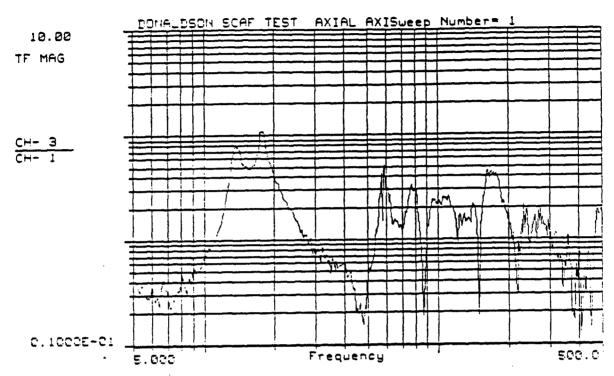
3/15/89 CH- 3: LDC. #7 DOMALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

UEXM #	30,299D	Test Date 15-MAR-89
Input Axis Resp. Axis	Axial Lateral	Filtered YES Unfiltered
Resp. Loc Resp. Accel Test Temp	#7 BK39 Room	Tape Chn. 3 Footage Filename DL1:Don3.swp Operator A. KENNY

Donaldson Air Filter (SCAF)



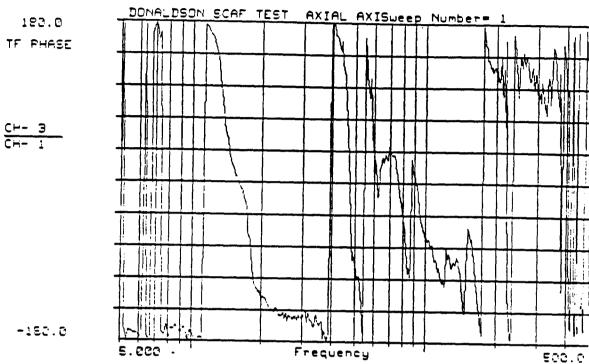
DL1:DON3
CH- 3: LDC. #7
3/15/89 CH- 1: CDNTROL
DOMALDSON SCAF TESTING

E

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Lateral	Unfiltered
•		Tape Chn. 3
Resp. Loc	#7	Footage
Resp. Accel	BK39	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DON3 . CH- 3: LOC. #7

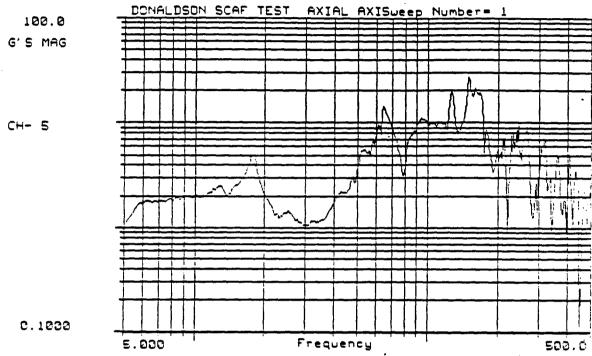
EXIEXES CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Lateral	Unfiltered
		Tape Chn. 3
Resp. Loc	#7	Footage
Resp. Accel	BK39	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF) Transfer Function Phase



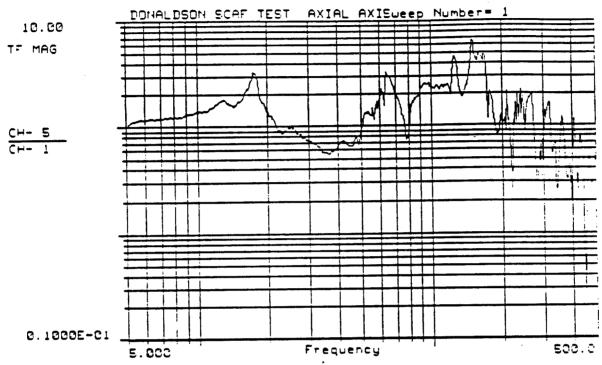
3/19/83 CH- 5: LOC. #9 DONALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 5
Resp. Loc	#9	Footage
Resp. Accel	BK63	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



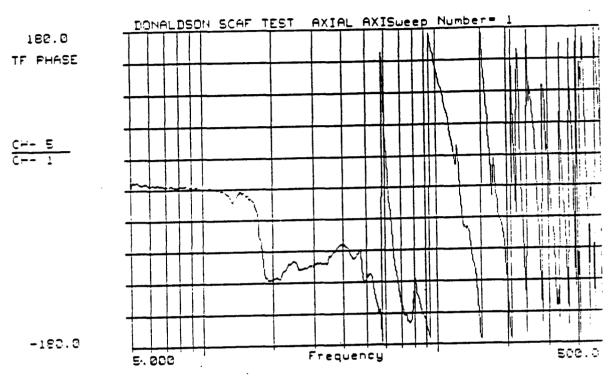
CH- 5: LOC. \*S 3/15/83 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 5
Resp. Loc	<b>#</b> 9	Footage
Resp. Accel	BK63	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



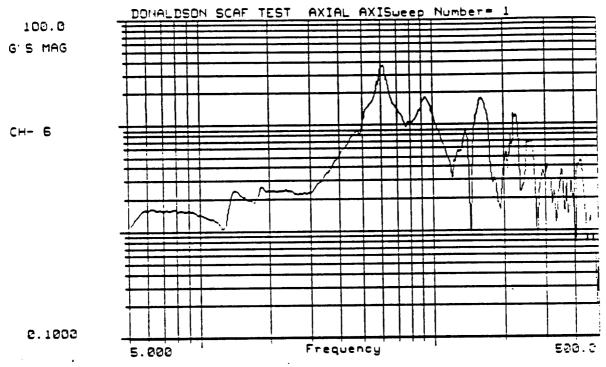
CH- 5: LDC. #3
3/15/85 CH- 1: CONTROL
DONALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
•		Tape Chn. 5
Resp. Loc	<b>#</b> 9	Footage
Resp. Accel	BK63	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



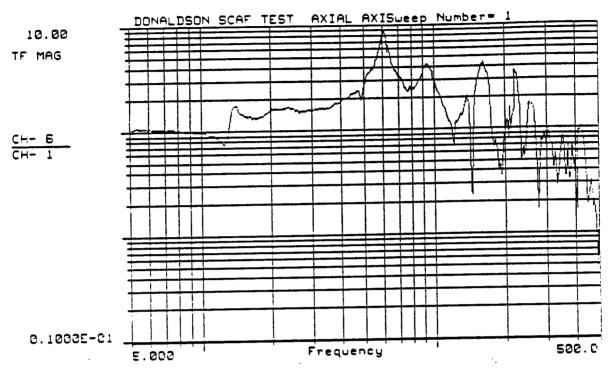
3/15/89 CH- E: LOC. #10 DOMALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

30,299D	Test Date 15-MAR-89
Axial	Filtered YES
Axial	Unfiltered
	Tape Chn. 6
#10	Footage
BK65	Filename DL1:Don3.swp
ROOM	Operator A. KENNY
	Axial #10 BK65

Donaldson Air Filter (SCAF)



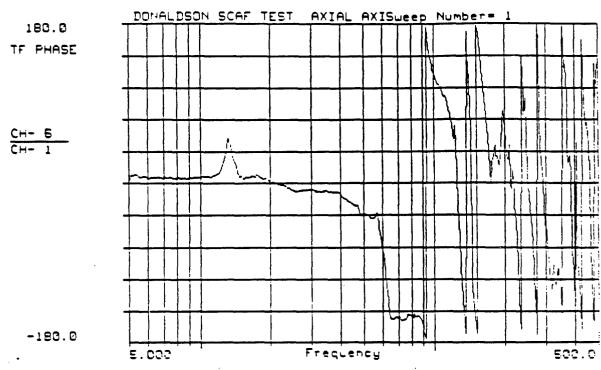
DL1:DON3
CH- E: LOC. #10
3/15/89 CH- 1: CONTROL
DONALDSON SCAF TESTING

E

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 6
Resp. Loc	#10	Footage
Resp. Accel	BK65	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



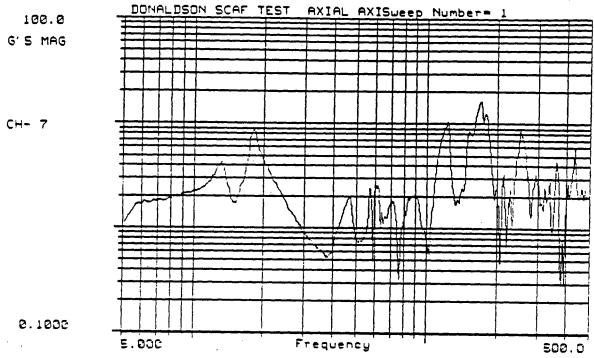
CH- E: LOC. #10 3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
•		Tape Chn. 6
Resp. Loc	#10	Footage
Resp. Accel	8K65	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



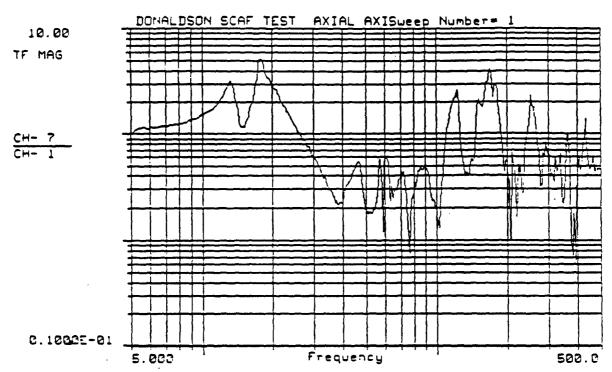
DL1:DDN3

3/15/89 CH- 7: LDC. #11 DONALDSON SCAF TESTING

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
•		Tape Chn. 7
Resp. Loc	#11	Footage
Resp. Accel	BF83	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



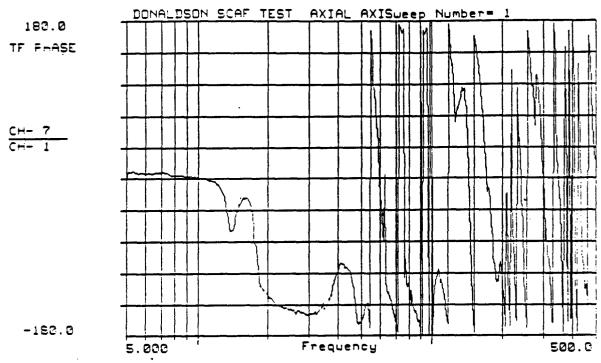
CH- 7: LDC. #11
3/15/89 CH- 1: CDNTROL
DONALDSON SCAF TESTING

1	F	•
	۰	

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis Resp. Axis	Axial Axial	Filtered YES Unfiltered
Kash. www.	,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	Tape Chn. 7
Resp. Loc	#11 BF83	Footage Filename DL1:Don3.swp
Resp. Accel Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



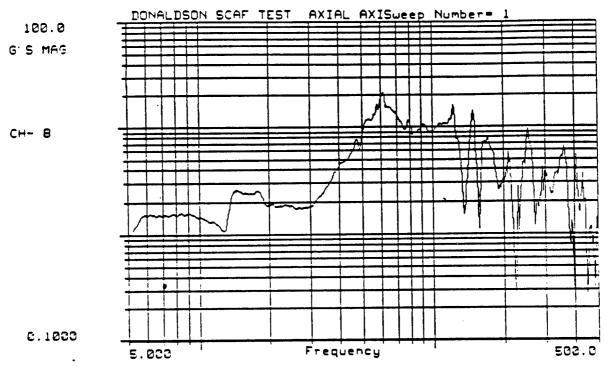
CH- 7: LDC. #11
3/15/89 CH- 1: CONTROL
DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
,		Tape Chn. 7
Resp. Loc	#11	Footage
Resp. Accel	BF83	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



3/15/89 CH- 8: LDC. #12 DONALDSON SCAF TESTING

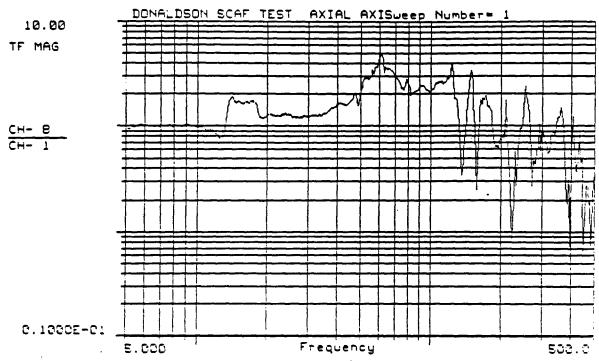
E

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis Resp. Axis	Axial Axial	Filtered YES Unfiltered
		Tape Chn. 8
Resp. Loc	<b>#</b> 12	Footage
Resp. Accel	BF82	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Magnitude Plot



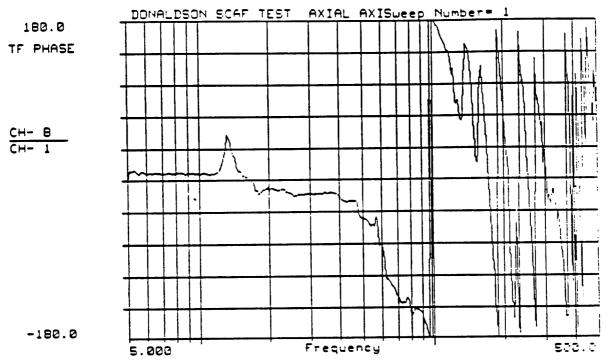
CH- 8: LDC. #12 3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

Ε

## Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



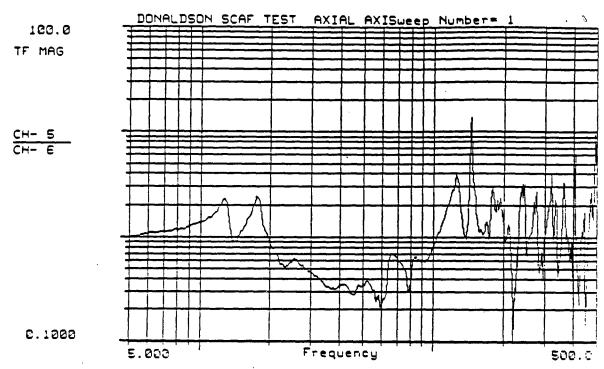
CH- 8: LDC. #12 3/15/89 CH- 1: CONTROL DONALDSON SCAF TESTING

E

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial	Unfiltered
Kespi ims		Tape Chn. 8
Resp. Loc	#12	Footage
Resp. Accel	BF82	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



CH- 5: LDC. #9

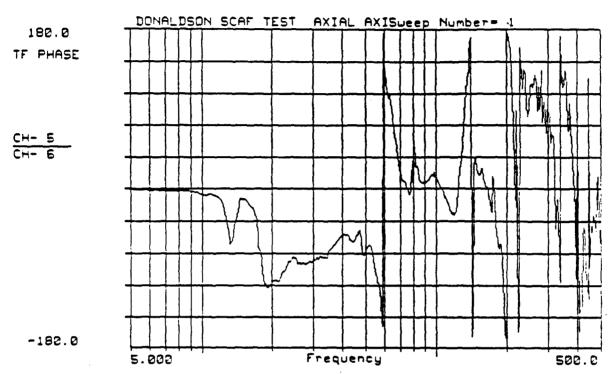
3/15/89 CH- E: LOC. #10 DOMALDSON SCAF TESTING

Ε

### Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
		Tape Chn. 5/6
Resp. Loc	<b>#</b> 9 VS. #10	Footage
Resp. Accel	BK43/BK45	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



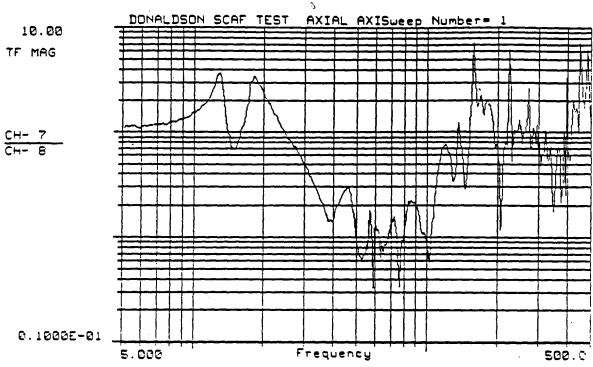
CH- 5: LOC. #9
3/15/89 CH- 6: LOC. #10
DONALDSON SCAF TESTING

E

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
Resp. Loc	#9 VS. #10	Tape Chn. 3/6 Footage
Resp. Accel	BK63/BK65	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)



DL1:DON3
CH- 7: LOC. #11
3/15/89 CH- 8: LOC. #12
DONALDSON SCAF TESTING

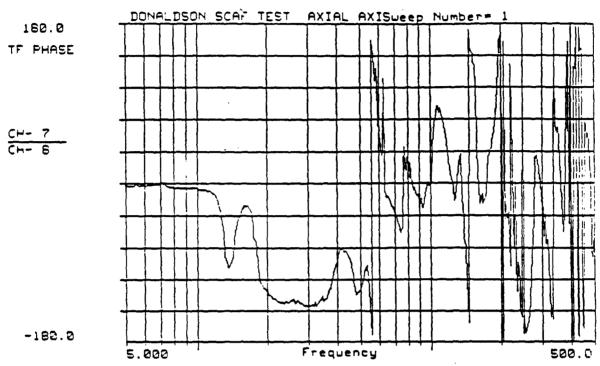
Ε

## Honeywell Hopkins Sine Vibration

OEXM *	30,299D	Test Date 15-MAR-89
Input Axis	Axial	Filtered YES
Resp. Axis	Axial/Axial	Unfiltered
•		Tape Chn. 7/8
Resp. Loc	#11 VS. #12	Footage
Resp. Accel	BF83/BF82	Filename DL1:Don3.swp
Test Temp	ROOM	Operator A. KENNY

Donaldson Air Filter (SCAF)

Transfer Function Magnitude



DL1:DDN3

CH- 7: LDC. #11
3/15/89 CH- 8: LDC. #12
DONALDSON SCAF TESTING

Ε

# Honeywell Hopkins Sine Vibration

OEXM #	30,299D	Test Date 15-MAR-89
Input Axis Resp. Axis	Axial Axial/Axial	Filtered YES Unfiltered
Resp. Loc Resp. Accel Test Temp	#11 VS. #12 BF83/BF82 ROOM	Tape Chn. 7/8 Footage Filename DL1:Don3.swp Operator A. KENNY

Donaldson Air Filter (SÇAF)

APPENDIX B

SHOCK TESTING

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**ENGINEERING AND TEST DIVISION** CHURCH STREET, BOHEMIA, LONG ISLAND, NEW YORK 11716 (516) 589-6300

TEST REPORT NO:

DTB04R89-0595

DATTON T. BROWN, INC. JOB NO: 406744-00-000

CUSTOMER:

DONALDSON COMPANY, INC.

P.O. BOX 1299

MINNEAPOLIS, MINNESOTA 55440-1299

SUBJECT:

SHOCK TEST PROGRAM PERFORMED ON ONE

SCAF, PART NUMBER 119385, SERIAL

NUMBER 82E018

ATTENTION: MR. HARRY CAMPLIN, MS NO. 222

THIS REPORT CONTAINS: FIVE PAGES AND TWO ENCLOSURES

PREPARED BY	G. HYLAND St. Hyland
Test Engineer	G. HYLAND G. Hyland
TEST OPERATIONS MANAGER	V. VIRGILIO
DATE	12 MAY 1989

THE DATA CONTAINED IN THIS REPORT WAS OBTAINED BY TESTING IN COMPLIANCE WITH THE APPLICABLE TEST SPECIFICATION AS NOTED

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# TABLE OF CONTENTS

Subject	Paragraph	Page Number
Abstract	1.0	2
References	2.0	3
Administrative Information	3.0	4
Test Program Outline	4.0	5
Enclosures		
(1) Shock Test and Results		91 Pages
(2) Photographs		7 Photos



#### 1.0 ABSTRACT

This test report details the results of the shock test program conducted on one SCAF, part number 119385, serial number 82E018 under reference (a) to the requirements of reference (c).

Results of the test are detailed in the following text.

The test item was nonoperating during testing.

Test data pertinent to this program will remain on file at Dayton T. Brown, Inc. for 90 days.



### 2.0 REFERENCES

(a) Customer Purchase Order Number: D78000943

(b) Dayton T. Brown, Inc. Job Number: 406744-00-000

(c) Test Specification: Dayton T. Brown, Inc. Quote ISL-89-0535



#### 3.0 ADMINISTRATIVE INFORMATION

Customer: Donaldson Company, Inc.

P.O. Box 1299

Minneapolis, Minnesota · 55440-1299

Test Item Description: SCAF

Quantity Received: One

Part Number: 119385

Serial Number: 82E018

Date Received: 17 April 1989

Date Shipped: 24 April 1989

### Customer Representative Present During Portions of Test:

#### Name

#### Affiliation

Mr. Harry Camplin Mr. Scott Nisbett Mr. Anthony Ligato Donaldson Company, Inc. Textron Lycoming Textron Lycoming



# 4.0 TEST PROGRAM OUTLINE

Test

Test Item

Description

Results

Shock

SCAF, Part Number 119385, Serial Number 82E018 Refer to Enclosure 1



Enclosure 1
Shock Test and Results



#### TEST REQUIREMENT

The shock test shall be conducted in accordance with reference (c).

#### TEST RESULTS

A pretest visual inspection of the test item revealed no anomalies.

All testing was performed in accordance with the referenced specification.

Refer to the shock test summary for tabulated results.

The test item did not complete all phases of testing.

A post test visual inspection of the test item revealed a crack in the turbine base and some small cracks on the SCAF.

Test Item: SCAF

Part Number: 119385

Sorial Number: 82E018

Record	•		Rogu	Roquired	Act	Actual	Graph Page	
Number	Axis	Test Condition	₩		₩	s	No. (Enc. 1)	KORBIES
4/18/89								
-	- Vert.	Reste Shock	40	18.0	19.5	15.5	*	Low Level Cal.
7	- Vert.		40	18.0	15.0	17.0	₩,	Low Level Cal.
€	- Vert.		40	18.0	28.0	16.0	9	Low Level Cal.
*	- Vert.	Basic Shock	40	18.0	22.0	18.0	9	Low Level Cal.
80	- Vert.	Basic Shook	4	18.0	24.0	18.0	9	See Note 1, Low Level Ca
•	- Vort.	Basic Shock	<b>7</b>	18.0	34.0	18.5	7	Low
_	- Vert.	Basic Shock	<b>7</b>	18.0	36.0	17.6	•••	Good
•	- Vert.	Basic Shock	40	18.0	39.0	17.5	9 - 11	Good
•	- Vort.	Basio Shock	<b>4</b>	18.0	39.4	16.8	12	Good
10	- Vort.	Cunfire Shook	58	2.5	85.0	2.4	13	High g
11	- Vert.	Gunfire Shook	<b>3</b> 8	2.5	40.0	2.7	14	Low &
12	- Vert.	Ganfire Shock	55	2.5	59.8	2.5	15	Good
13	- Vort.	Gunfire Shock	55	2.5	55.5	2.5	16 - 20	Good
14	- Vort.	Operational Shock	58	(2.0	52.5	2.1	21	Good
15	- Vert.	Operational Shock	58	(2.0	50.5	2.1	22	Good
16	- Vert.	Operational Shock	58	(2.0	61.0	2.0	23 - 27	High &
11	- Vert.	Ballistic Shock	200	(2.0	100.0	1.5	28	Low Level Cal.
18	- Vort.	Ballistic Shock	200	(2.0	183.0	1.0	29	Good
19	- Vert.	Bailistic Shook	200	<2.0	190.0	1.1	30	Good
20	- Vort.	Ballistic Shock	200	<2.0	193.0	1.1	31 - 35	Good

SHOCK TEST SUMMARY (Continued)

Part Number: 119385

Test Item: SCAF

Serial Number: 82E018

Number Aris	Test Condition	Rogu	Required	Act	Actual	Graph Page	•
		4	i	4		NO. (Enc. 1)	Komarks
4/19/89							
+ Vert.	Basto Shook	9	18.0	40.0	17.7	y	Good
+ Vort.	Basic Shock	07	18.0	36.4	18.0	36 - 40	<b>1</b> 000
+ Vert.	Besto Shook	04	18.0	36.2	17.0	41	Good
+ Vert.	Gunfire Shock	55	2.5	0.99	3.6	4.2	9009
+ Vert.	Gunfire Shock	55	2.5	45.2	4.0	42	Low
+ Vort.	Gunfire Shock	55	2.5	41.4	7. 4	7	Low .
+ Vort.	Gunfire Shook	53	2.5	56.0	3.9	43 - 47	e poor
+ Vort.	Gunflre Shook	55	2.5	0.99	<b>60</b>	73	good
+ Vort.	Operational Shook	55	(2.0	58.5	2.0	48 - 52	Good
+ Vert.	Operational Shock	55	(2.0	70.0	2.0	784	9000
+ Vort.	Operational Shook	55	(2.0	57.5	1.7	53. 54	Good
+ Vert.	Ballistic Shock	200	(2.0	176.0	1.3	55	
+ Vert.	Bailistic Shook	200	(2.0	236.0	1.2	95	, poor
+ Vort.	Ballistic Shock	200	42.0	250.0	1,3	1y - 25	Good
+ Vort.	Ballistic Shook	200	<b>42.0</b>	220.0	· •	7, 79	9000
+ Long.	Basic Shock	40	18.0	43.0	16.5		900
+ Long.	Basic Shock	0	18.0	39.0	16.0	63 – 67	5005

SHOCK TEST SUMMARY (Continued)

Test Item: SCAF

Part Number: 119385

Sorial Number: 82E018

Record			Regu	Required	Actual	I an	Graph Page		
Numbor	Az is	Test Condition	₩	9	₩	<b>S</b>	No. (Enc. 1)		Romarks
4/19/89							-		
38	+ Long.	Basio Shook	04	18.0	39.0	19.0	89	Good,	See Note 2
39	+ Long.	Gunfire Shock	53	2.5	81.0	2.1	69	Cood	
40	+ Long.	Gunfire Shock	53	2.5	51.0	2.4	70 - 74	Good	
41	+ Long.	Gunfire Shock	53	2.5	50.0	1.8	7.8	Good	
42	+ Long.	Operational Shock	53	(2.0	57.5	1.8	16	Good	
43	+ Long.	Operational Shock	58	<2.0	59.0	1.8	77 - 81	Good	
7	+ Long.	Operational Shock	55	<2.0	52.5	1.8	82	Good	
45	+ Long.	Ballistic Shock	550	(2.0	250.0	1.5	83	Low g	
46	+ Long.	Ballistic Shook	550	(2.0	330.0	1.5	*	Low	
47	+ Long.	Ballistic Shook	550	(2.0	420.0	1.7	85	Low 8	, See Note 3
48	+ Long.	Ballistic Shook	550	(2.0	510.0	1.3	06 - 98	Good.	Good, See Note 4

The strut at the four o'clock position broke during this shock. It appeared that there was not enough thread engagement. It was removed and testing continued. Notes: 1.

The strut pin at the ten o'clock position bent during this shock. It was replaced and testing continued. 7

After three low level shocks, the base of the turbine was cracked. The crack was welded and testing continued.

The base of the turbine cracked again. There were some small cracks in various places on the SCAF (refer to ti photographs in enclosure 2). Testing was stopped. DAYTON ANOWN.

Test Is

Test thom SCAE

Serial Number(s) Recent Num operational M

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Lab Form D24-2

Job Number: F.C.6.7.414-00-070 Date B. April 89.

Pickup Sensitivity: 10,0 Direction: NE Co. Co.

C) Tape

M Live

Picture Location: Grant Hey . 721. 

Time. 1570.

Test Horn . SCAF Serial Number(s) Non operational M

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Job Number: 4067.44.00.-070. Direction . WE'VE MET. LAPELE Pickup Location: . ( Add 12 20 L. .

Pickup Serial Number: . ASSAFF.

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Dave 17. April. 89.

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Pichup Sensing Axis: (4)

John Number: Fich 2414 . OP. Date: 16 April . 89.

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Direction: A. C. C. Pickup Semilivity ....

Job Number: \$262.414-00.-070 Date: 1. 3. April 89. Time: . / fel !!

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89-0595 Enc 1 Pg 13

Lab Form D24-2

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Pichup Serial Number: NIBK.

Picture Location: . . (CALIF 20)

Pickup Sensing Axis: . .

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Job Number: FO62.41/4-00.-079 Dave. 18 April 89

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Direction: A Live

Date: April 89.

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89-0595 Enc 1 Pg 15

Lab Form D24:2

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Direction: N Live

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Job Number: #26249-09-030

Date: 18 April 89.

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Lab Form D24 2

Dave 18 April 89 Job Number: #26749.00.-070

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Pictiup Serial Number: UB.H.

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Pickup Sensing Ans: . . . . . . VKR. Pichup Location: . CANTROL.

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89-0595 Enc 1 Pg 17

Pickup Sensitivity: . . . . . .

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Lab Form D24-2

Pickup Sensing Axis: Pickup Location: .

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TEST SHOT NO 13 TEST SHOT NO ACTUAL READING

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Lab Form D24 2

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Lab Form D24-2

Job Number: 1906 744 Date: 189

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Lab Form D24 2

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Job Number 40 6744-00-000

19 APR 89

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Pickup Sensitivity 10.0 mypeak

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Date 19 APR 89 Job Number: 40 6744-00-000.

Direction VLKT POS NOW WEST REPORTED

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Pickup Serial Number: 682 Pickup Location: (224:7-4%) Pickup Sensing Axis: O.C. 4.7

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Job Number 40 6744-00-000.

19 APR 89 Time ... /520 Direction VLKT POT NOW SEEN WITHER Pickup Sensitivity (O.O. gpreak 文 Tage T Live Pickup Sensing Axis VC- / Pickup Location .... 772 / .... Pickup Serial Number: . . 744

89-0595 Enc 1 Pg 58

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Lab Form D24-2

Job Number: 40 6744-00-000

Date 19 APR 89

Direction: VEKT POS Nov 1257 111766

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Pickup Sensitivity (O. O. my peak

Pickup Serial Number: ( Ho.1.

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19 APR 89

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Pickup Serial Number:

Pickup Sensitivity: 10.0

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Non-operational M Operational [.] Chit

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Lab Form D24-2

Job Number: 40 6744-00-000.

19 APR 89

Direction VEKT POS NON DESCRIPTIONS

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Pickup Sensitivity 10.0

Pickup Sensing Axis: ... VC

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Serial Number(s) 82 E 018 Test Hem. SCAF

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Job Number 40 6744-00-000

Date 19 APR 89

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Serial Number(s) 82 E 018

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Date 19 APR 89 Time: /2% Pickup Sensitivity: 10.0 my prak

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89-0595 Enc 1 Pg 66

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Serial Number(s) 82 E 018

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Direction Pickup Sensitivity

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Lab Form D24 2

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Lab Form D24 2

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Lab Form D24 2

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Lab Form D24-2

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Pickup Sensitivity.

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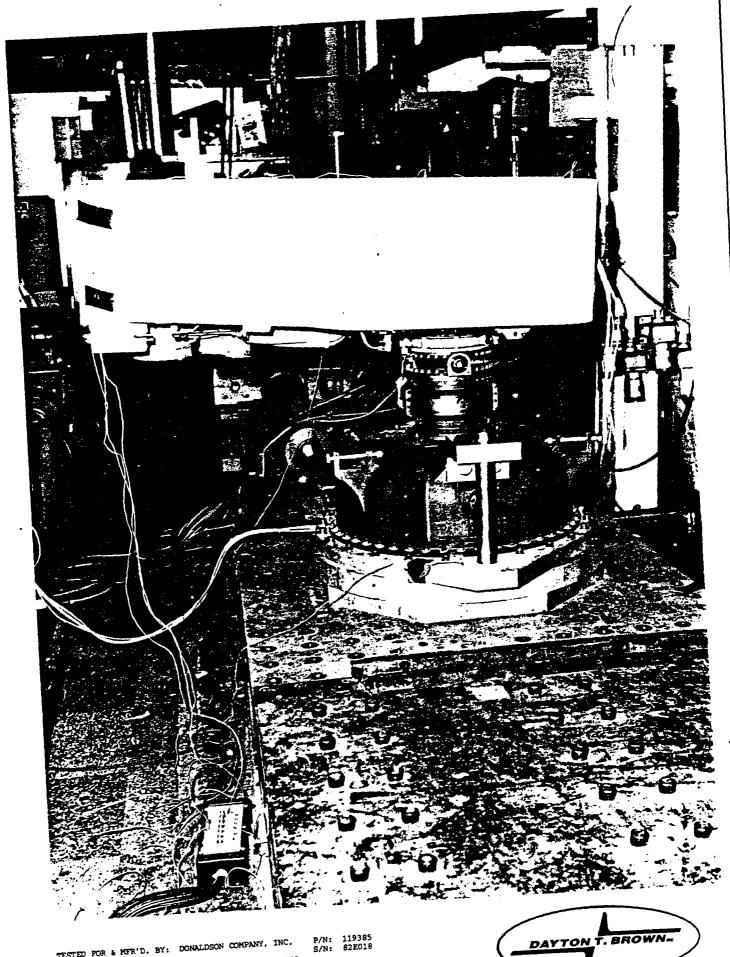
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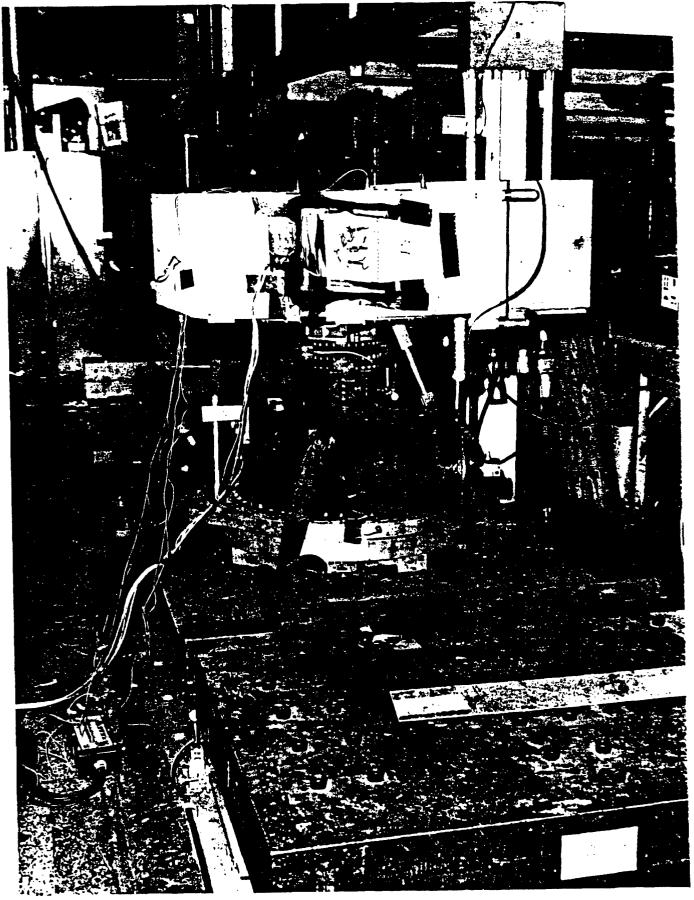
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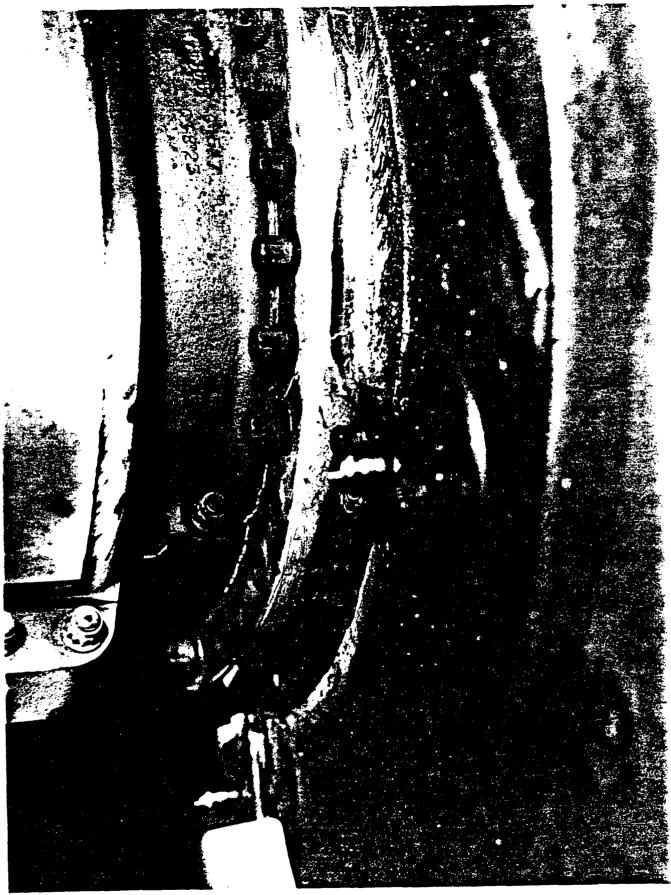
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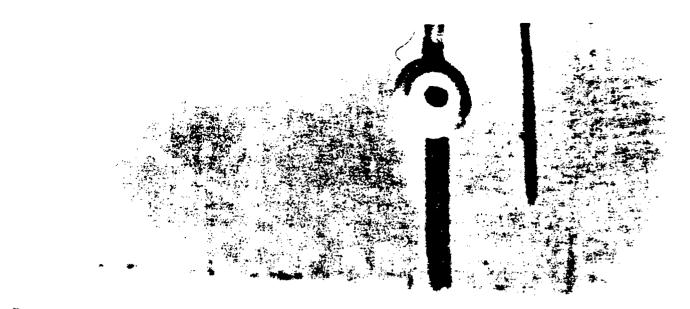
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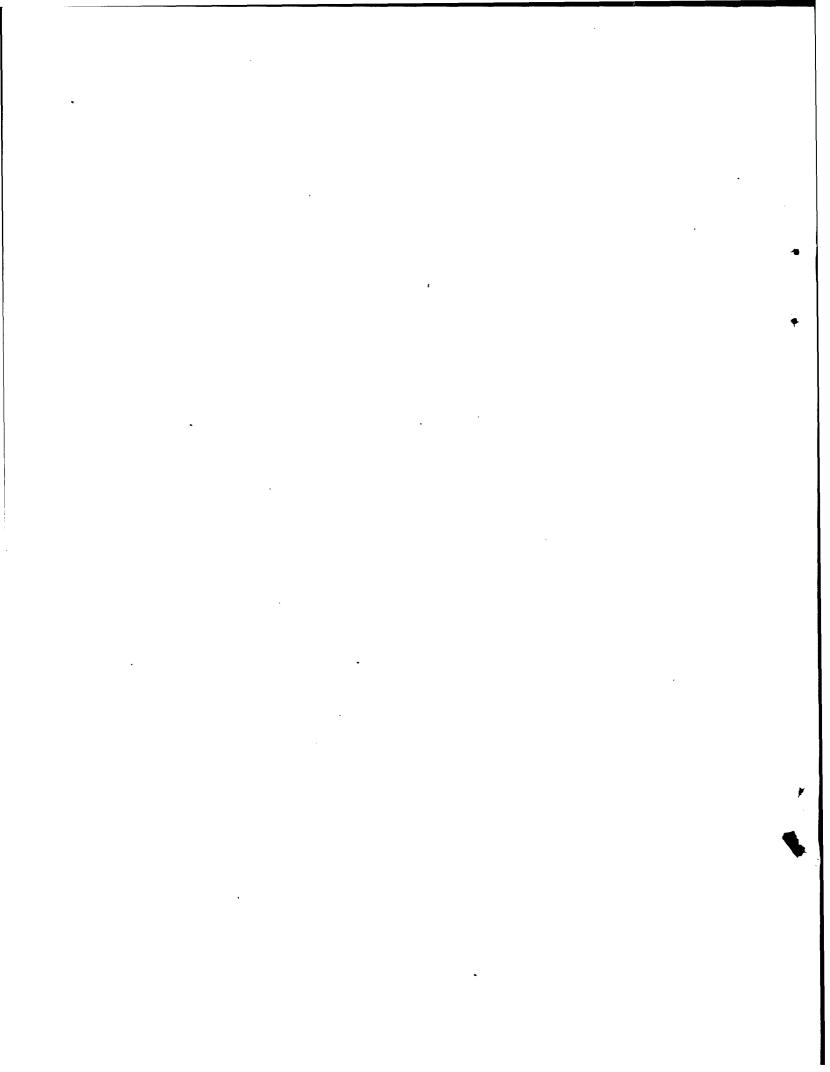


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### Appendix C

Comparison Data for M1A1 SCAF, RESCAF, and Conventional 2 Stage Air Cleaner

#### 1.0 Life Characteristics Comparisons

The data in this appendix is provided so that a comparison of dust life characteristics can be made between the conventional 2 stage air cleaner and the RESCAF, and the M1A1 SCAF system. The data provided is basic data from which the user can construct his own curves. The data provided is listed below.

#### 1.1 Vehicle Installation Pressure Loss Data

This set of curves gives the conventional 2 stage air cleaner and early phase II and phase III SCAF system data. The SCAF systems tested at Yuma Proving Ground in 1988-1989 were phase IV systems which had pressure loss characteristics 2"-4" lower than the phase III systems at 10,000 cfm due to elimination of head exchangers and enlarged filter element outlet areas. This data is with clean filters.

#### 1.2 M1A1 SCAF Performance Curves

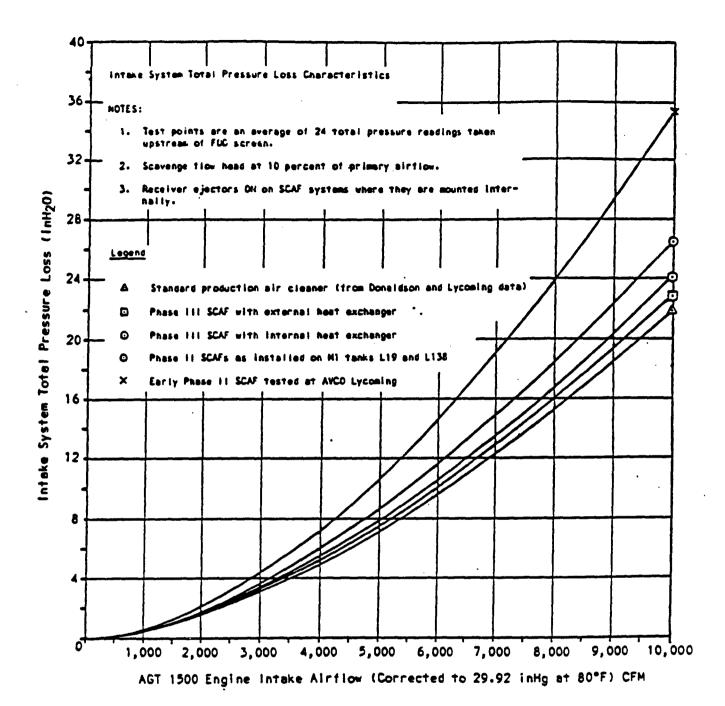
This data shows the life characteristics of the SCAF systems as they were fielded at Yuma after corrections were made to eliminate the cause of engine failures. This data shows dust loading rates.

#### 1.3 Conventional 2 Stage Life Characteristics

This data shows the life characteristics of the conventional system currently installed on the MIA1 SCAF.

1.4 Comparison of SCAF and the Conventional Air Cleaner at Yuma Proving Ground.

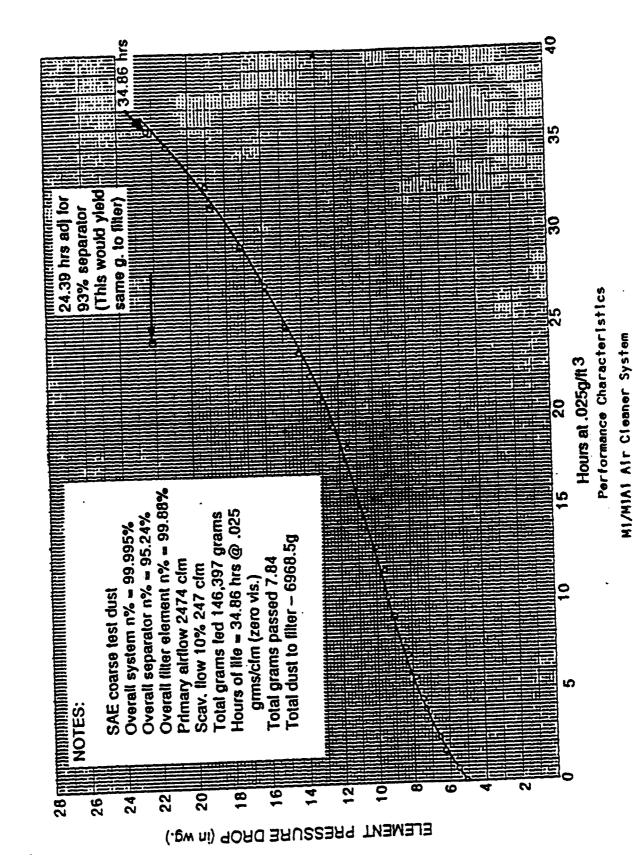
This data shows a comparison based on recent field experience at Yuma Proving Ground. It includes two tables with comparative data.



#### Phase III SCAF Vehicle Installation Loss Data

The AGT 1500 gas turbine engine used in the Ml Tank requires protection from damage caused by the ingestion of dust. An air cleaning system is therefore a requirement. The presence of air cleaning system components in the intake airstream causes a pressure loss which results in reduced power output of the engine. The greatest power loss occurs when the highest airflow is required, ie at maximum engine power. The curve shows the pressure loss characteristics of the standard air cleaner, the Phase II SCAF system, and the Phase III SCAF systems, with their variations.

When power loss is calculated, an additional loss of approximately 4.5 in. wg. must be added for the vehicle ballistic grills and the F.O.D. screen. These losses may differ between vehicles, but would be the same for either the SCAF or the standard air cleaner.



Simulated with 1 Filter

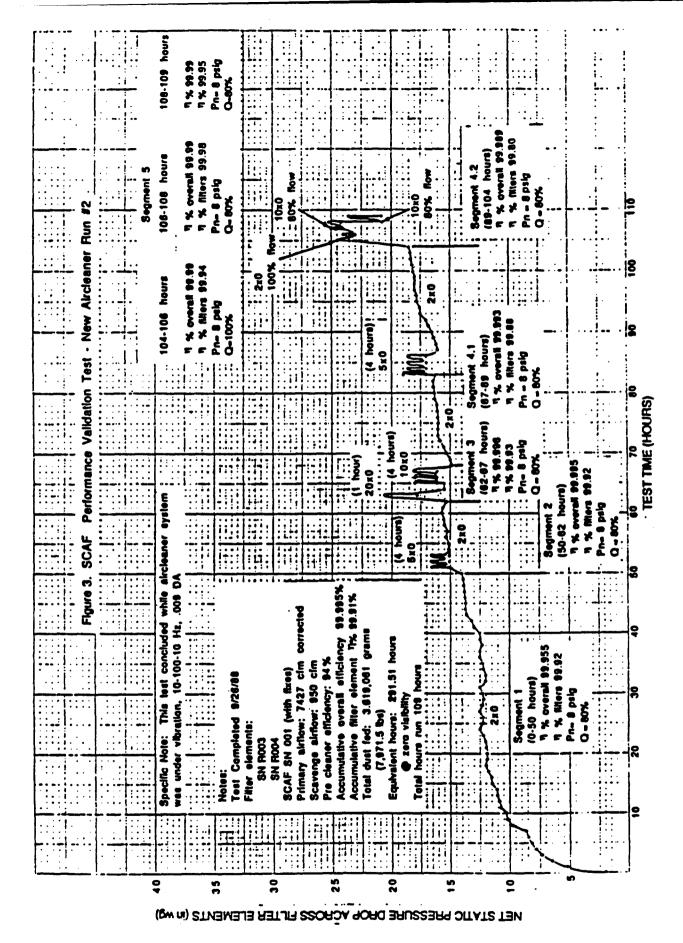


Figure 3. SCAF Performance Validation Test New Air Cleaner Run #2

Table 1. Demonstrated Filter Element Service Life in Kilometers Traveled During Durability Testing

MIAI TANK	FILTER ELEMENT SET #	KILOMETERS TRAVERSED AT CHANGE	NOTES
L6403 (S1) L6403 (S1)	ELEMENT SBT #1	2957 Km 2637 Km	NORMAL LIFE FILTERS STILL IN SERVICE AT END OF TEST (RESTRICTION 39" - 40" Wg)
1.6404 (S2)	ELEMENT SBT #1	812 Km	FILTERS PULLED FOR BFFICIBNCY CHECK PER MI PMO REQUEST
L640A (S2)	ELEMENT SBT #2 ELEMENT SBT #3 ELEMENT SBT #4	1116 Km 423 Km 113 Km	SPILL VALVE PROBLEMS CAUSED SHORT FILTER LIFE ON SETS 2,3, AND 4. (PROBLEM CORRECTED)
1.6404 (S2)	ELEMENT SET #5	3474 Km	NORMAL LIFE

NORMAL LIFE OF SCAF FILTERS during durability miles average 3022Km, based on an average of element Sets 1 and 2 from S1 and Set 5 from S2.

Standard V-Pac equipped tank during durability mileage based upon the observed performance of M1A1 tank L9107 exhibited a NORMAL LIFE OF STANDARD FILTERS of 438 Km. This is based upon data accumulated during consecutive cleanings from 11/1/88 unid 1/17/89.

# CONCLUSION:

The SCAF equipped tanks wernt 6.9 times further between filter service intervals than the standard V-Pac equipped tank during the durability test phase. This conclusion is based upon a comparison of NORMAL LIFE.

Table 2. Demonstrated Filter Element Service Life in Kilometers Traveled and Time in Service During Dust Bowl Tests

MII TANK	TIME ON TEST	MILAS TRAVERSED	AVERAGE SPEED	RESTRICTION AT END
L6403 (S1)	5 HRS. 25 MIN.	124,3	22.94 MPH	42
L9107(STD)	1 HR. 40 MIN.	30.3	18.18 MPH	57
L6404 (S-2)	(\$2 FAILED TO ACCUM RECEIVER CIRCUIT)	ACCUMULATE SIGNIFICANT MILEAGE DUE TO FAILED COMPRESSOR AND PLUGGED CUIT)	UE TO FAILED COMPRESSOR	AND PLUGGED

SCAF wen 4.12 times further and 3.25 times longer than the standard V-Pro equipped tank. This was short of goal performance for SCAF and Donaldson Company engineering believes SCAF system life would have been much longer if the receiver system hadn't plugged due to the extraordinarily high dust concentrations on the dust coarse. The receiver circuit problems can be simply corrected if a minor revision were allowed.

## Motor Technology Inc.

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FAX: 513-294-8336

2796 CULVER AVENUE DAYTON, OHIO 45429 513 - 294 - 1041
TO: PAGE   OF DATE 10 17 89
FAX NO .: 612/887-3155 FROM: R. BUCHWALDER
COMPANY: DONALDSON
ATTN: HARRY CAMPLIN
REF: FAILURE BUALYSIS
& CORRECTIVE ACTION
REF: MTI P/N 276A141
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